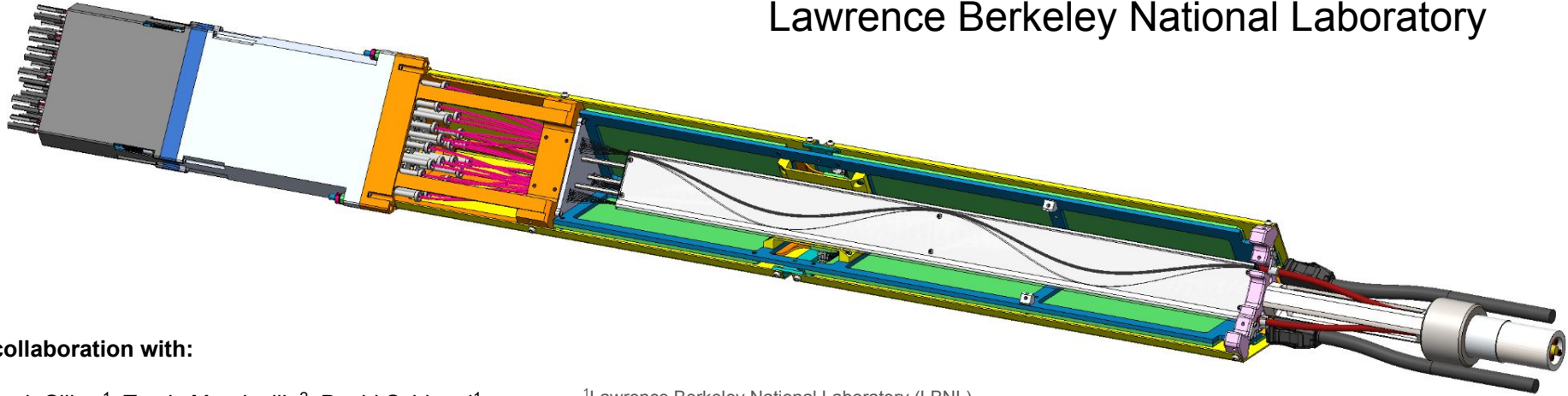


Design of High-Density Fiber Positioning Raft Modules: Integration with Fibers and Positioners

Nicholas Wenner

Lawrence Berkeley National Laboratory



In collaboration with:

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²University of Washington

³Space Sciences Laboratory (SSL)

⁴École Polytechnique Fédérale de Lausanne (EPFL)

⁵Yale University

⁶University of Colorado

⁷University of Michigan

SPEC-S5 Instrumentation Workshop

University of Chicago

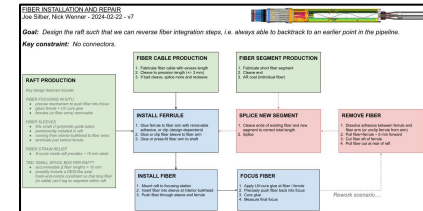
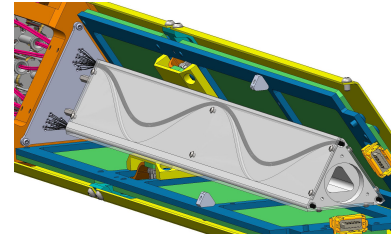
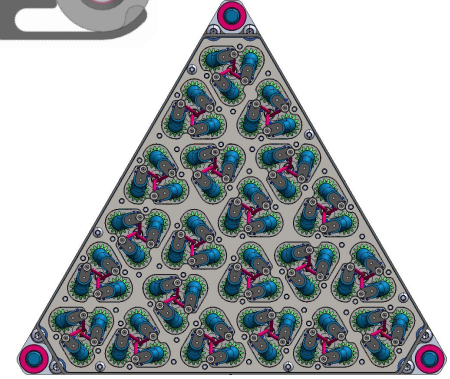
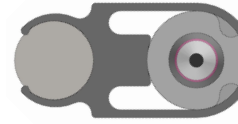
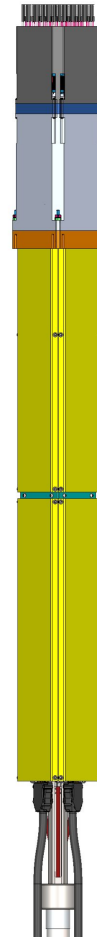
February 27, 2024

Outline

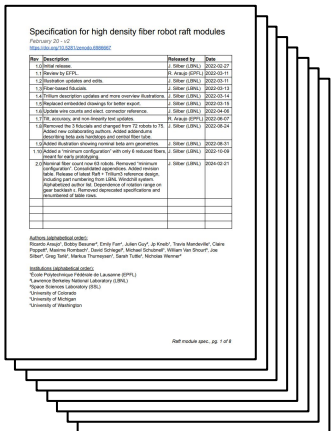
- Raft Design
 - Fiber Routing
 - Fiber Arms

- Fiber Installation Process
 - “Push-to-Focus”

- Repair / Rework Process



Key Specifications: Raft

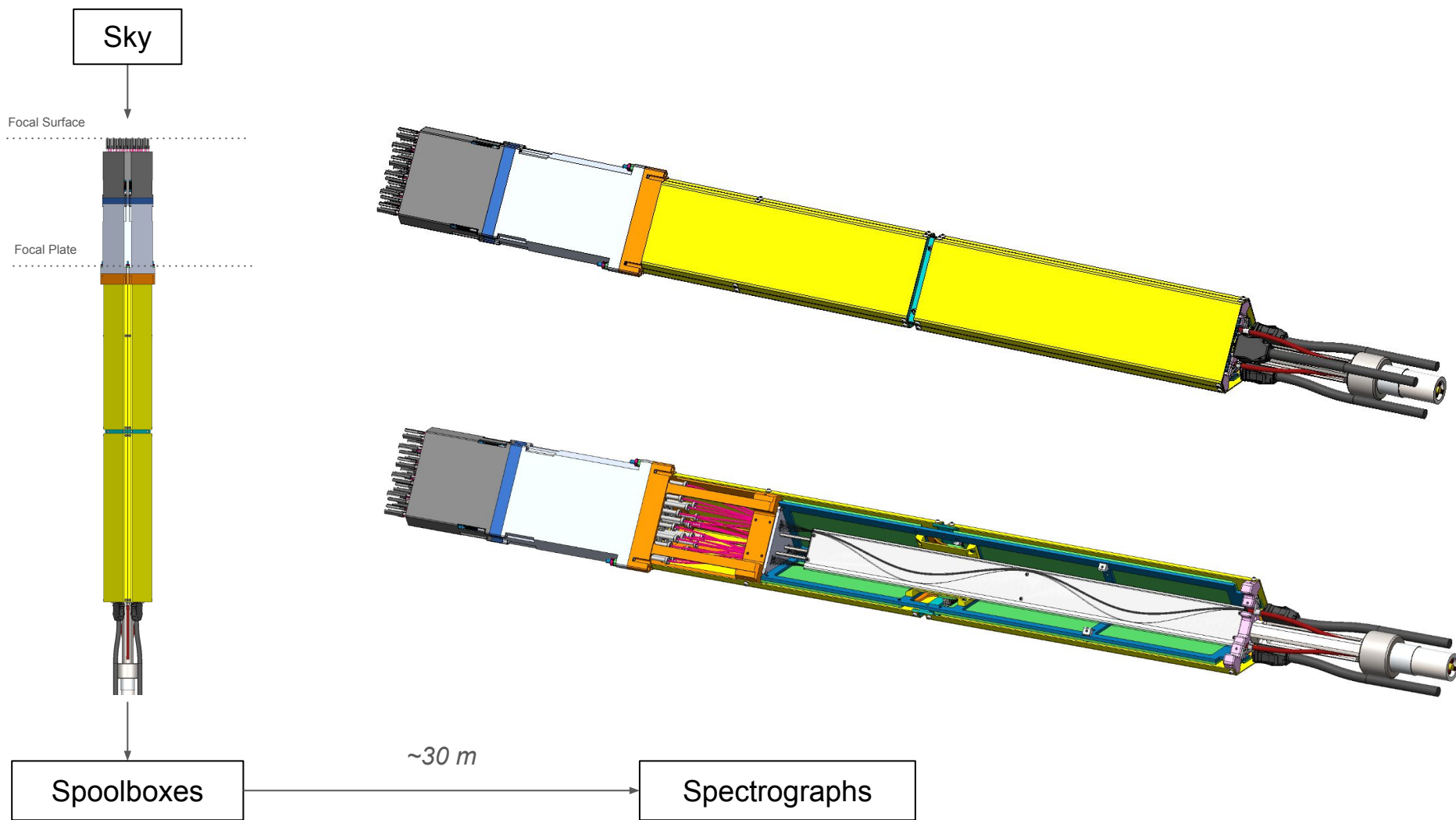


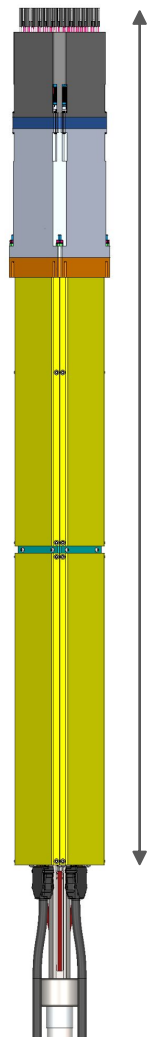
39 total
specs

Title	Requirement or Feature
Envelope	Equilateral triangle with 74 mm sides, chamfered by 2.5 mm at corners
Length & mass	≤ 650 mm, ≤ 1.5 kg
Robot spacing	6.2 mm center-to-center
Number of fiber robots	63
Kinematic arm lengths	Ra = 1.8 mm (from central axis to eccentric axis) Rb = 1.8 mm (from eccentric axis to fiber center)
Arm length variation	$ Ra - Rb \leq 0.1$ mm
Ranges of motion	$(-2^\circ - \epsilon) \leq \alpha \leq (362^\circ + \epsilon)$ Example: If backlash = 5° , then $-7^\circ \leq \alpha \leq 367^\circ$ $(-2^\circ - \epsilon) \leq \beta \leq (182^\circ + \epsilon)$ Example: If backlash = 5° , then $-7^\circ \leq \beta \leq 187^\circ$
Gear backlash	$\epsilon \leq 5^\circ$
Hard stops	both axes
Fiber guide tubes	OD ≤ 0.7 mm ID ≥ 0.19 mm (TBC)
Min fiber bend radius	50 mm
Power consumption	≤ 1.2 W per motor, moving at full speed
XY positioning precision	≤ 5 μ m rms
XY positioning accuracy	≤ 50 μ m rms
Fiber defocus	≤ 50 μ m, for all (α , β)
Fiber tilt	$\leq 0.5^\circ$, for all (α , β)
Speed at output shaft	180°/sec (goal), 30°/sec (minimum)
Lifetime	$\geq 100,000$ move cycles
Operating conditions	Temperature: -20°C to $+40^\circ\text{C}$ Humidity: ~ 0 to 80% RH

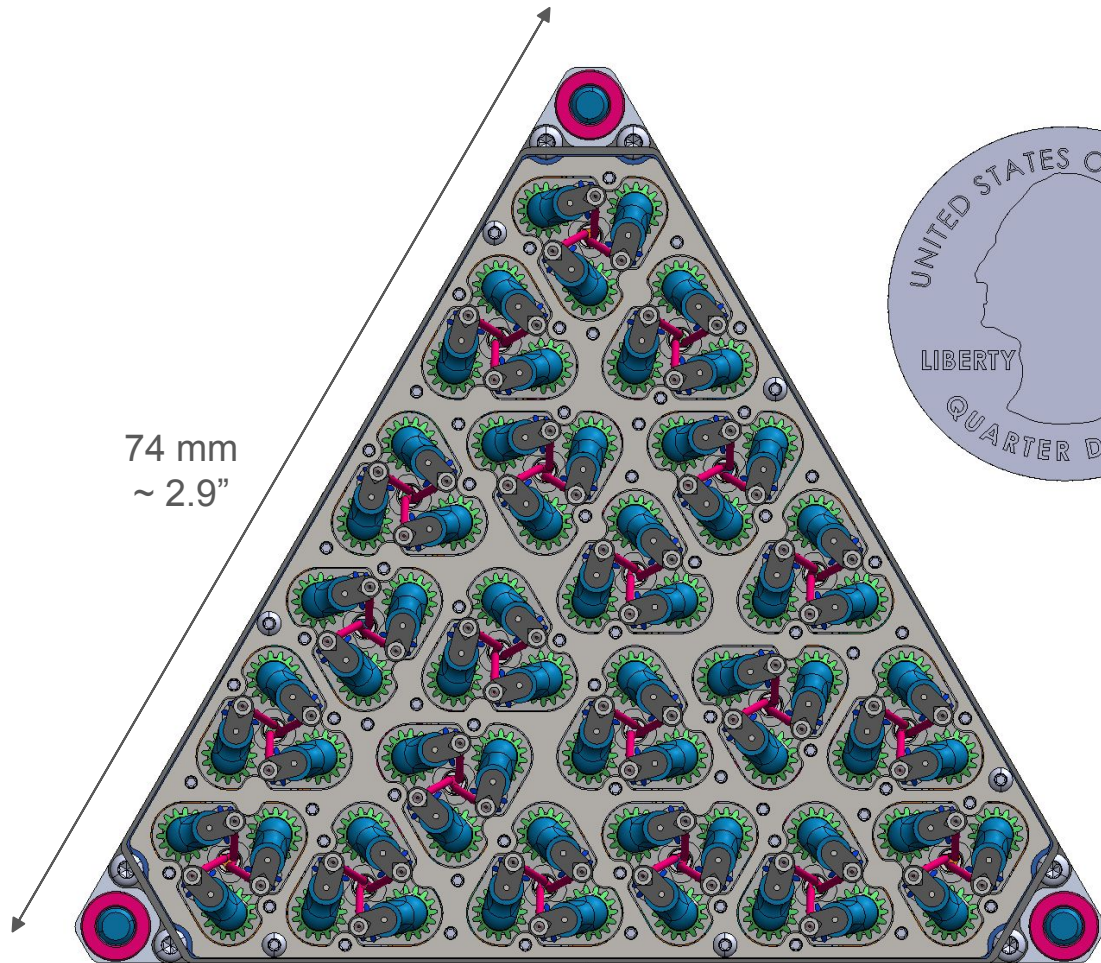
Additional Goals/Constraints

- **Repairability** ⇒ Make as many steps as possible **reversible**
- **“Push-to-focus”** fiber alignment process
 - Employ disposable glass ferrules with in-raft UV cure to mount, protect, focus, and align fiber tips
- **Continuous fiber run**
 - Avoid losses from splicing and connectors
 - Connectors could be easily integrated, if sufficiently performant ones were designed
- **63 fibers per raft**, based on studies of packing density in the focal plane
 - Currently based on **Trillium fiber positioners**, but others could be used



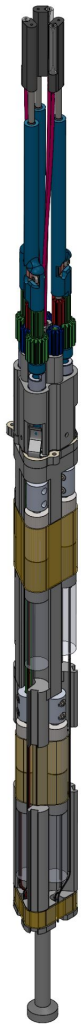


~ 600 mm
~ 24"



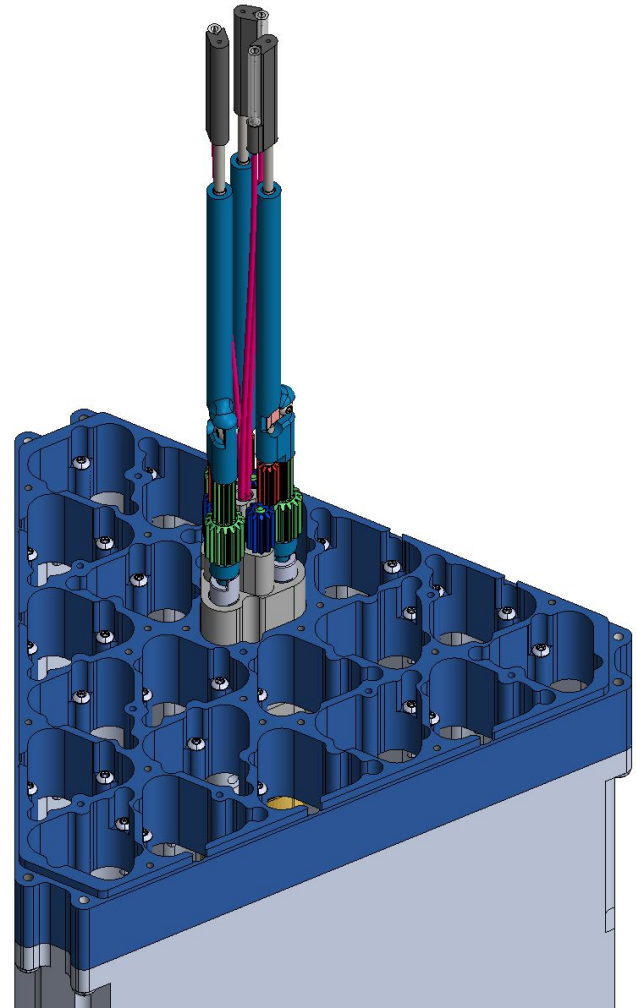
74 mm
~ 2.9"

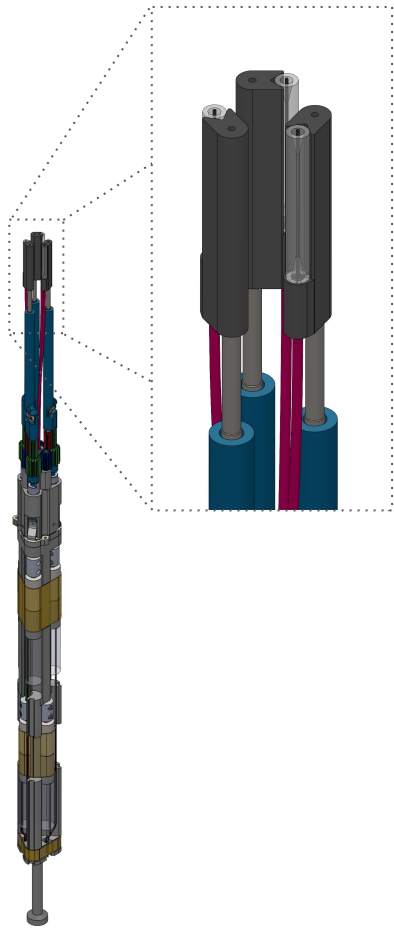




Raft currently designed for Trillium3

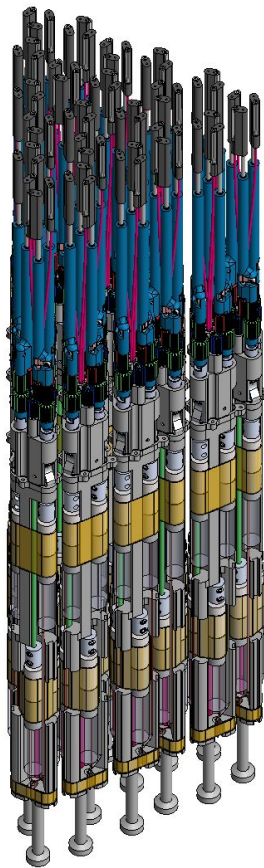
Other types of positioners could be used





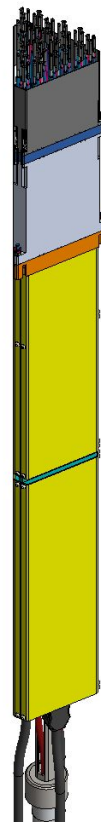
3 Fibers / Trillium

X

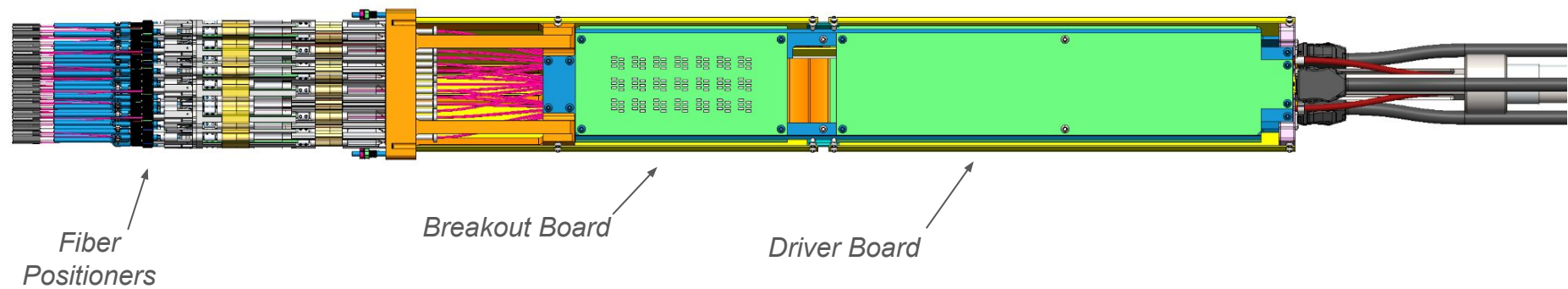
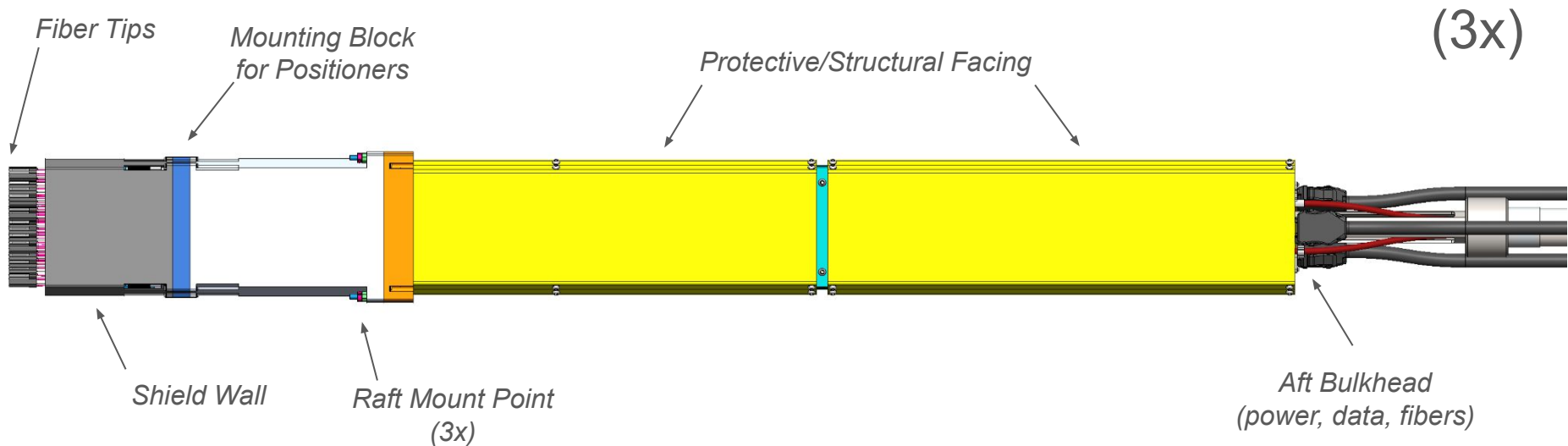


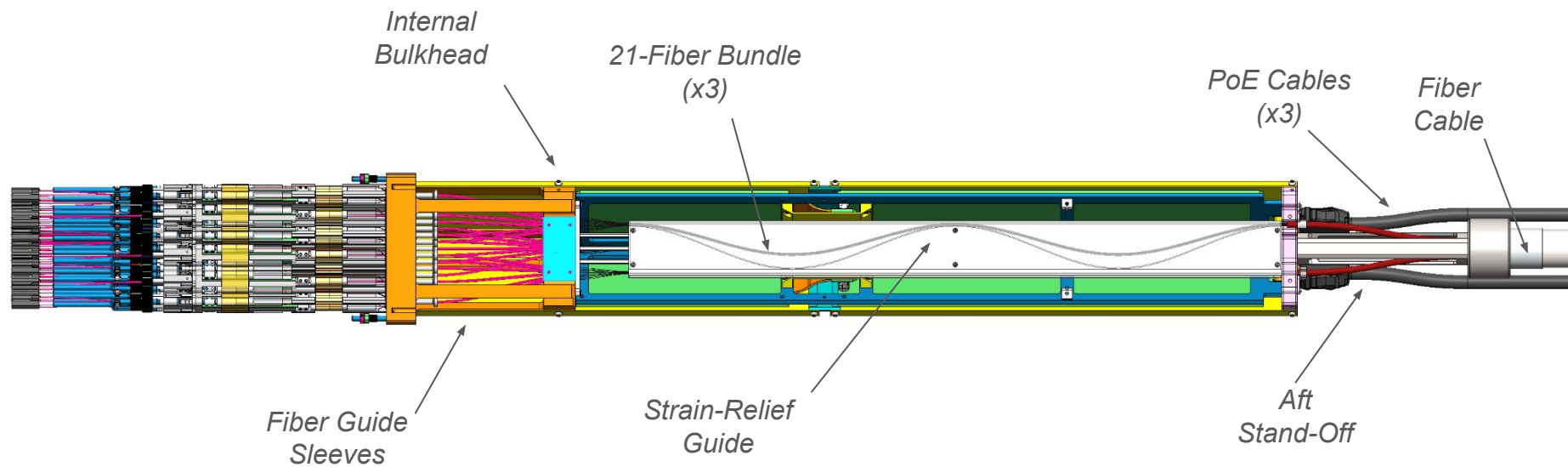
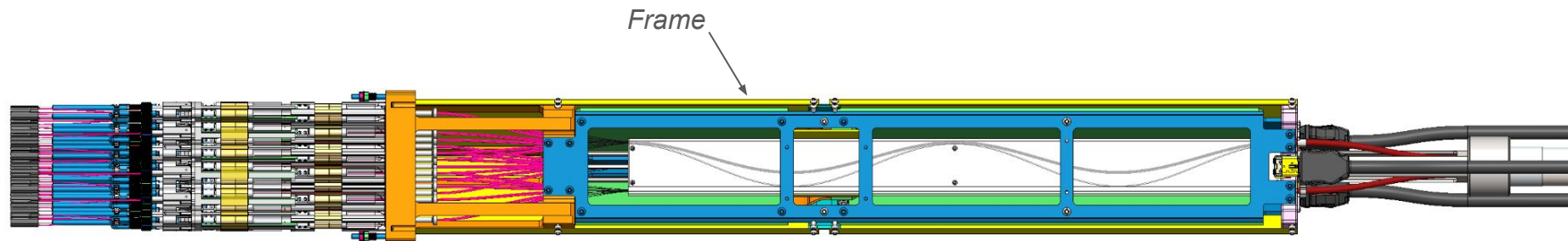
21 Trilliums / Raft

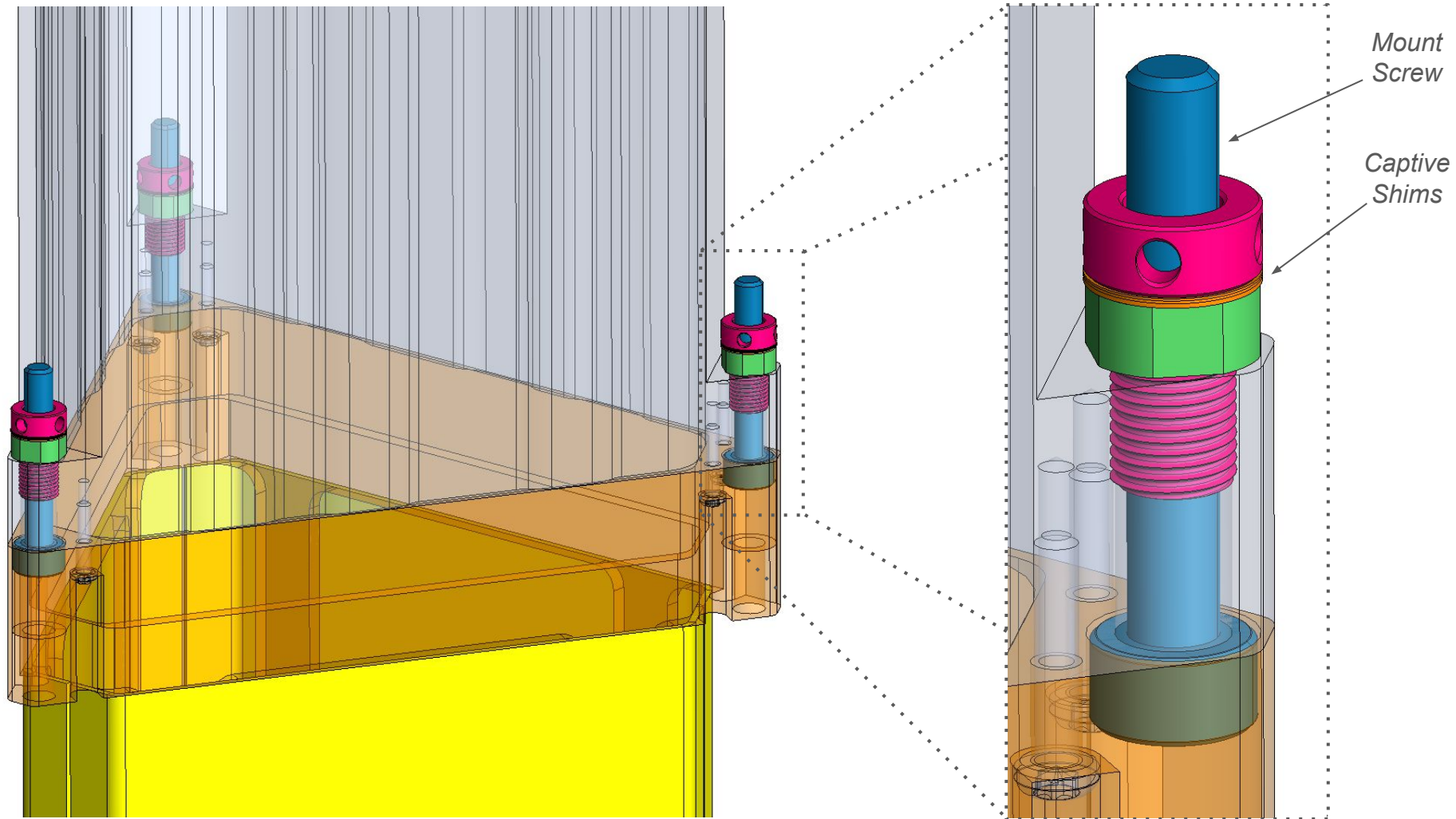
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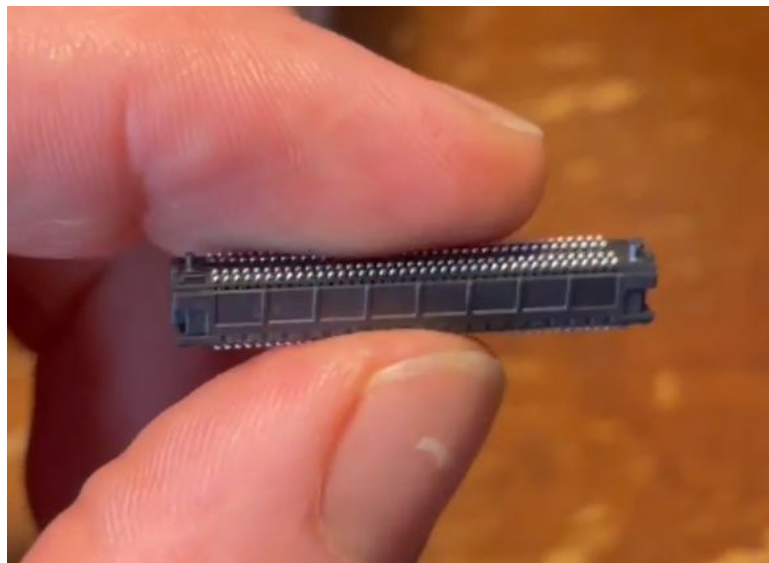
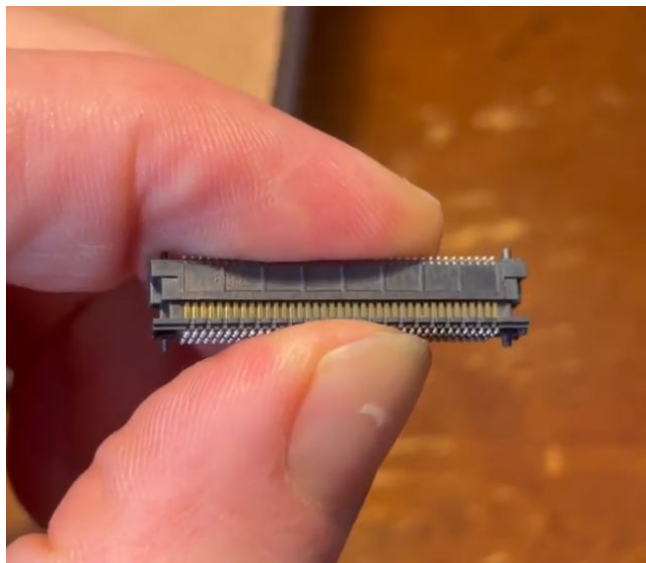
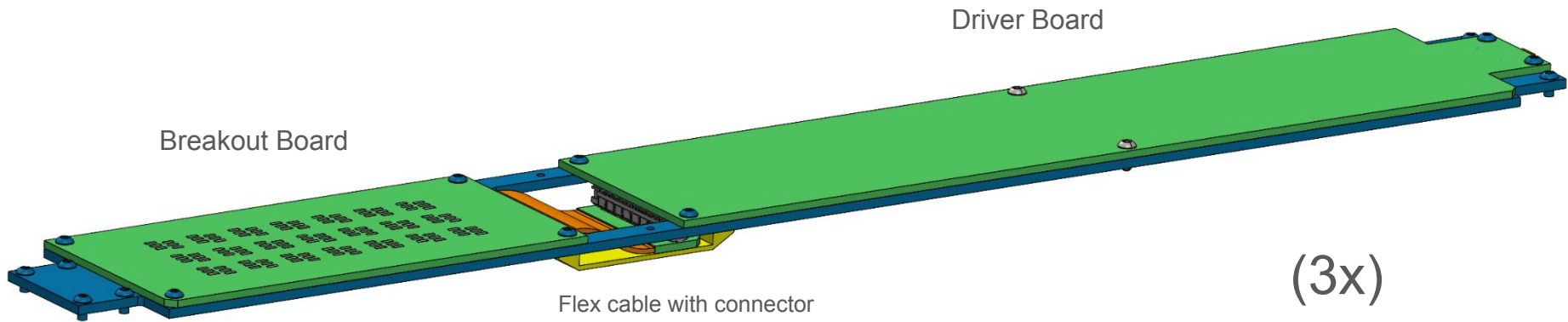


63 Fibers / Raft

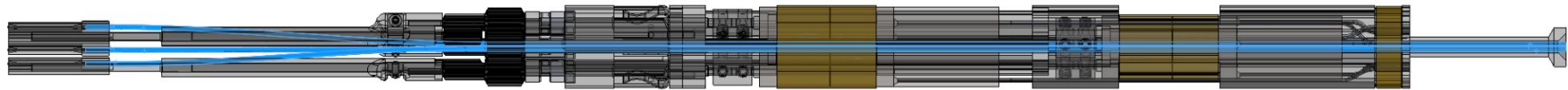




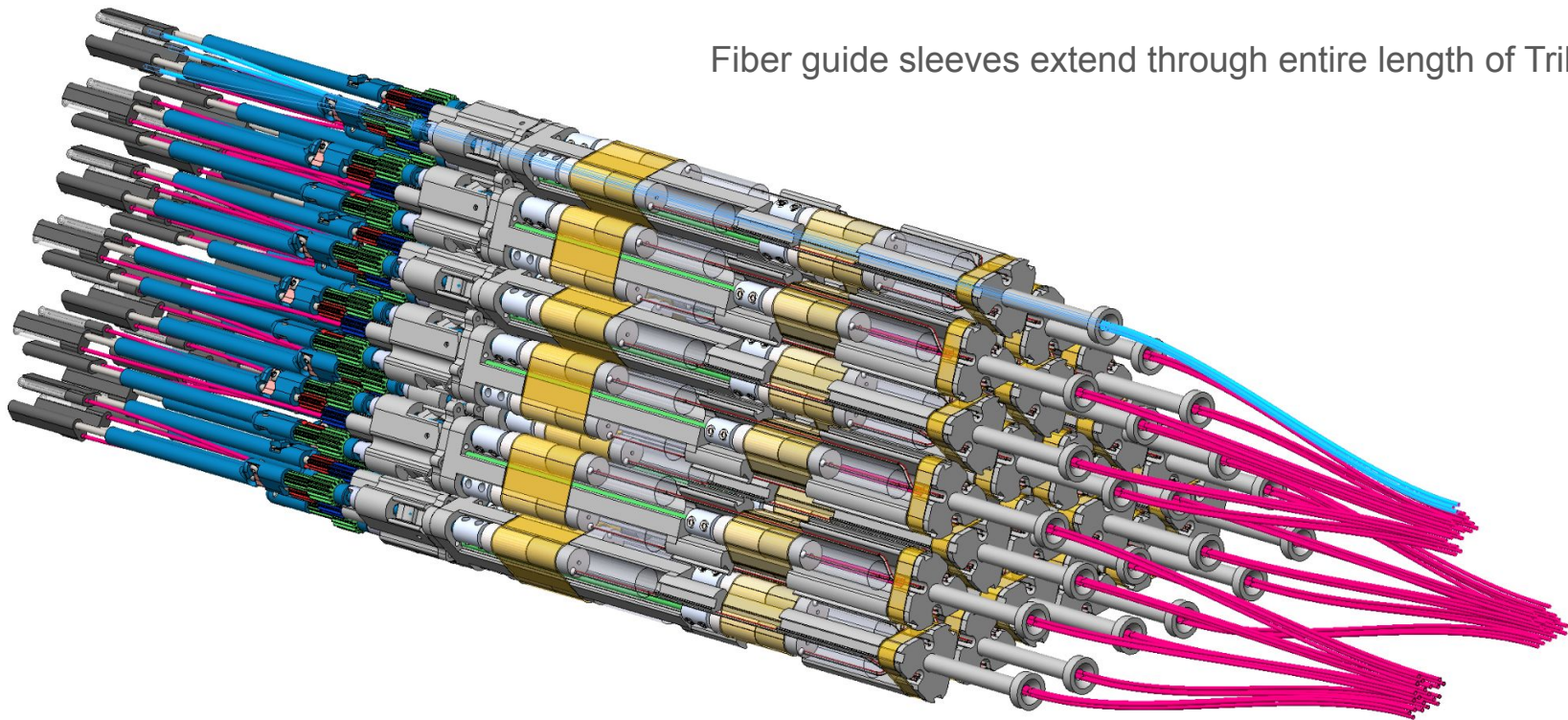




Samtec ADF6-40-03.5-L-4-2-A-TR



Fiber guide sleeves extend through entire length of Trilliums





Aft Bulkhead

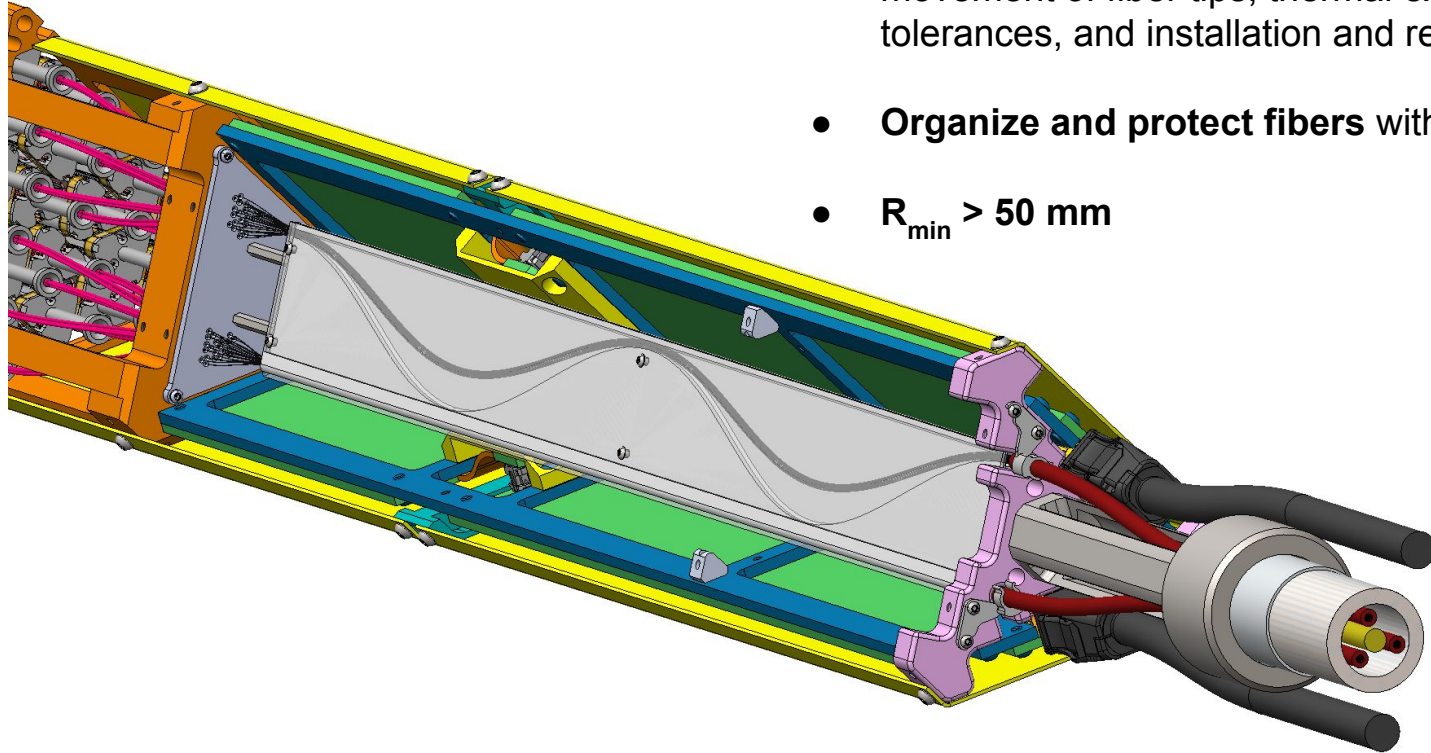
The diagram shows a yellow bulkhead on the left with a purple internal structure. Three red PoE cables and three black fiber cables are connected to the bulkhead. The fiber cables are bundled together and terminate in a white connector with three red and yellow ports. The PoE cables terminate in black ix connectors.

3x PoE cables with
ix connectors

Fiber Cable

3 bundles x 21 fibers / bundle
= 63 fibers

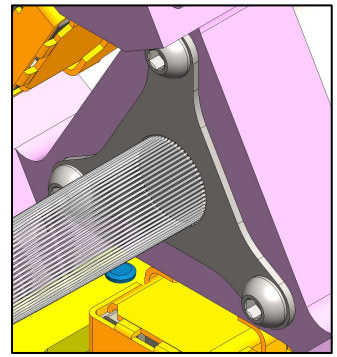
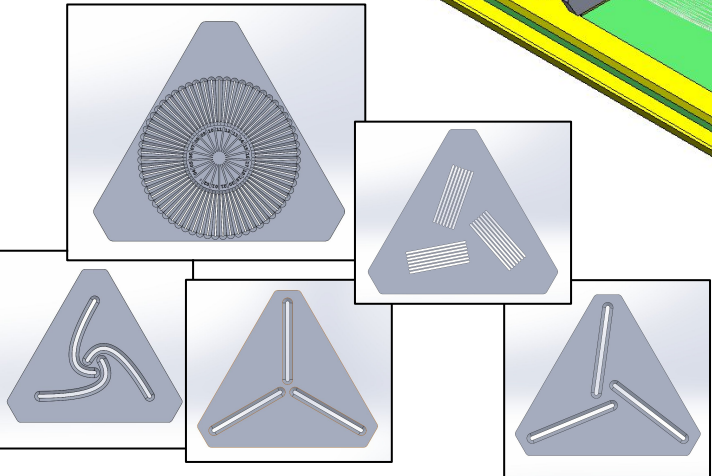
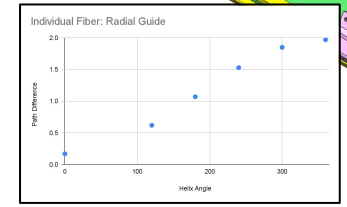
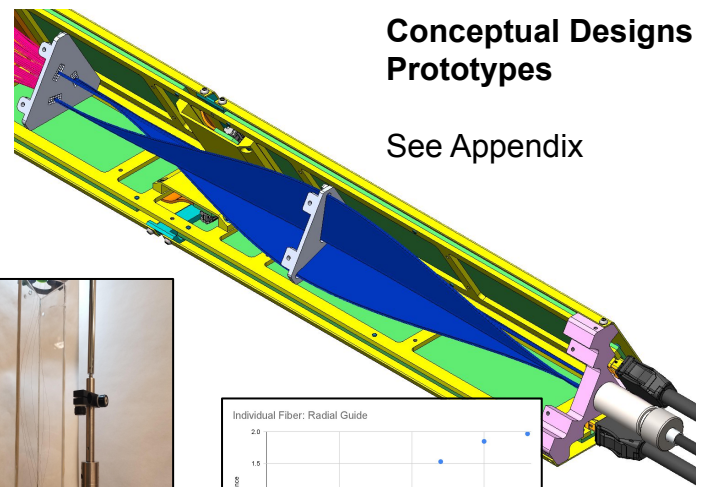
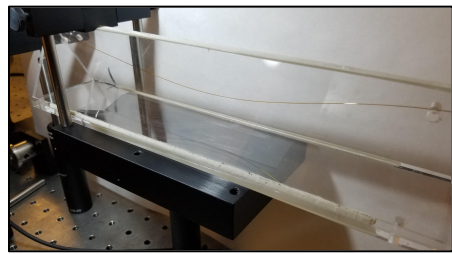
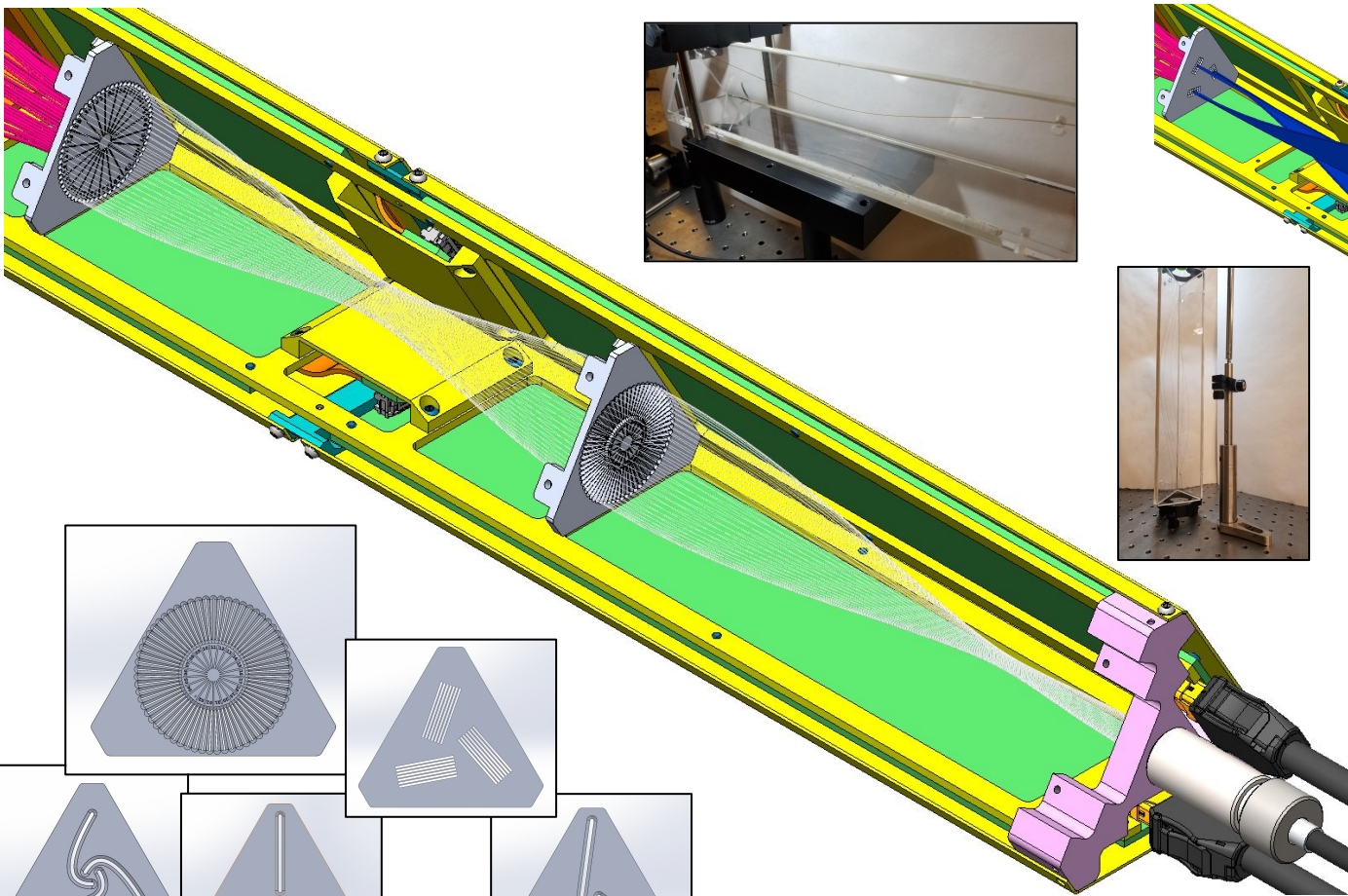
Fiber Slack / Strain Relief

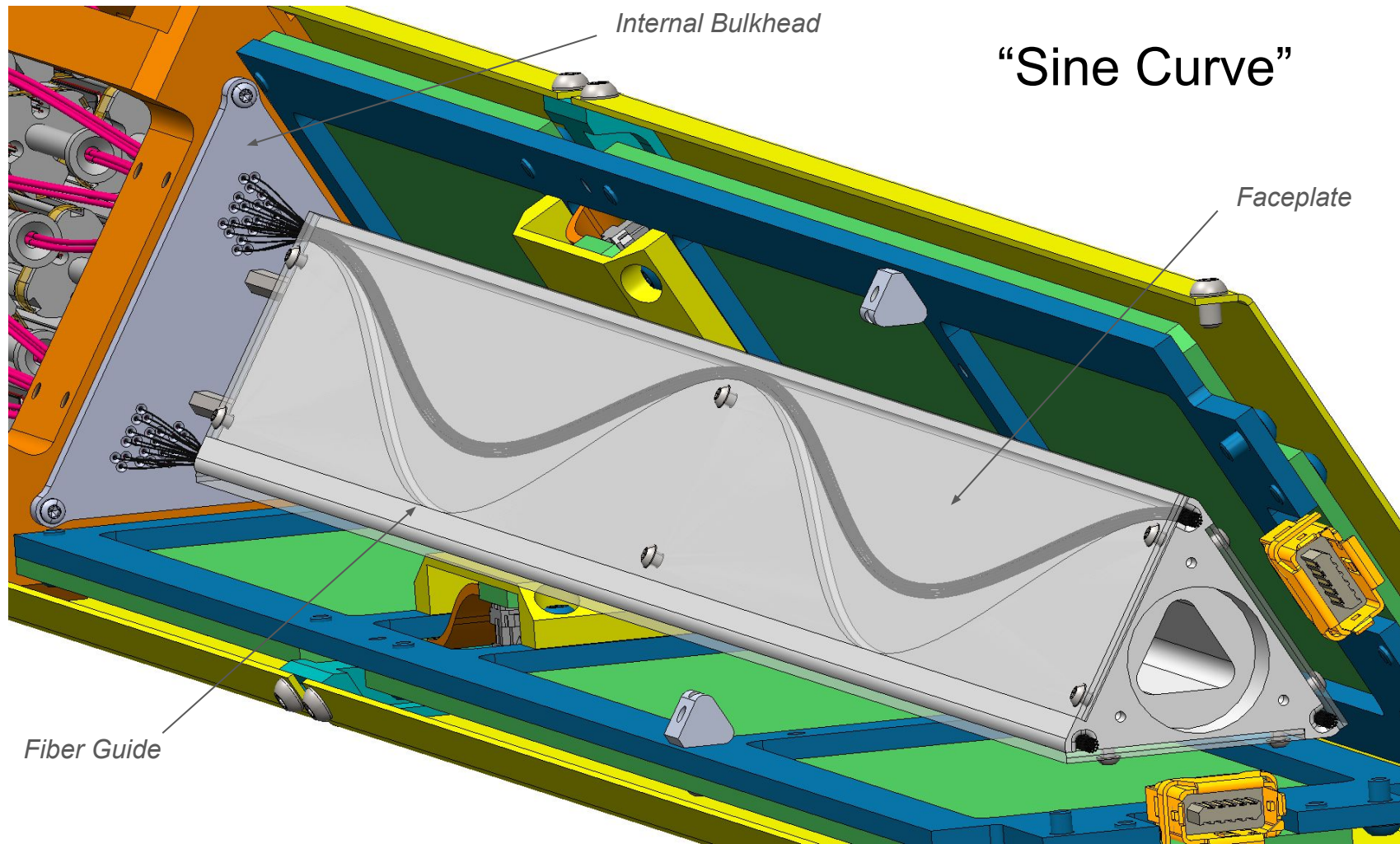


- Provide **> 10 mm fiber slack** within raft to accommodate movement of fiber tips, thermal expansion, manufacturing tolerances, and installation and rework processes
- **Organize and protect fibers** within raft
- $R_{\min} > 50 \text{ mm}$

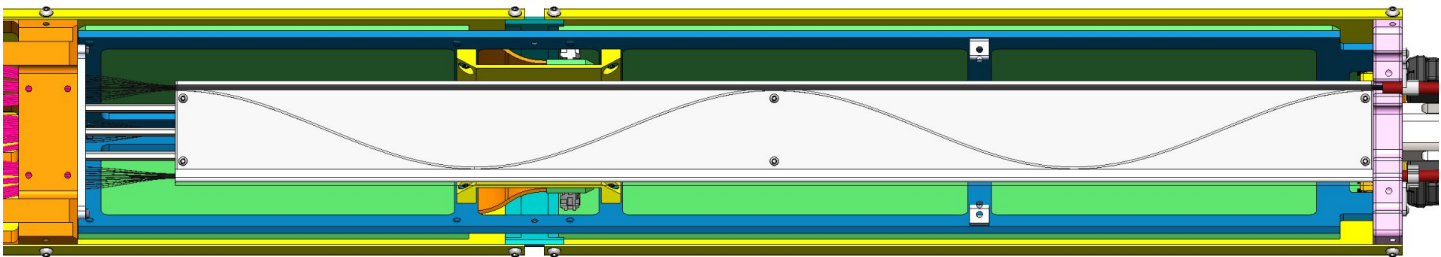
Conceptual Designs Prototypes

See Appendix





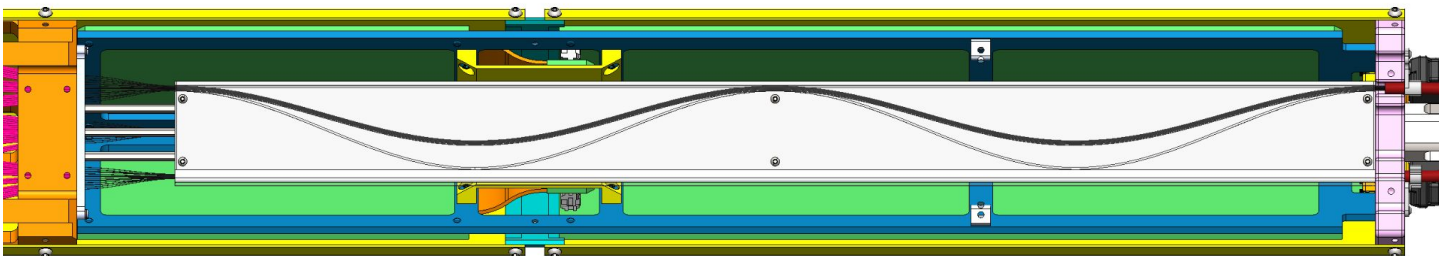
**Min
Length**



Slack:
0 mm

$R = \infty$

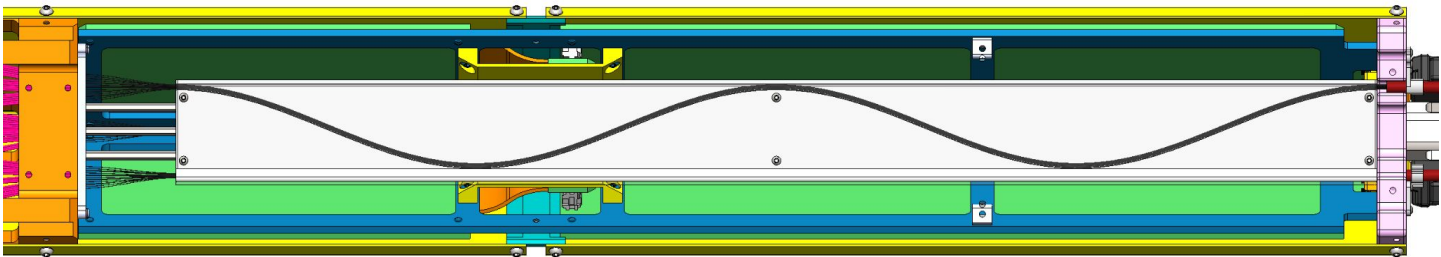
**Mid
Length**



Slack:
+/- 6 mm

$R_{\min} = 85 \text{ mm}$

**Max
Length**



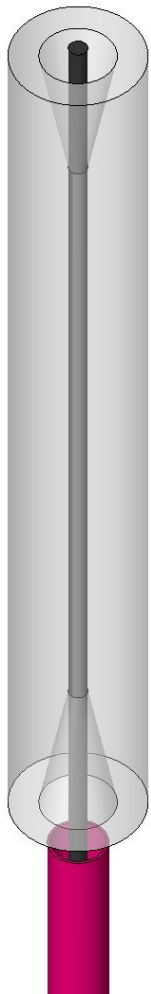
Slack:
12 mm

$R_{\min} = 59 \text{ mm}$

Fiber Installation Process

“Push-to-Focus”

developed particularly with contributions from Travis Mandeville and Joe Silber



Key features:

- Disposable ferrule, reversibly attached to fiber arm
- Fiber inserted, focused, and UV-glued to ferrule

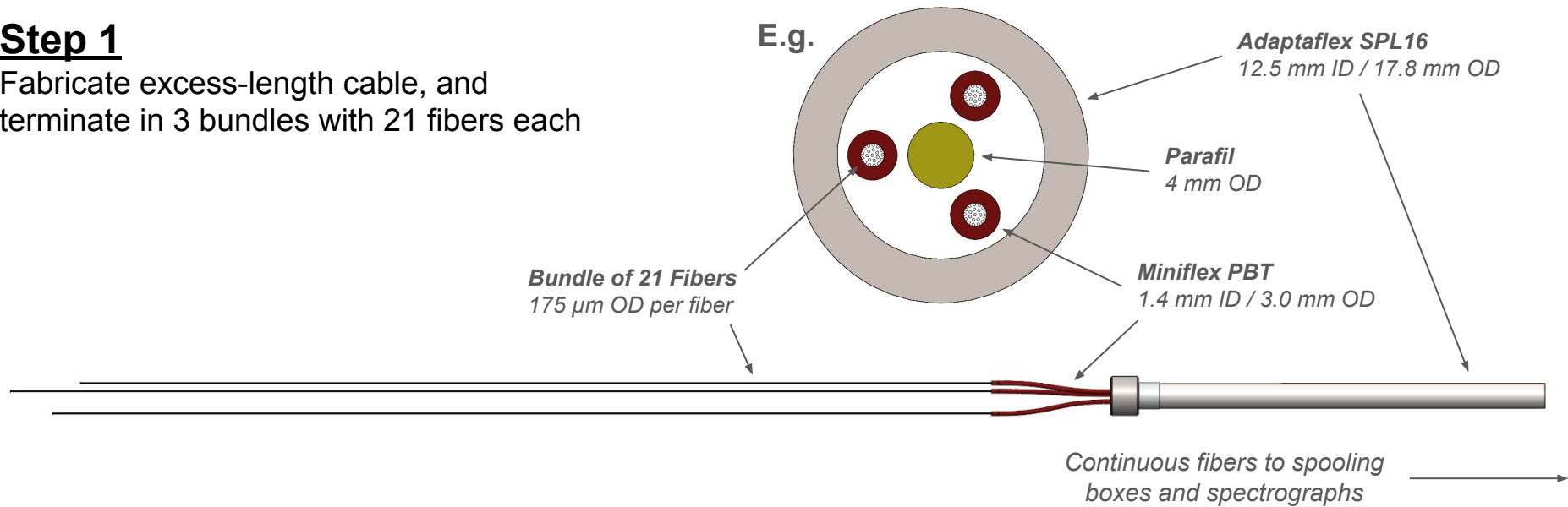
For rework:

- Cut fiber
- Throw away ferrule
- Splice new fiber
- Reinstall

Based on DESI ferrule, which remains the best option for FRD and alignment performance

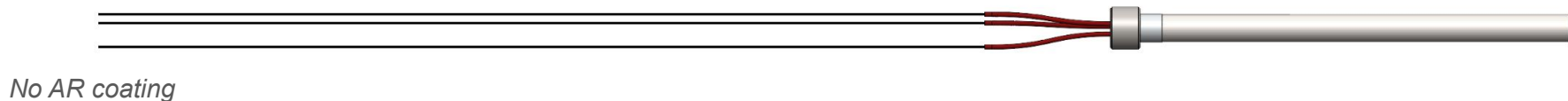
Step 1

Fabricate excess-length cable, and terminate in 3 bundles with 21 fibers each



Step 2

Cleave fibers to length (+/- 3 mm)



Fiber guide sleeves

Forward bulkhead

Step 3

Route fibers through raft and robots

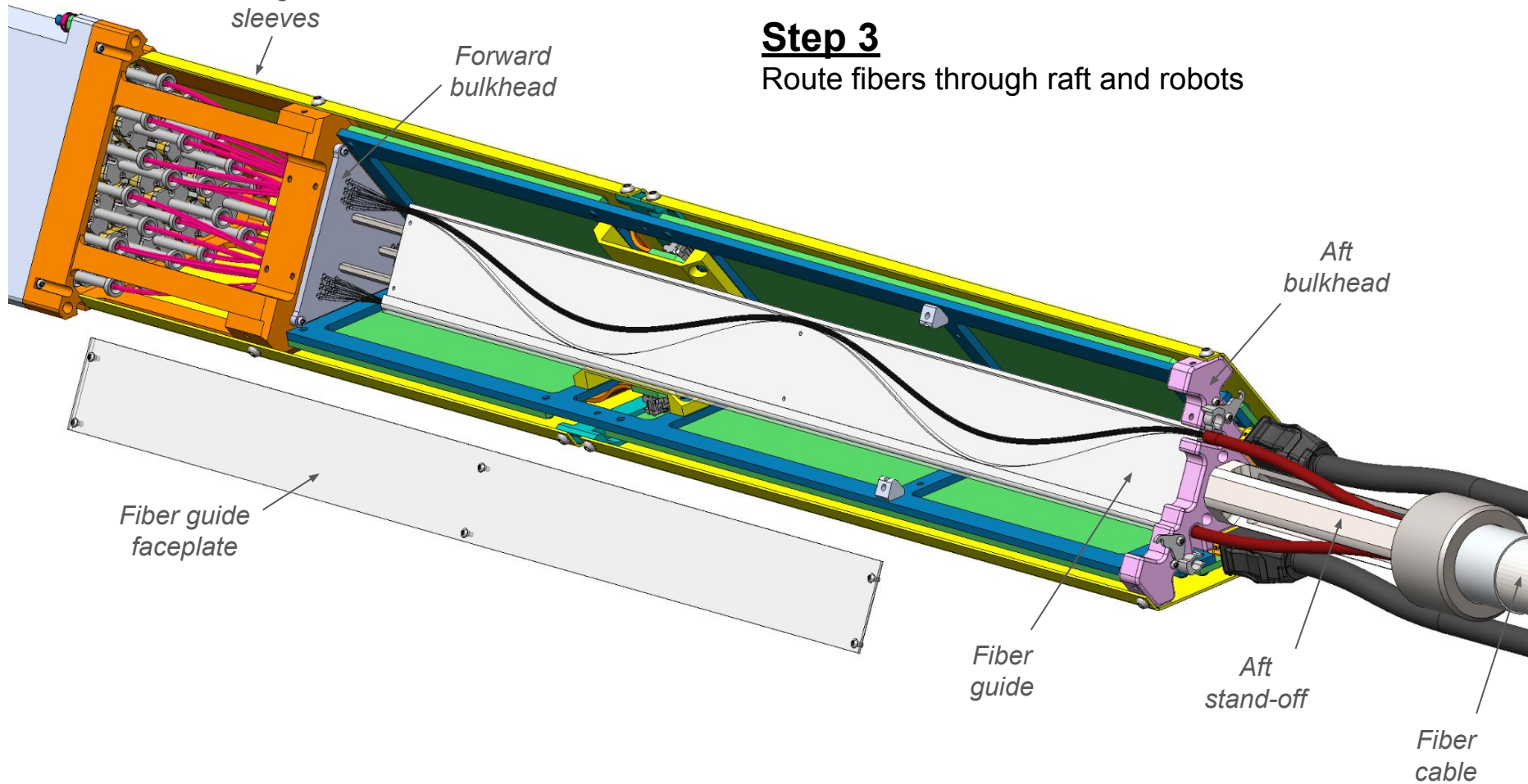
Aft bulkhead

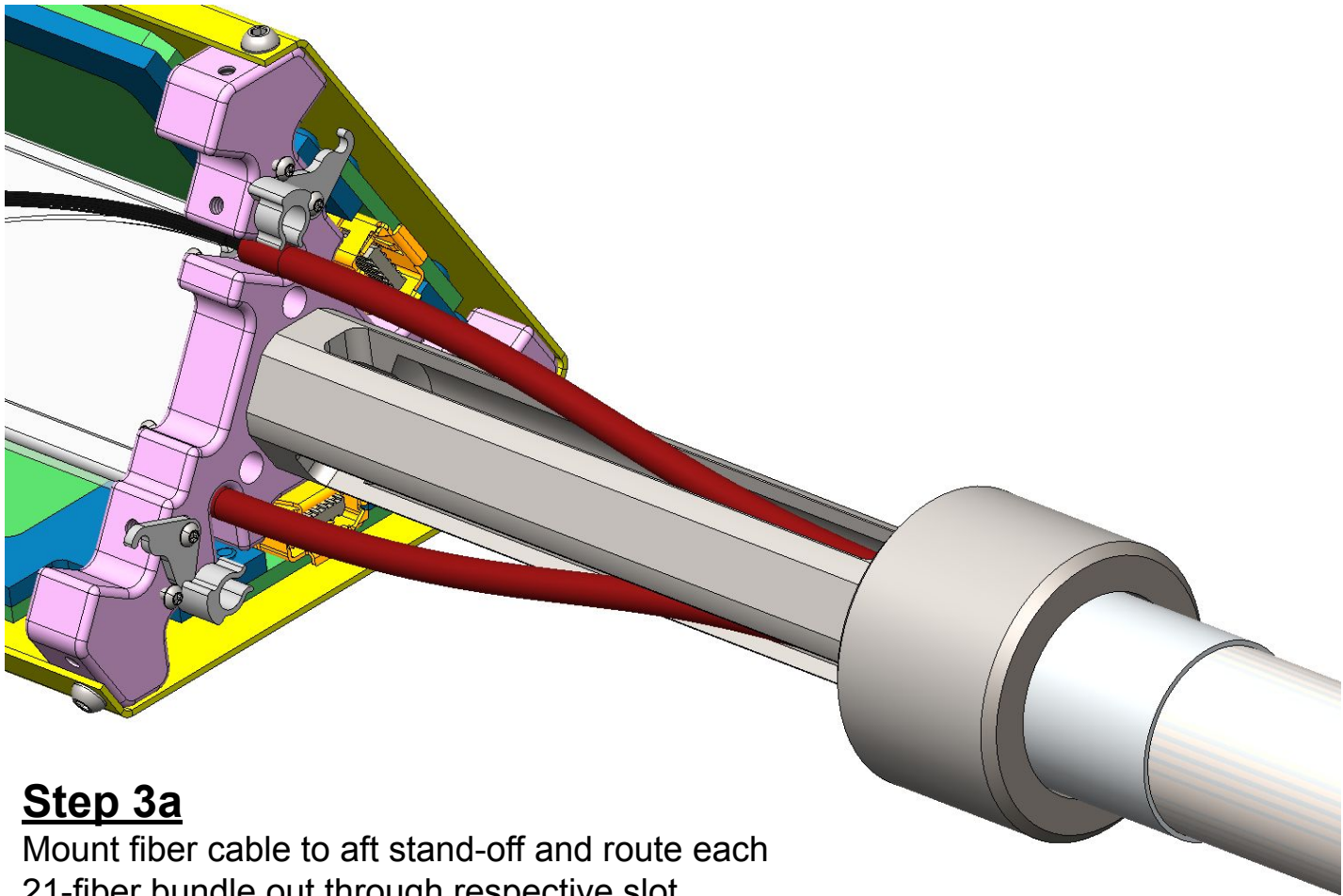
Fiber guide faceplate

Fiber guide

Aft stand-off

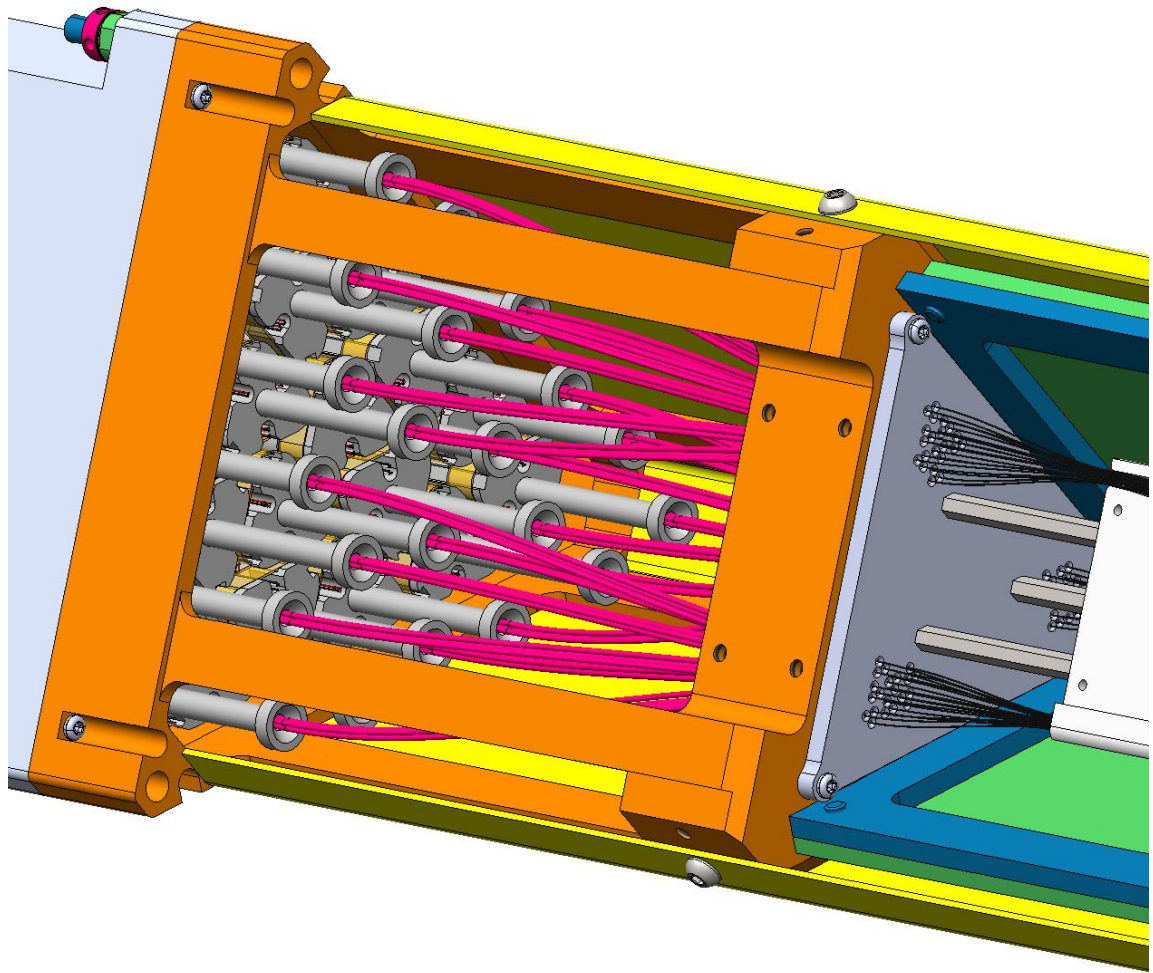
Fiber cable





Step 3a

Mount fiber cable to aft stand-off and route each 21-fiber bundle out through respective slot

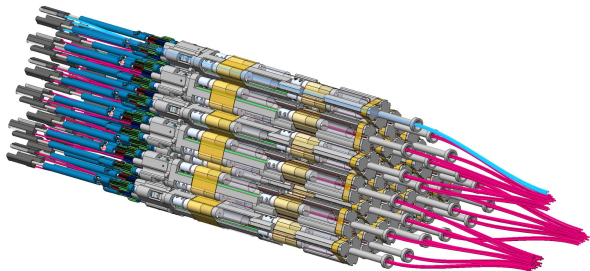
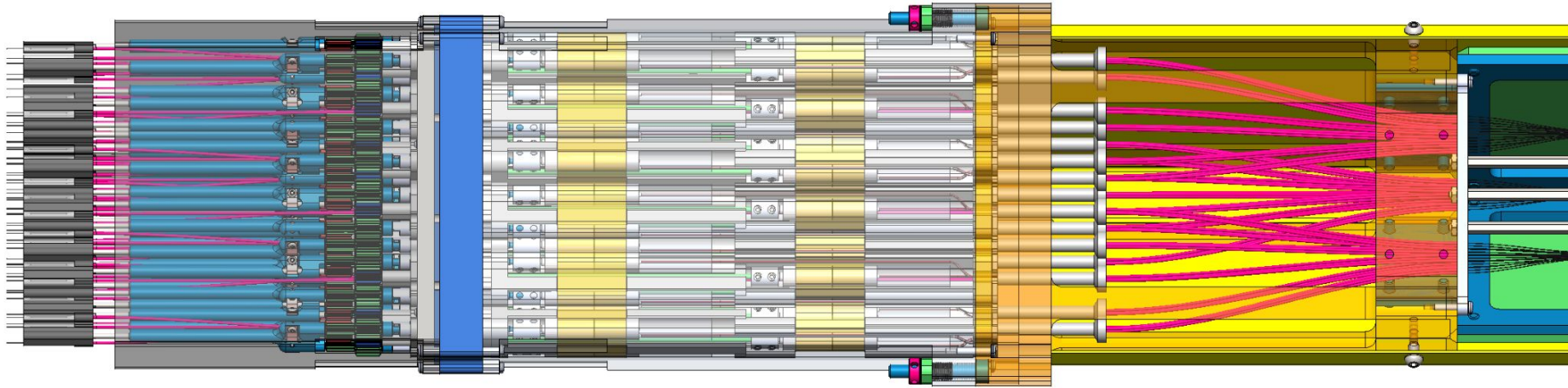


Step 3b

Route fibers through internal bulkhead and into individual fiber guide sleeves

Step 3c

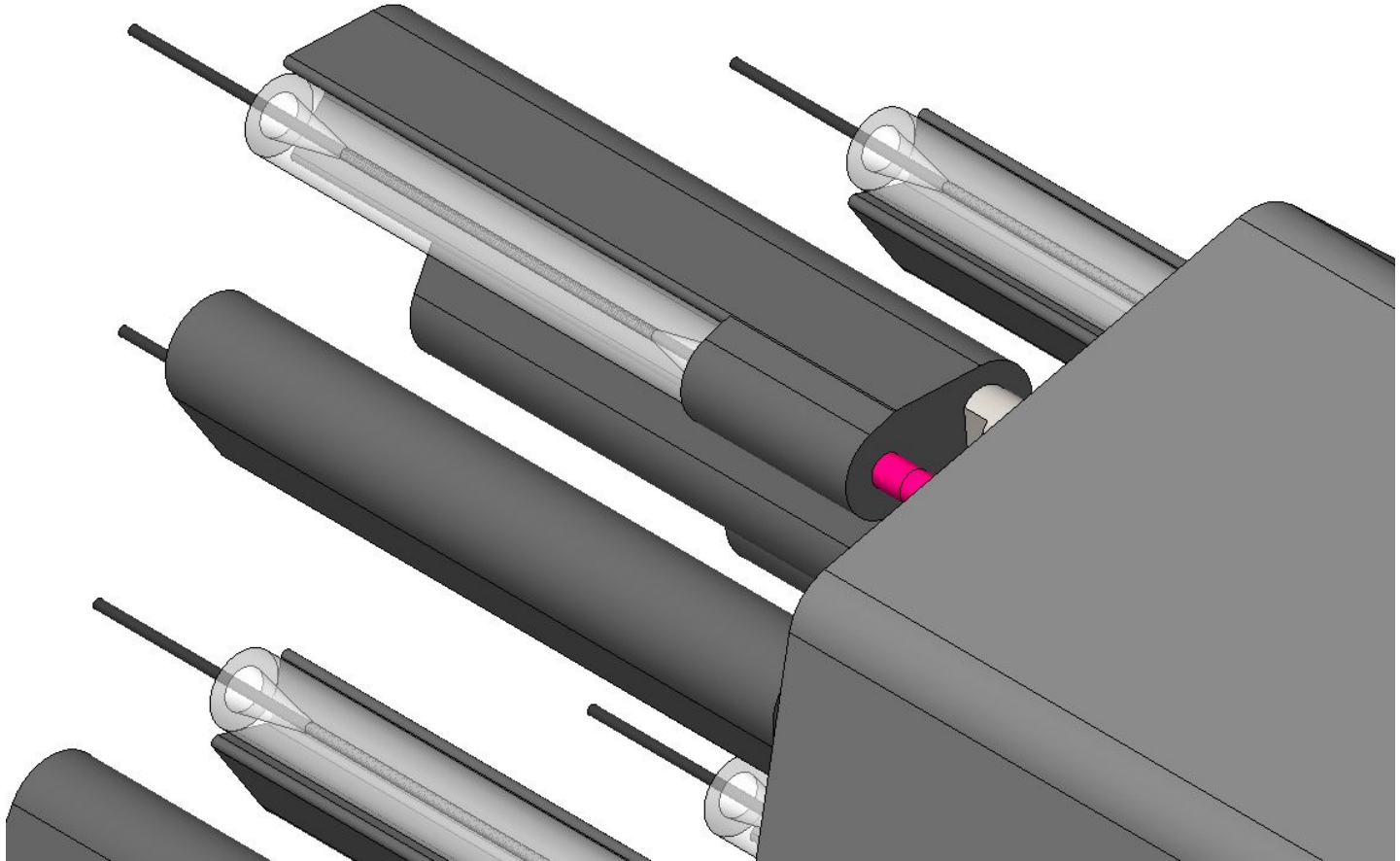
Push fibers through fiber guide sleeves and through robots to forward end of raft



Fiber guide sleeves extend through entire length of robots

Step 4

Push fibers through ferrules past intended focal surface



Step 5

Focus and secure fiber

5a Apply UV-cure adhesive



5b Push fiber into focus



5c Cure adhesive

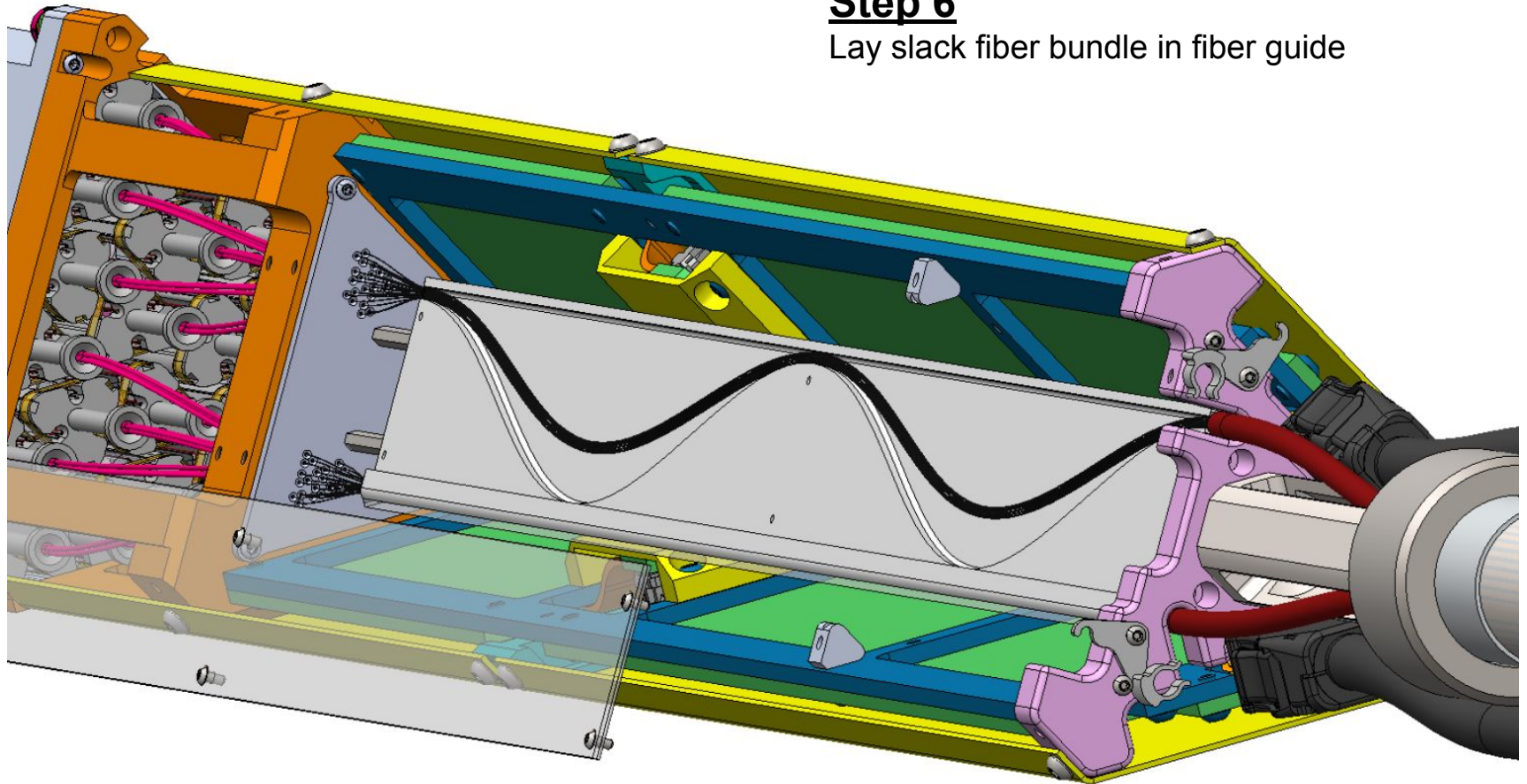


UV Light



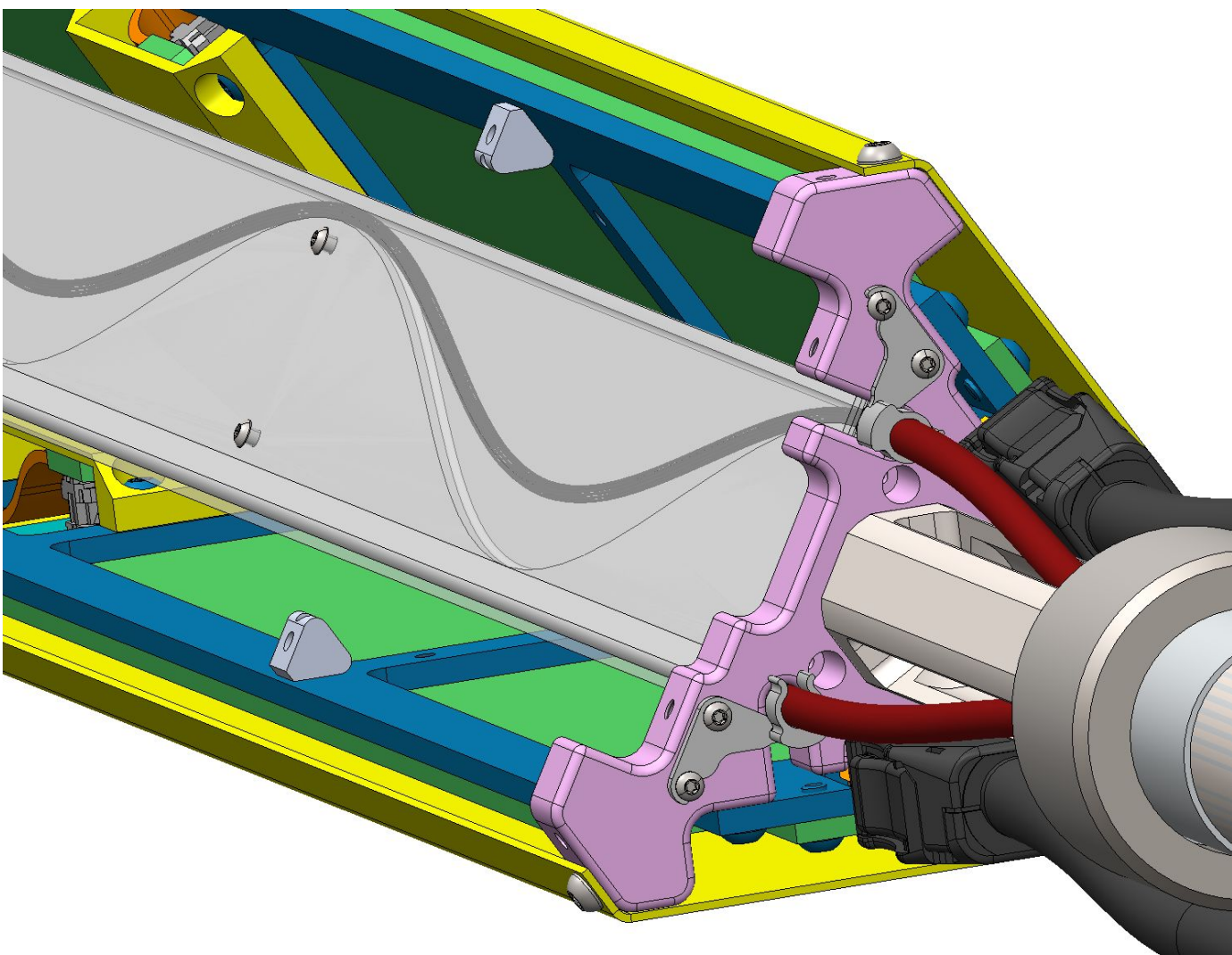
Step 6

Lay slack fiber bundle in fiber guide



Step 7

Secure fiber bundle with
aft clip and fiber guide
faceplate



Rework Process

Rework

Step 8: Press-Fit Fiber Arm

Detach ferrule from fiber arm then cut fiber

8a Dissolve adhesive between ferrule and fiber arm

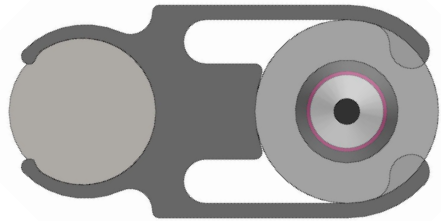
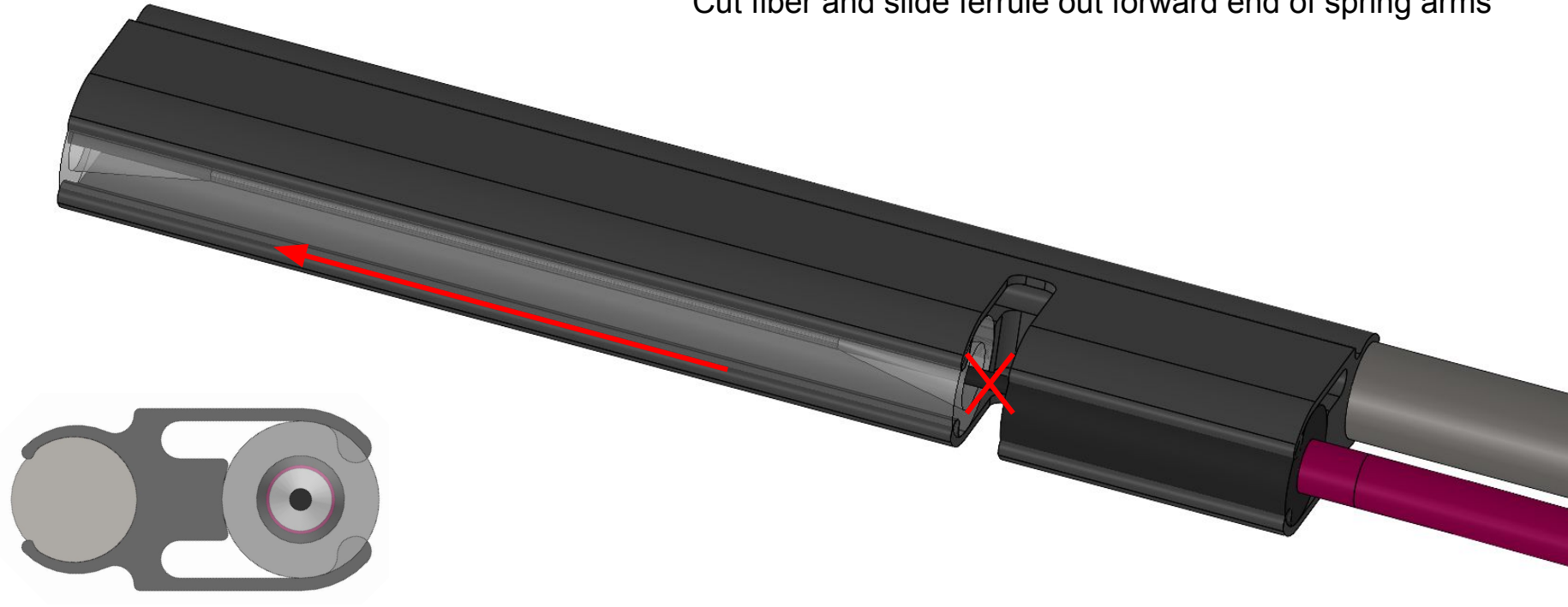


8b Pull ferrule and fiber ~5 mm forward and cut fiber

Rework

Step 8: EDM Fiber Arm

Cut fiber and slide ferrule out forward end of spring arms

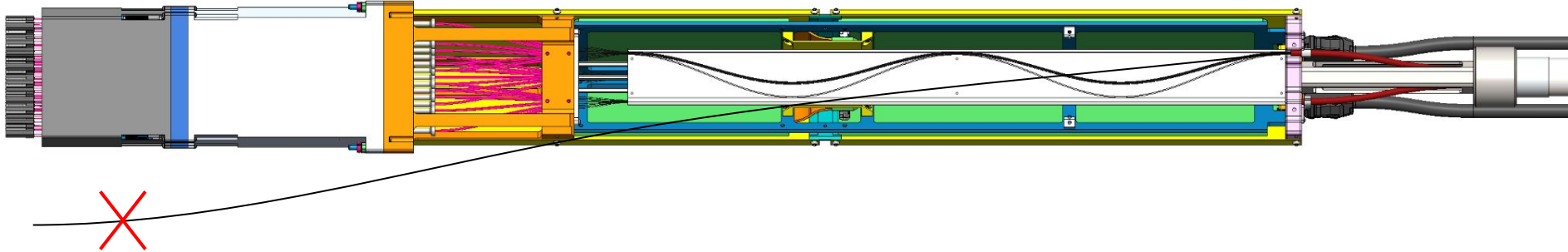


Top View

Rework

Step 9

Pull fiber back, out, and away from raft to aft bulkhead



Step 10

Cleave fiber to length in preparation for splicing

Step 11

Prepare new AR-coated fiber segment and cleave to length

Step 12

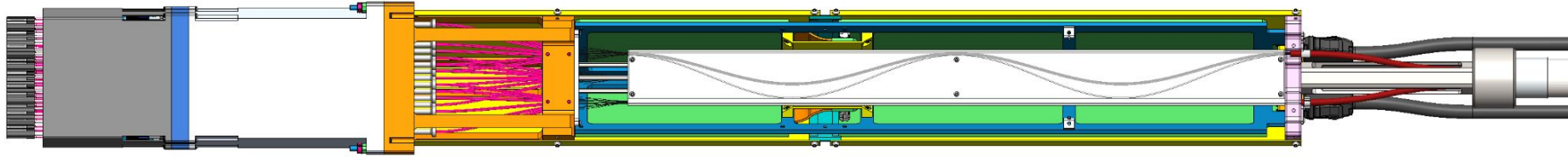
Splice new fiber segment with remainder of original fiber

Effects of AR coating and splicing on FRD roughly cancel out, although stability is diminished

Rework

Step 13

Start at **Step 3b** to reroute, resecure, refocus, and reglue the repaired fiber



*Splice is protected by fiber guide sleeve
and can be located anywhere within it,
allowing for multiple reworks*

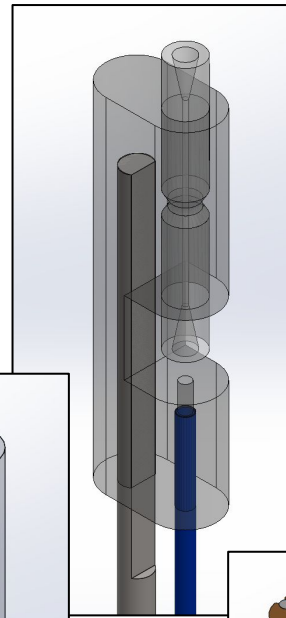
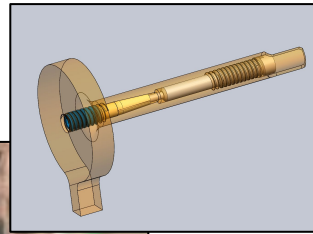
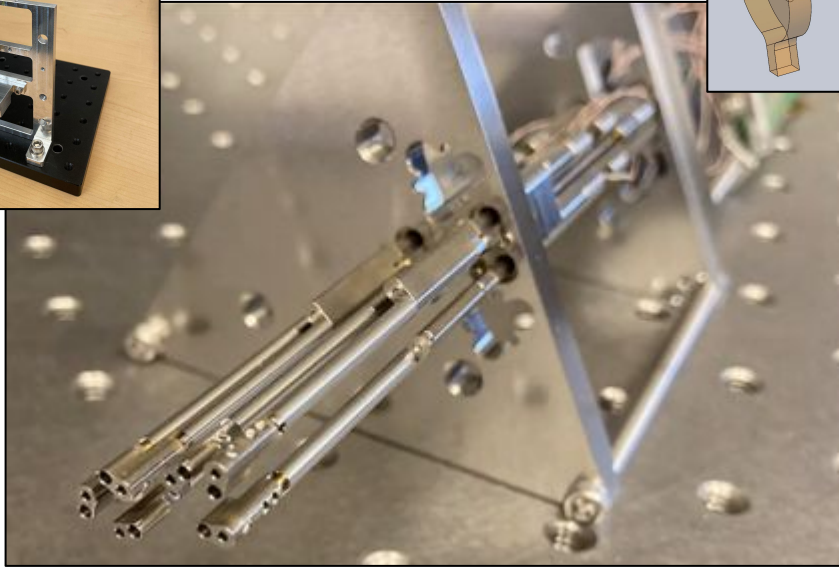
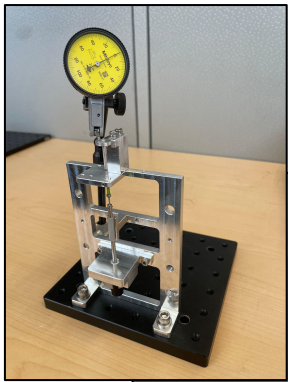
Additional protection may be necessary

Fiber Arms

Reversibly connect fiber/ferrule to fiber positioner

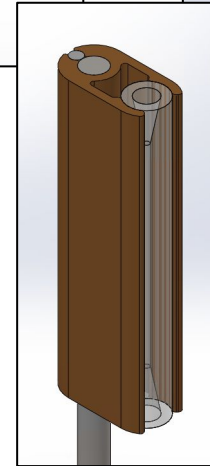
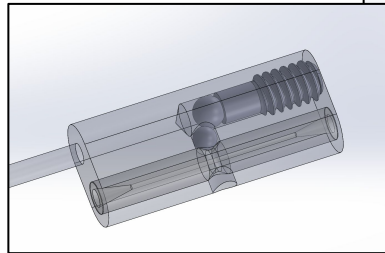
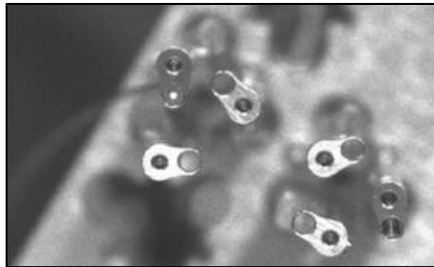
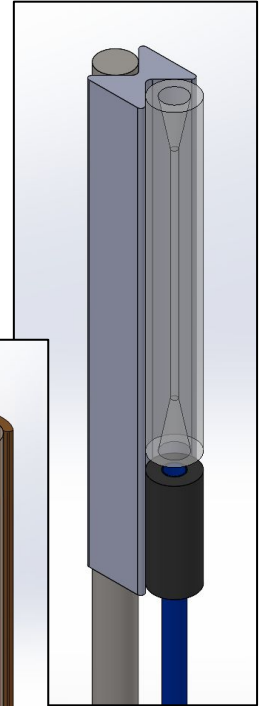
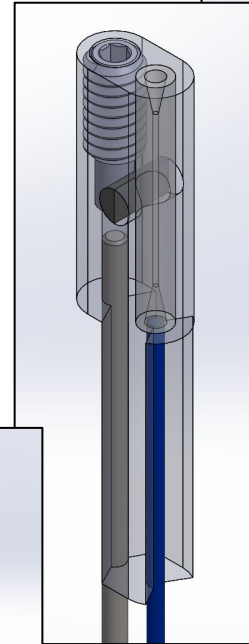
- Precise alignment
- Precise axial displacement
- Ease installation and rework



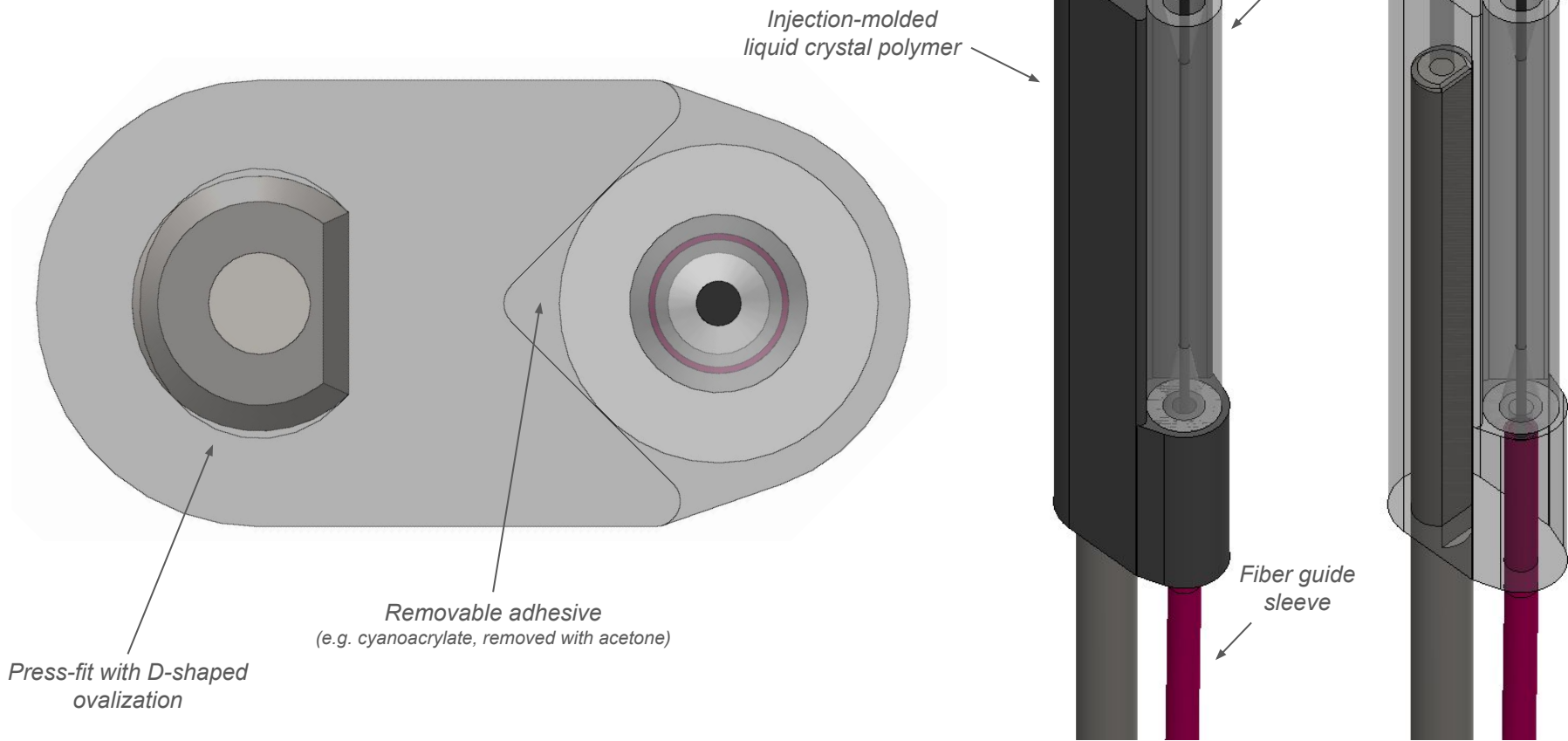


Conceptual Designs Prototypes

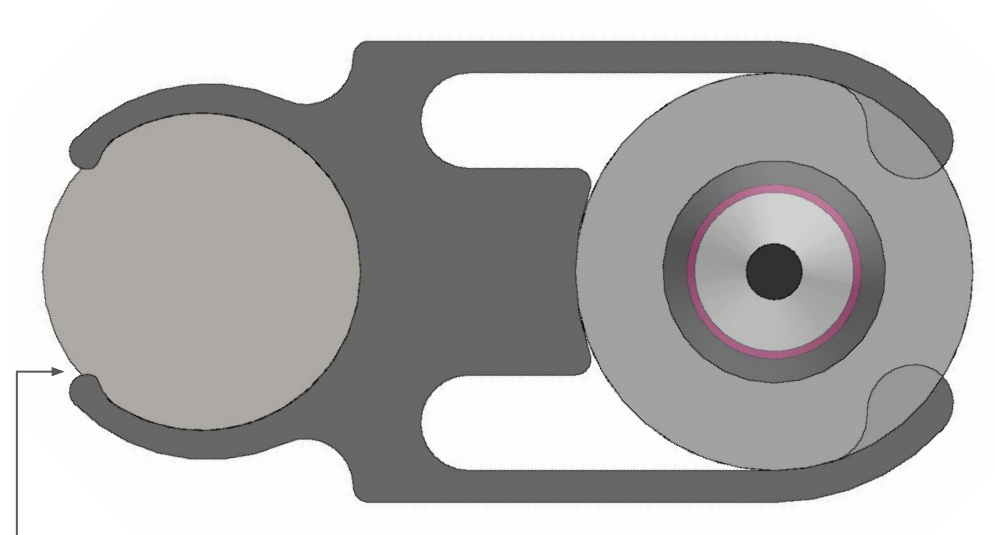
See Appendix



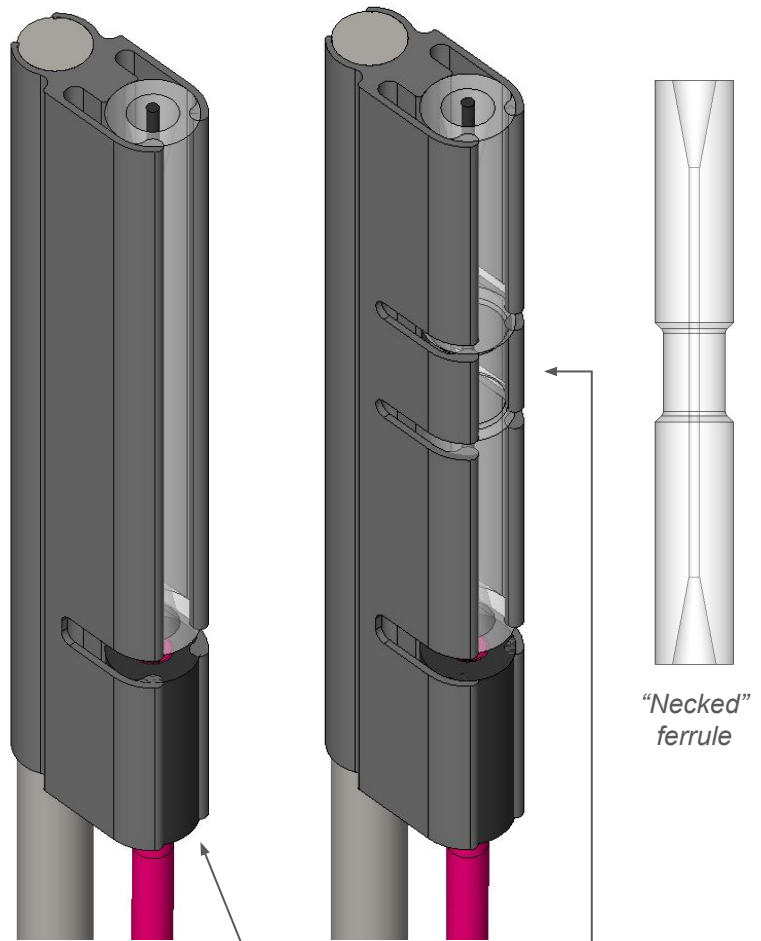
Fiber Arm A: V-Block with Ovalized Press-Fit



Fiber Arm B:
EDM with Spring Arms



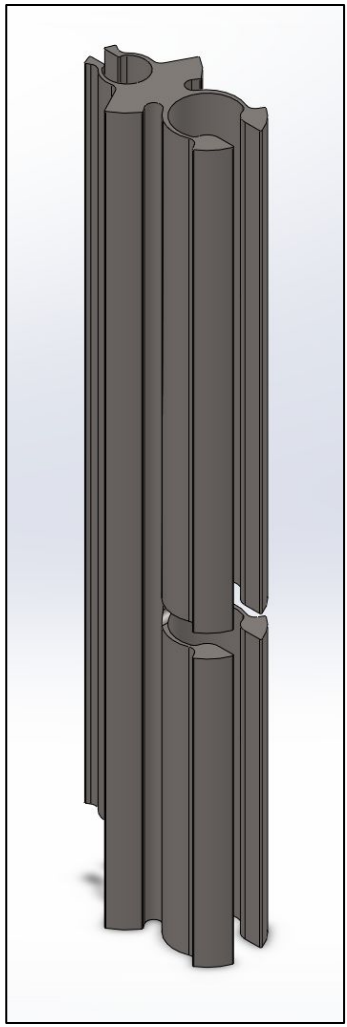
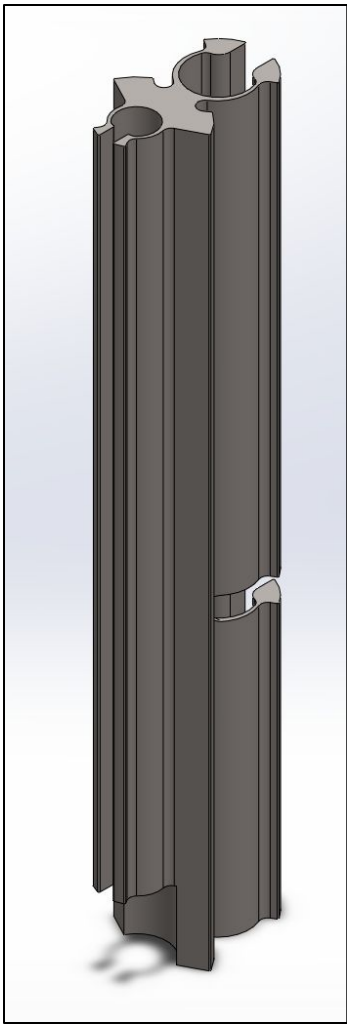
Adhesive



Adhesive
(Optional)

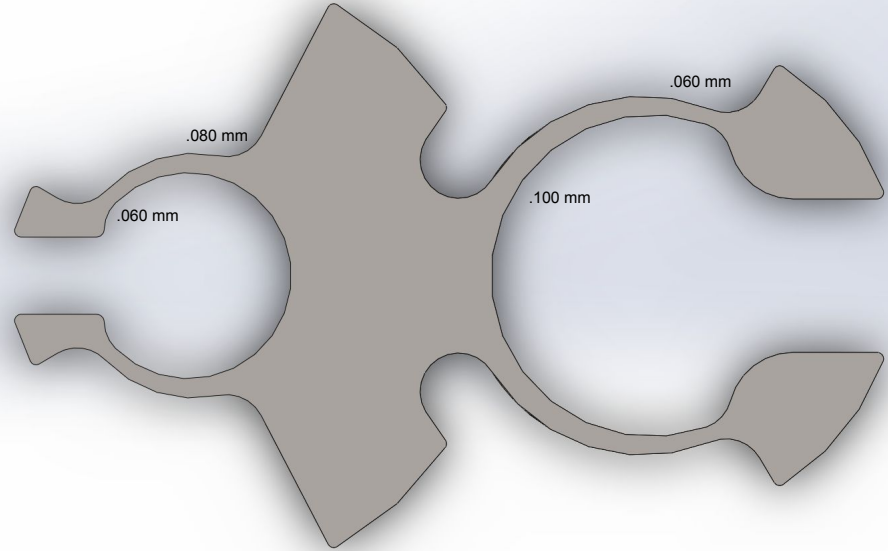
Axial constraint tabs
(Optional)

"Necked"
ferrule



Prototype EDM Fiber Arm

Ti-6Al-4V



DE-1000-0460A

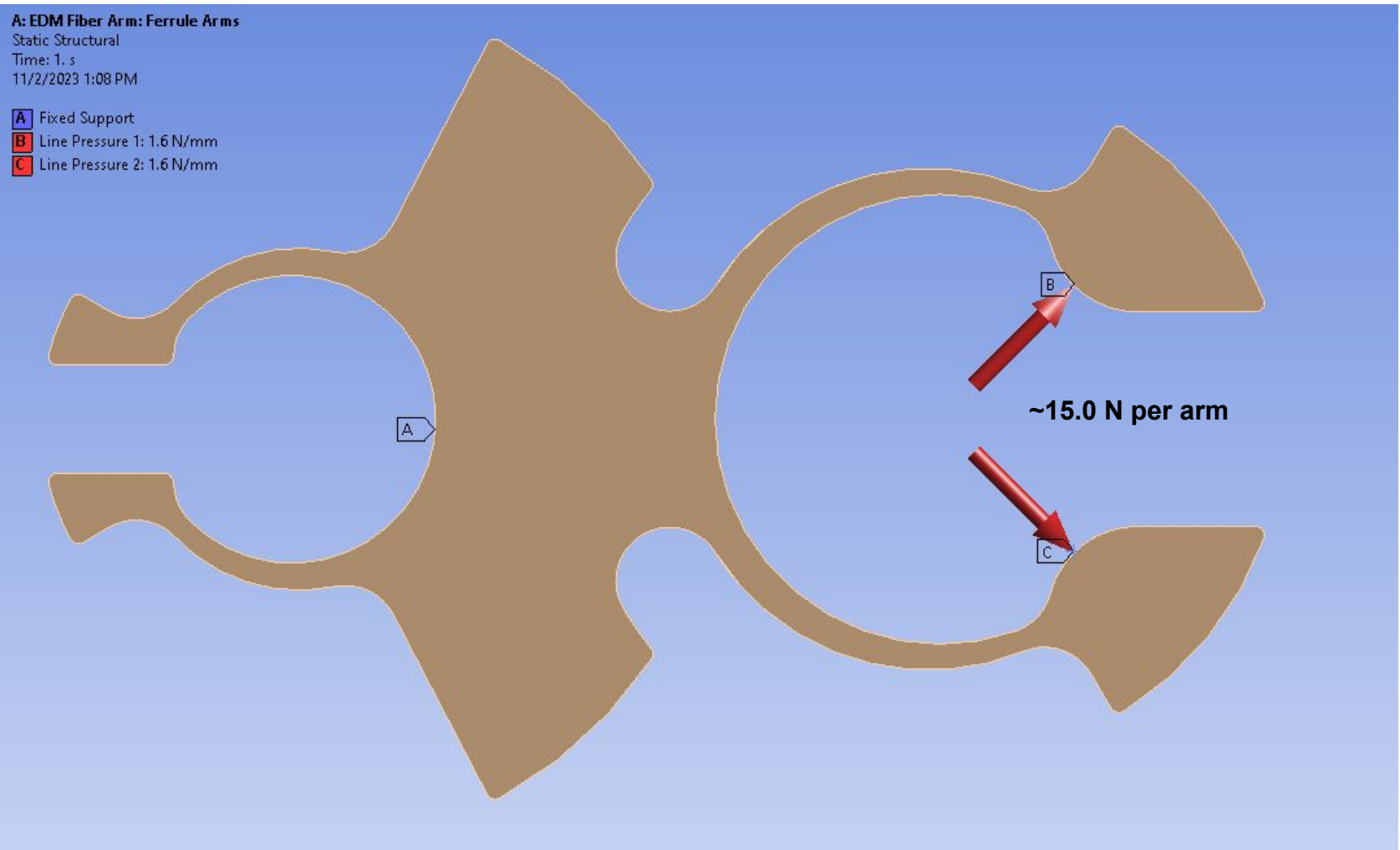
A: EDM Fiber Arm: Ferrule Arms

Static Structural

Time: 1. s

11/2/2023 1:08 PM

- A** Fixed Support
- B** Line Pressure 1: 1.6 N/mm
- C** Line Pressure 2: 1.6 N/mm



A: EDM Fiber Arm: Ferrule Arms

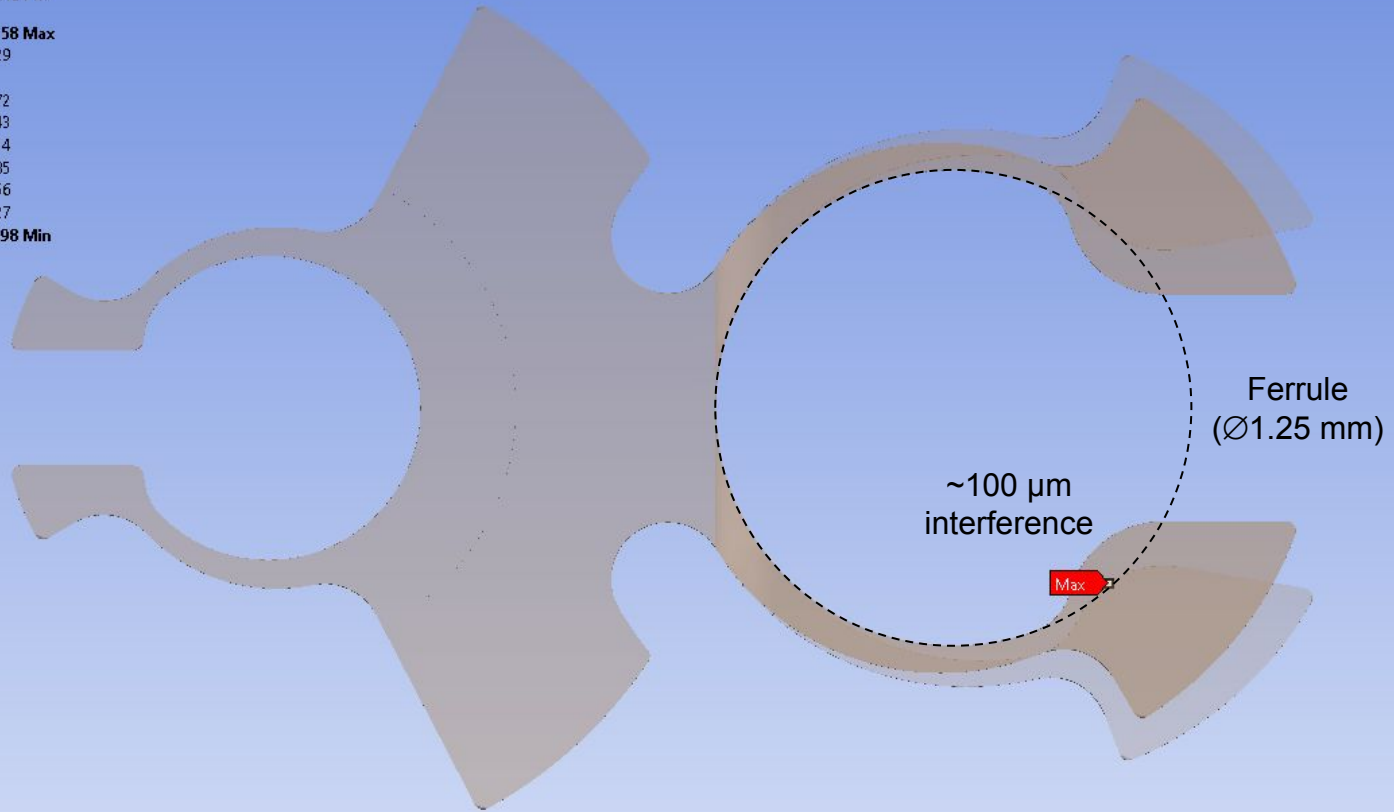
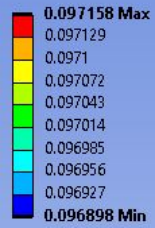
Total Deformation

Type: Total Deformation

Unit: mm

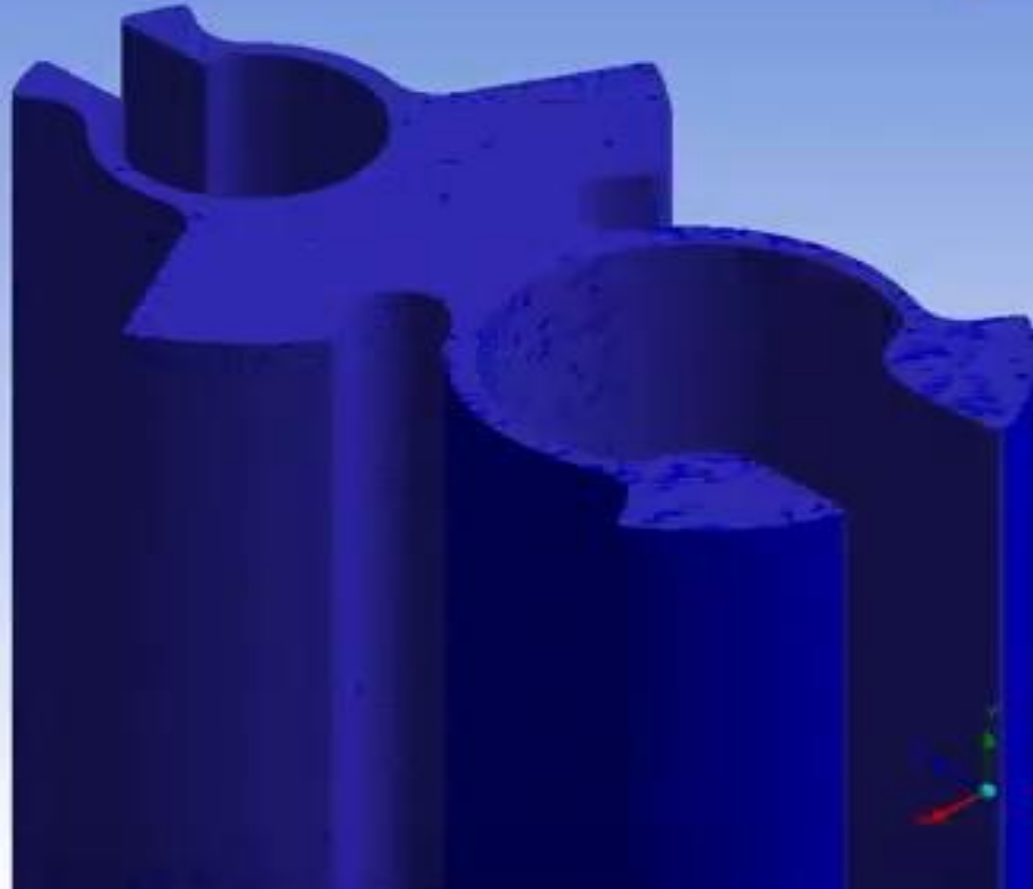
Time: 1 s

11/2/2023 1:12 PM



A: EDM Fiber Arm: Ferrule Arms
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 1.0103e-002
2/25/2024 8:59 PM

Ansys
2021 R2



Arms are tapered to distribute stress

A: EDM Fiber Arm: Ferrule Arms

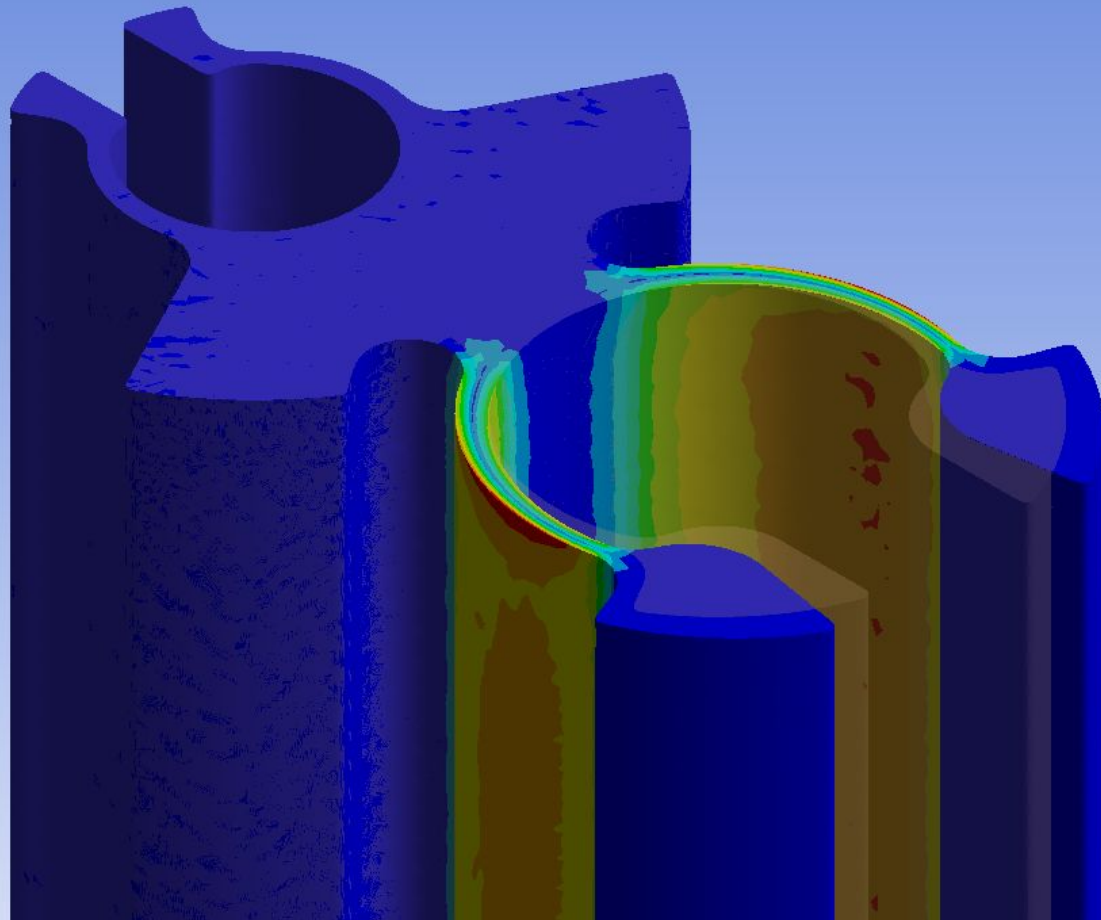
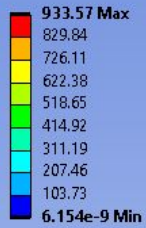
Equivalent Stress

Type: Equivalent (von-Mises) Stress

Unit: MPa

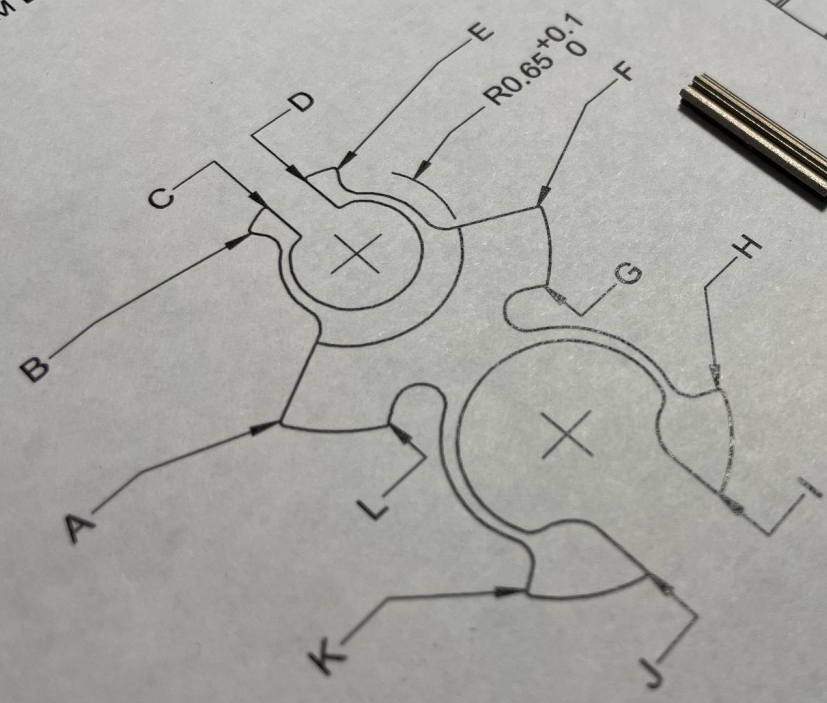
Time: 1 s

11/2/2023 1:11 PM



Arms are tapered to distribute stress

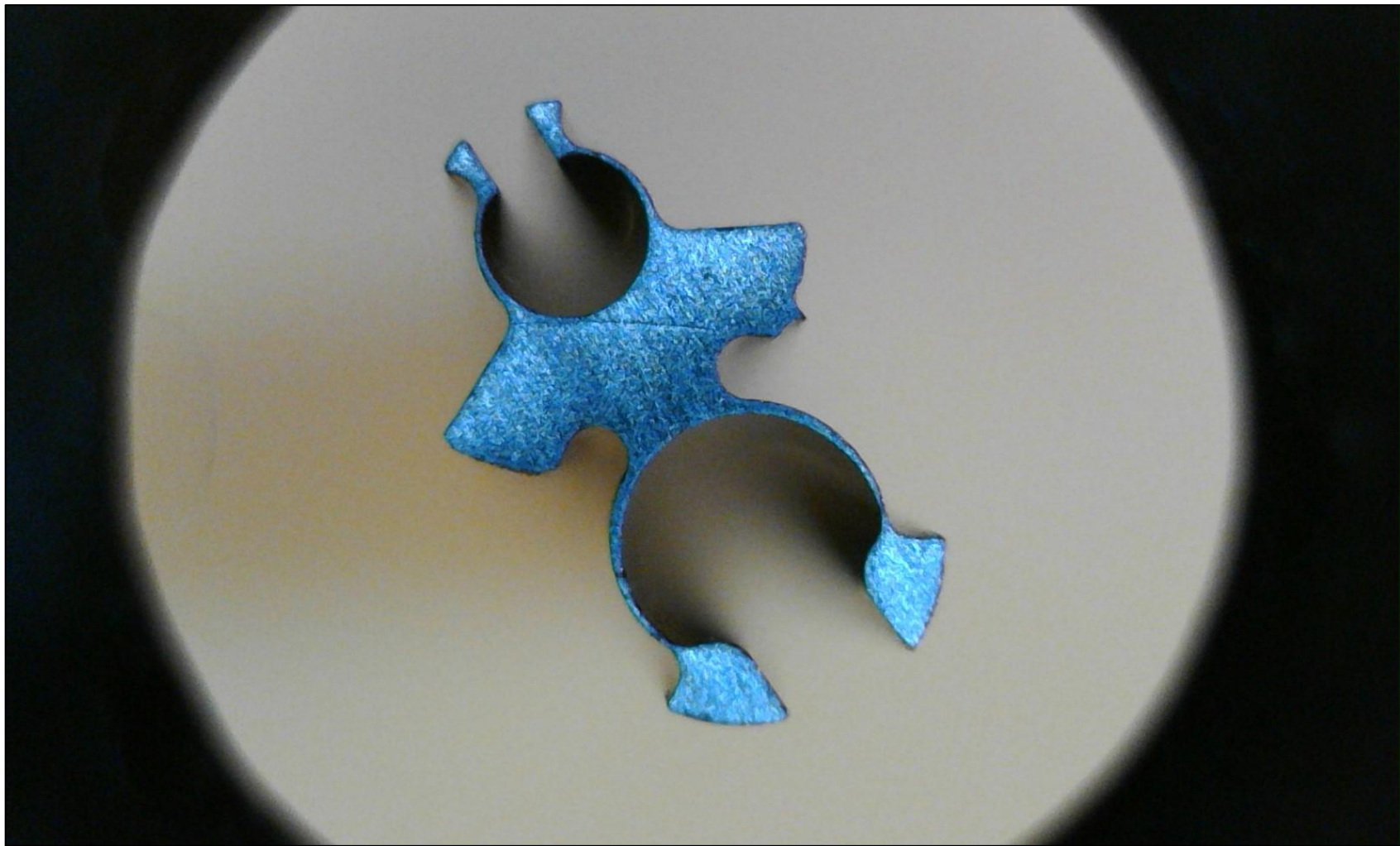
WITH PART NAME, NUMBER, AND
DIMENSION IS WITH RESPECT TO SOLID
FROM B → C, D → E, F → G, H

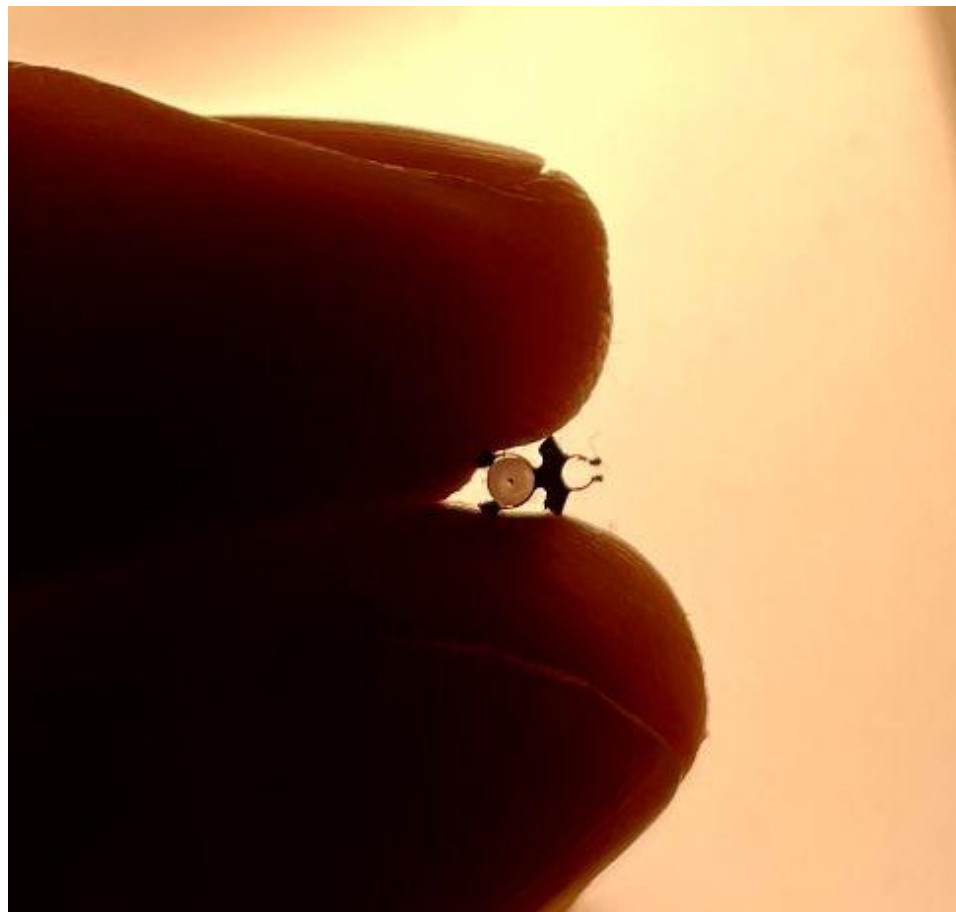
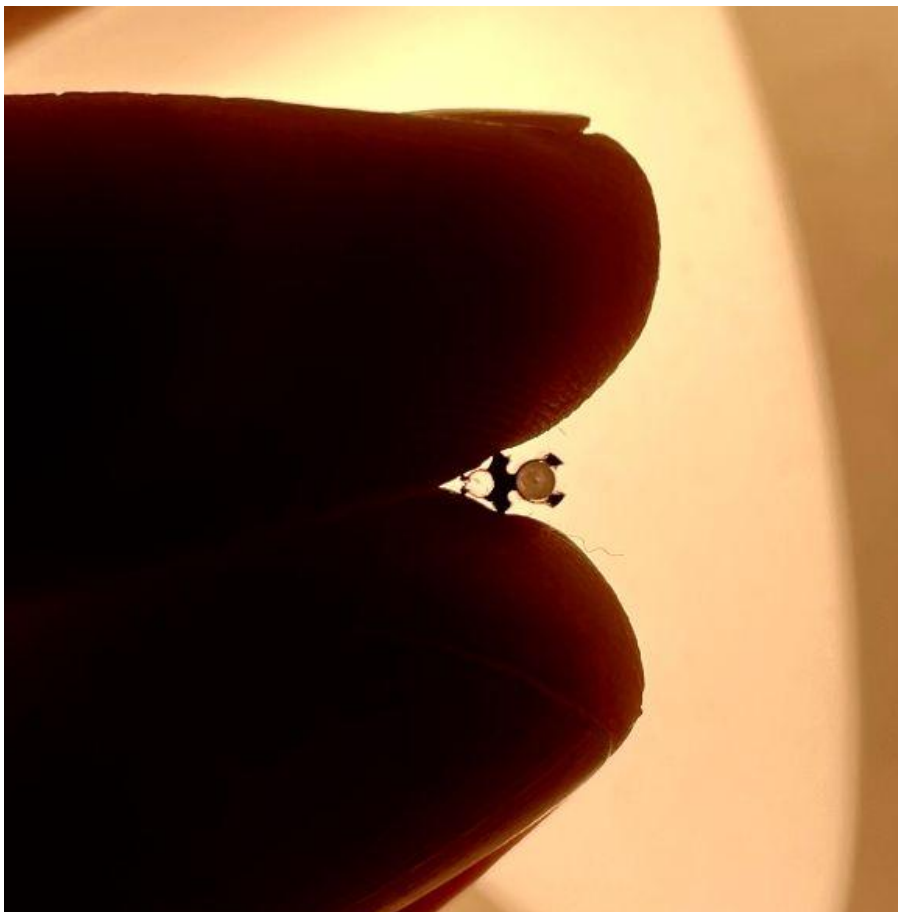


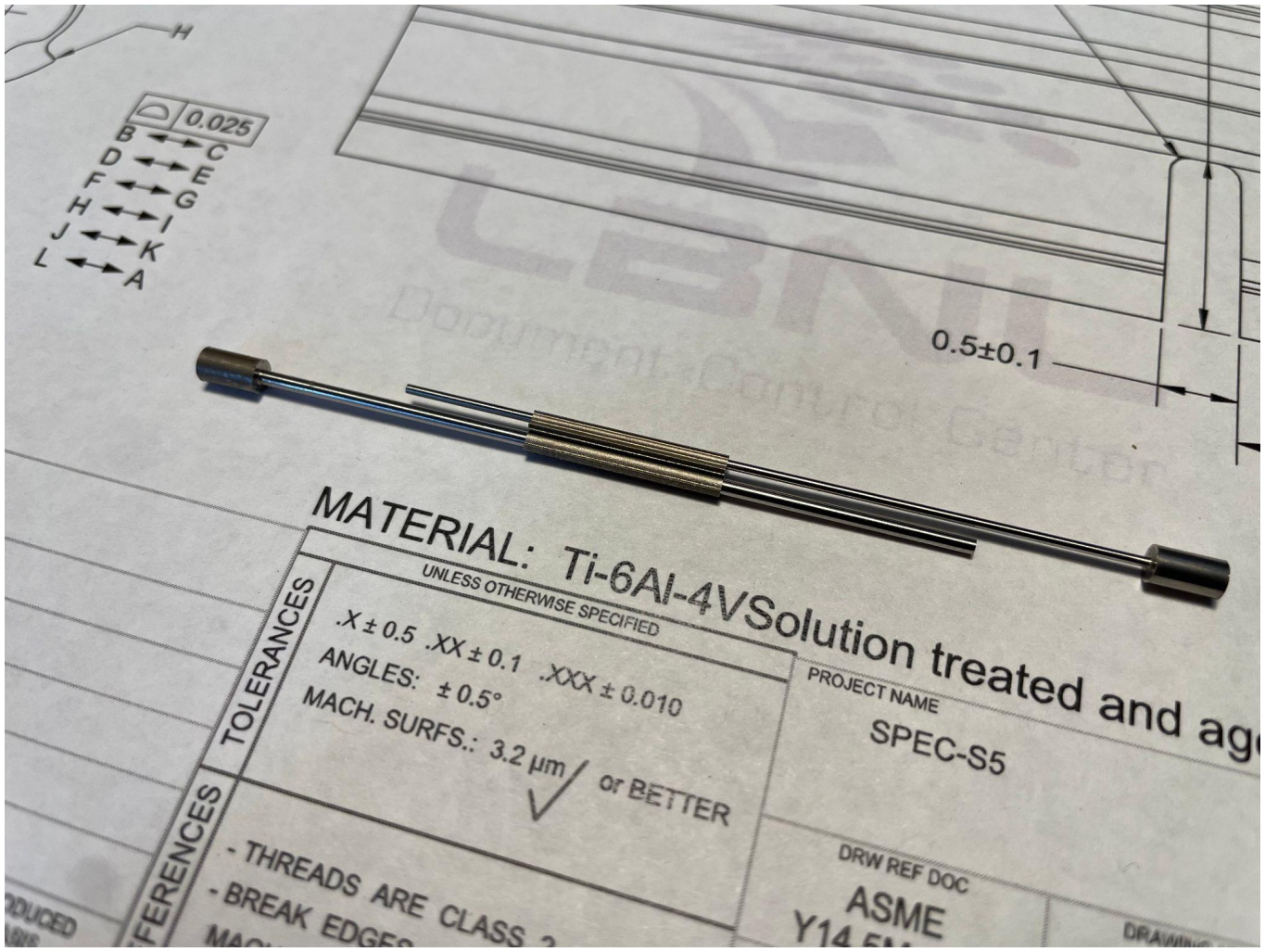
0.025	C
↔	E
↔	G
↔	I
↔	K
↔	A
B	
↔	D
↔	F
↔	H
↔	J
↔	L

0.01
B

107



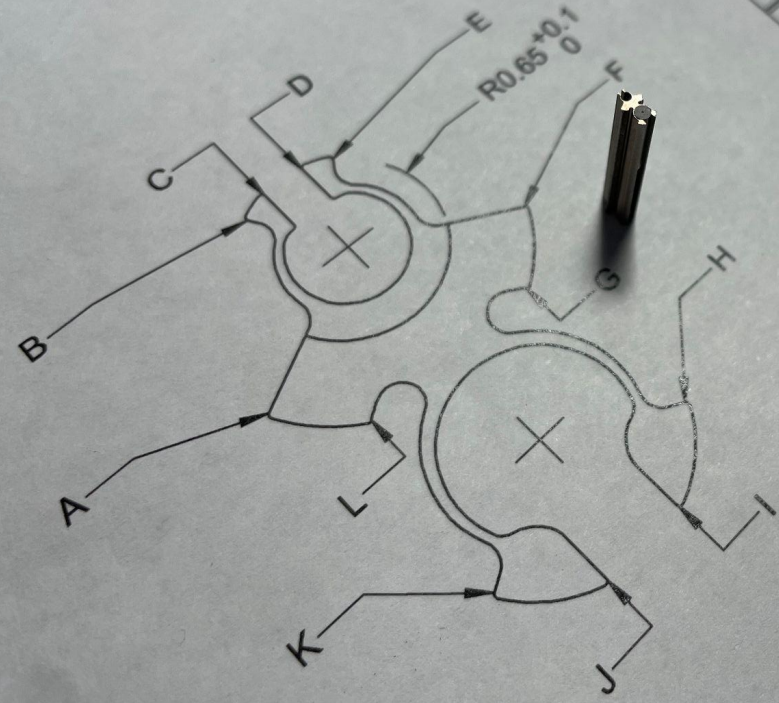




Relatively straightforward to QA

- Angular alignment
- Insertion forces

1 PART NAME, NUMBER, AND REVISION
ANCE IS WITH RESPECT TO SOLID MODEL
FROM B ↔ C, D ↔ E, F ↔ G, H ↔ I, J



0.025	C	E	G	I	K	A
↔	B	D	F	H	J	L

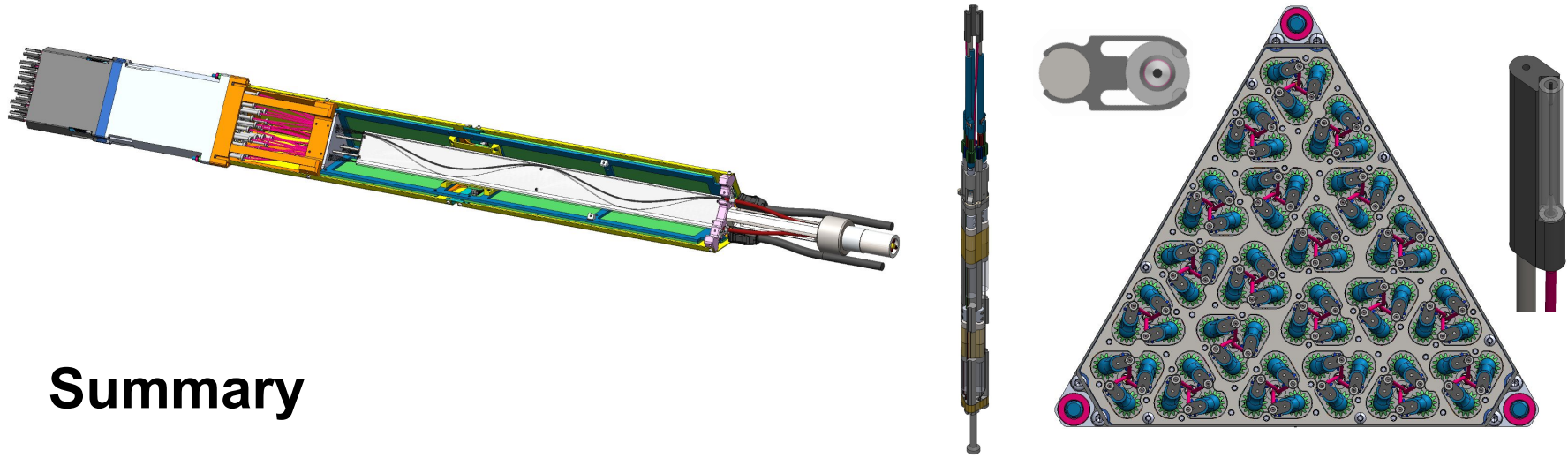
Stiction:

Ferrule

~3.1 N (0.7 lbf)

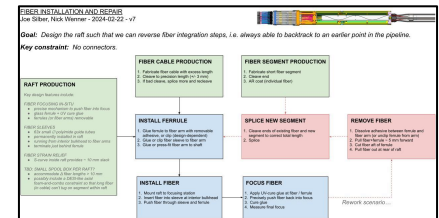
Shaft

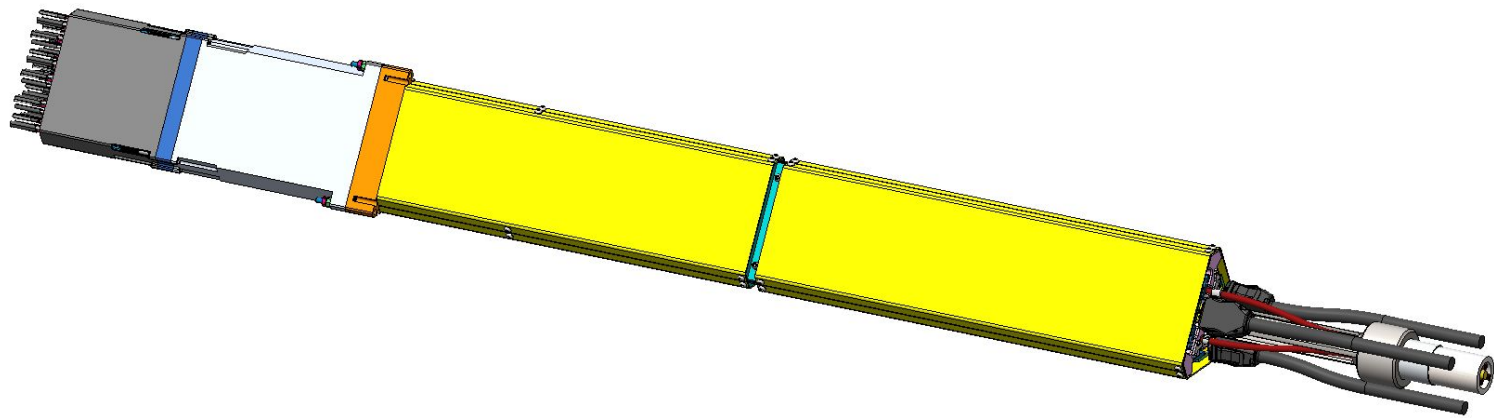
~0.7 N (0.2 lbf)



Summary

- Raft and positioner conceptual designs are well-developed
- Reversible fiber installation/repair processes are well-defined
- Several key constraint systems have been prototyped
- **Next steps:** Build and test functional prototype raft





Acknowledgements

Joseph Silber¹, Travis Mandeville², David Schlegel¹, William Van Shourt³, Ricardo Araujo⁴, Charles Baltay⁵, Robert Besuner¹, Emily Farr⁶, Julien Guy¹, Jean-Paul Kneib⁴, Maxime Rombach⁴, Claire Poppett³, Michael Schubnell⁷, Markus Thurneysen⁴, Sarah Tuttle², Greg Tarlè⁷

¹Lawrence Berkeley National Laboratory (LBNL)

²University of Washington

³Space Sciences Laboratory (SSL)

⁴École Polytechnique Fédérale de Lausanne (EPFL)

⁵Yale University

⁶University of Colorado

⁷University of Michigan

References

- *Specification for high density fiber robot raft modules* (2024)
<https://zenodo.org/records/10688871>
- *25,000 optical fiber positioning robots for next-generation cosmology* (2022)
<https://arxiv.org/abs/2212.07908>
- *The Robotic Multi-Object Focal Plane System of the Dark Energy Spectroscopic Instrument (DESI)* (2022)
<https://arxiv.org/abs/2205.09014>

Appendix

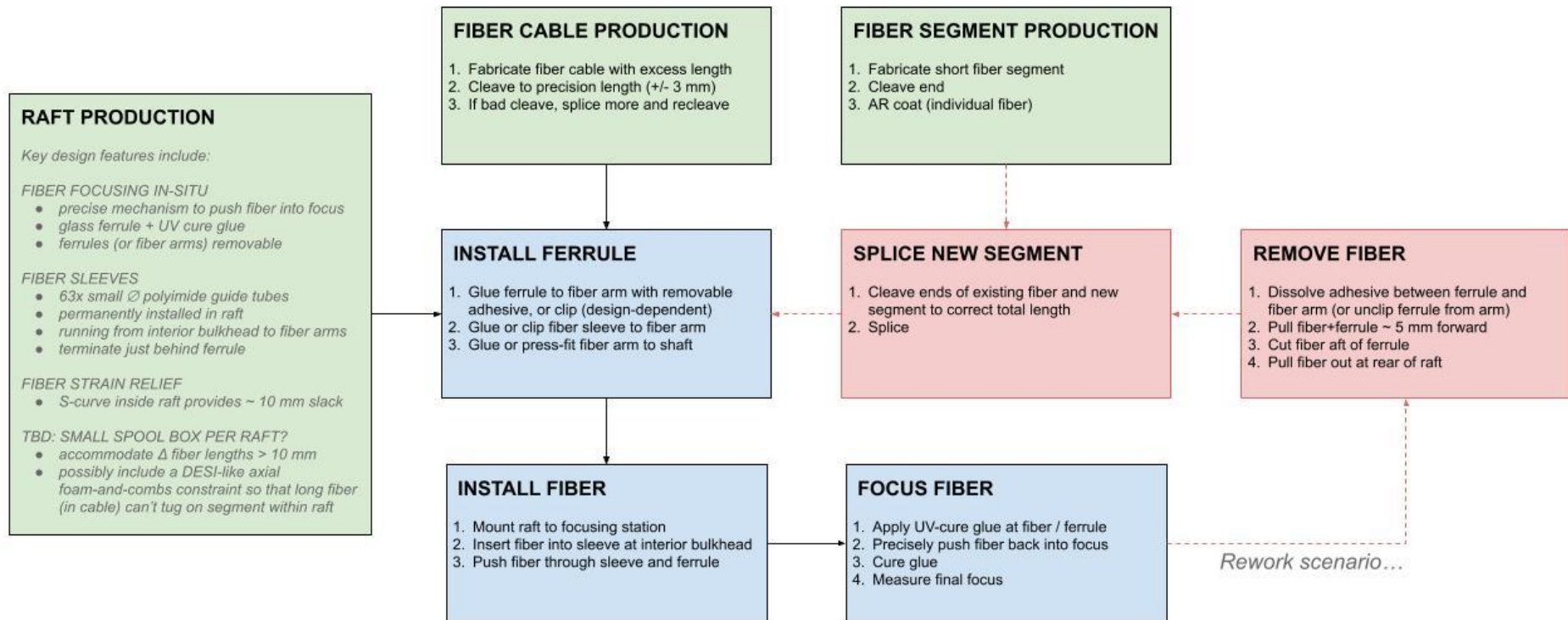
FIBER INSTALLATION AND REPAIR

Joe Silber, Nick Wenner - 2024-02-22 - v7



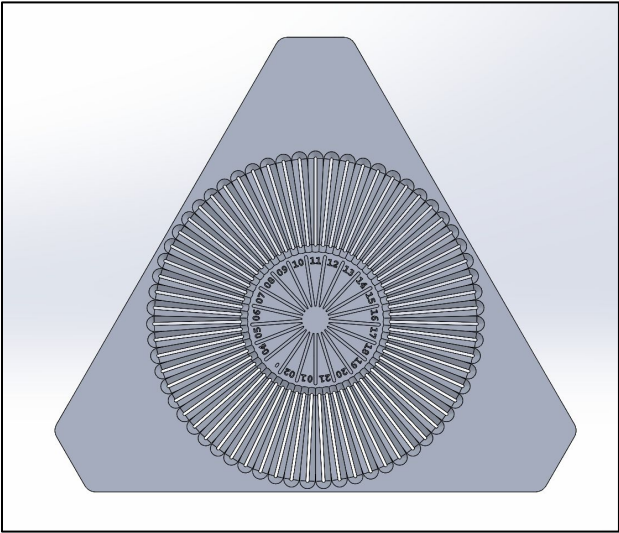
Goal: Design the raft such that we can reverse fiber integration steps, i.e. always able to backtrack to an earlier point in the pipeline.

Key constraint: No connectors.

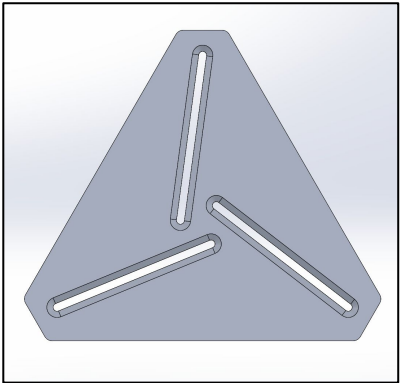
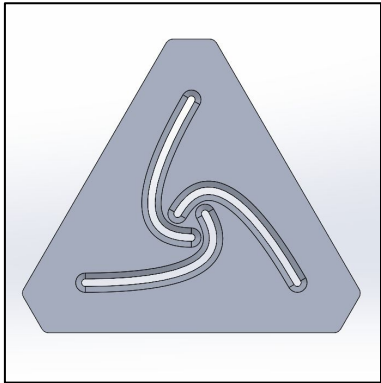
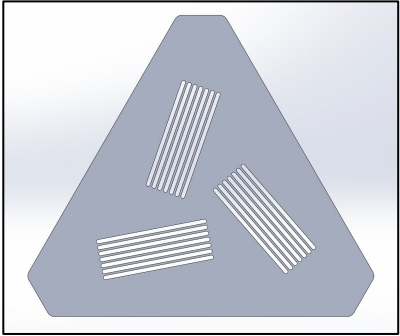
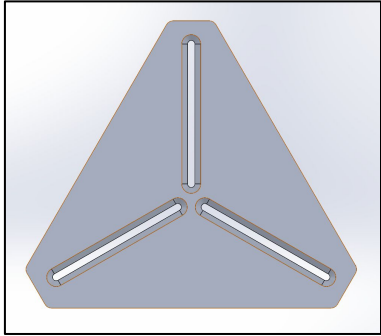


Alternative Fiber Strain Relief Designs

Fiber Guide Examples

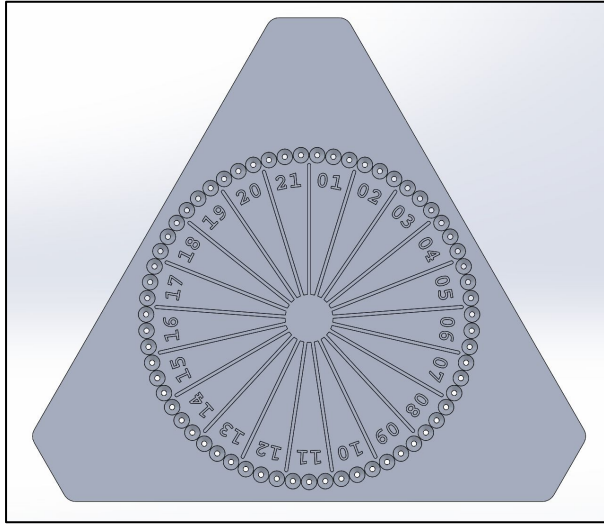


Individual Fibers

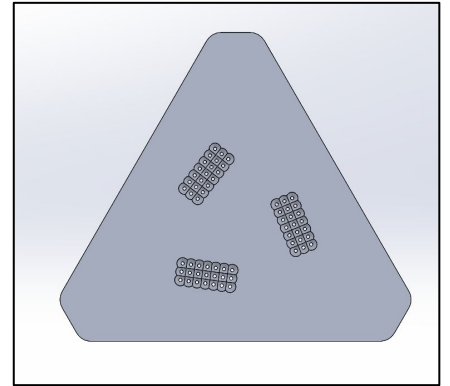
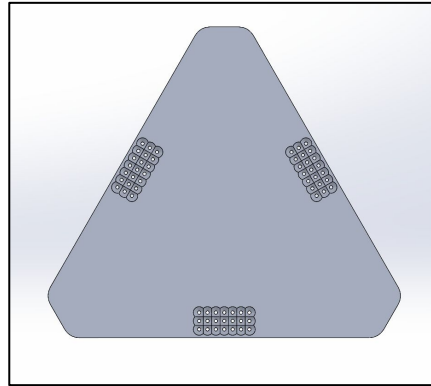
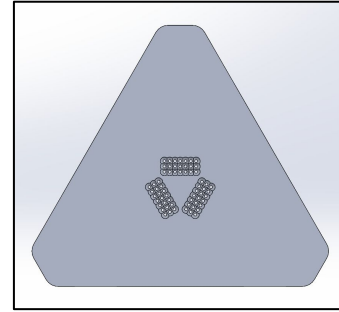


Bundled Fibers (21 ea)

Forward Bulkhead Examples

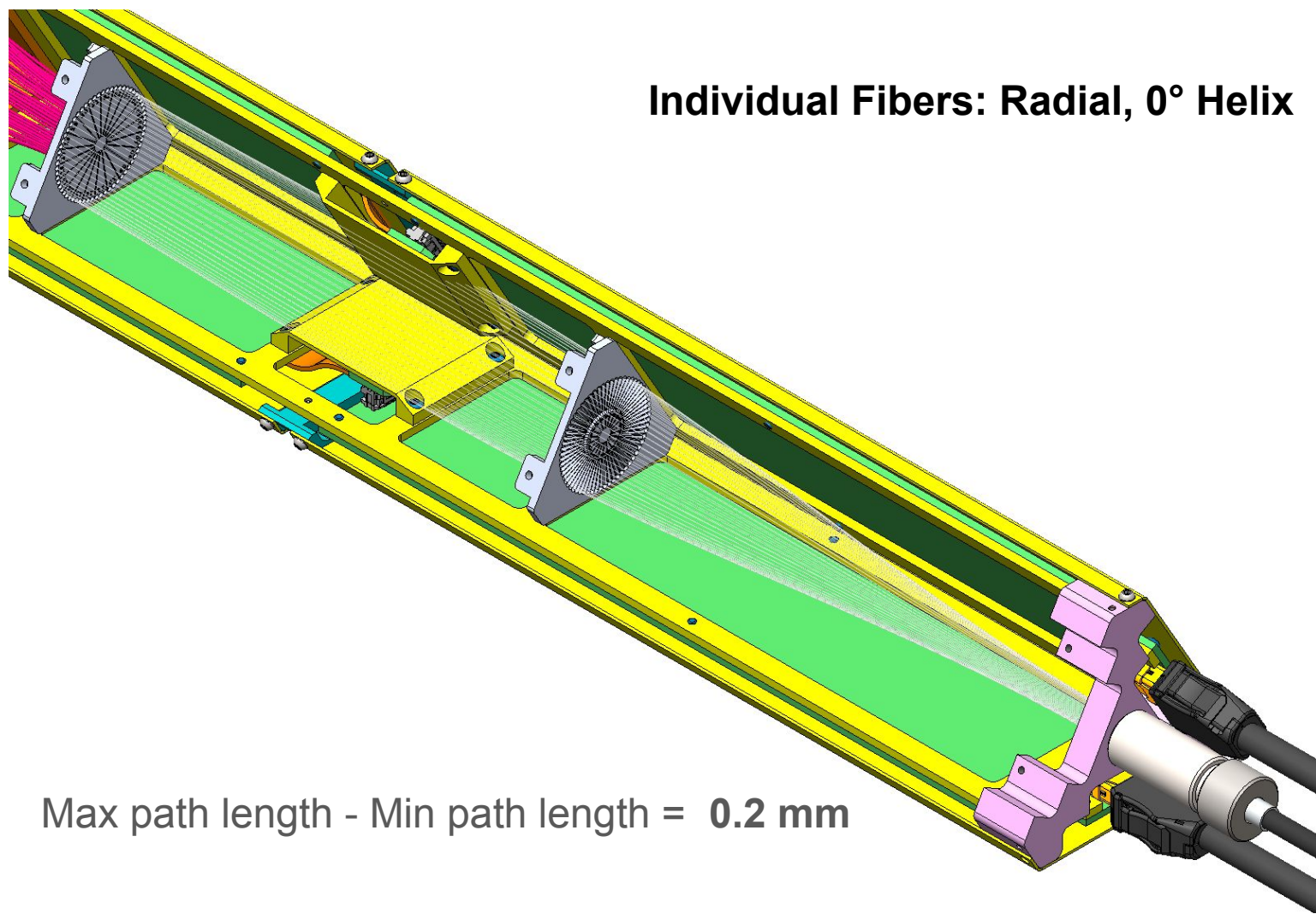


Individual Fibers



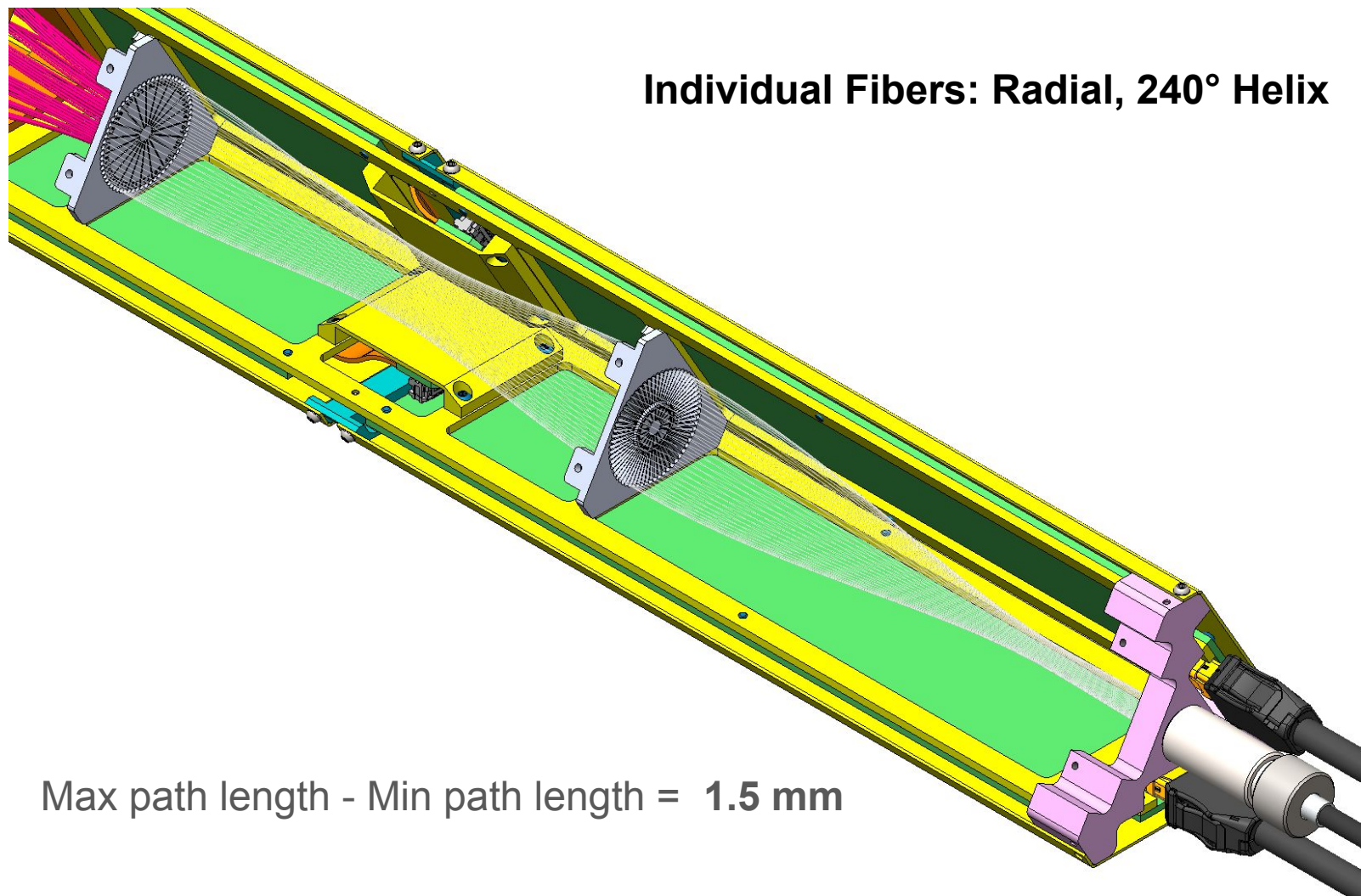
Bundled Fibers (21 ea)

Individual Fibers: Radial, 0° Helix



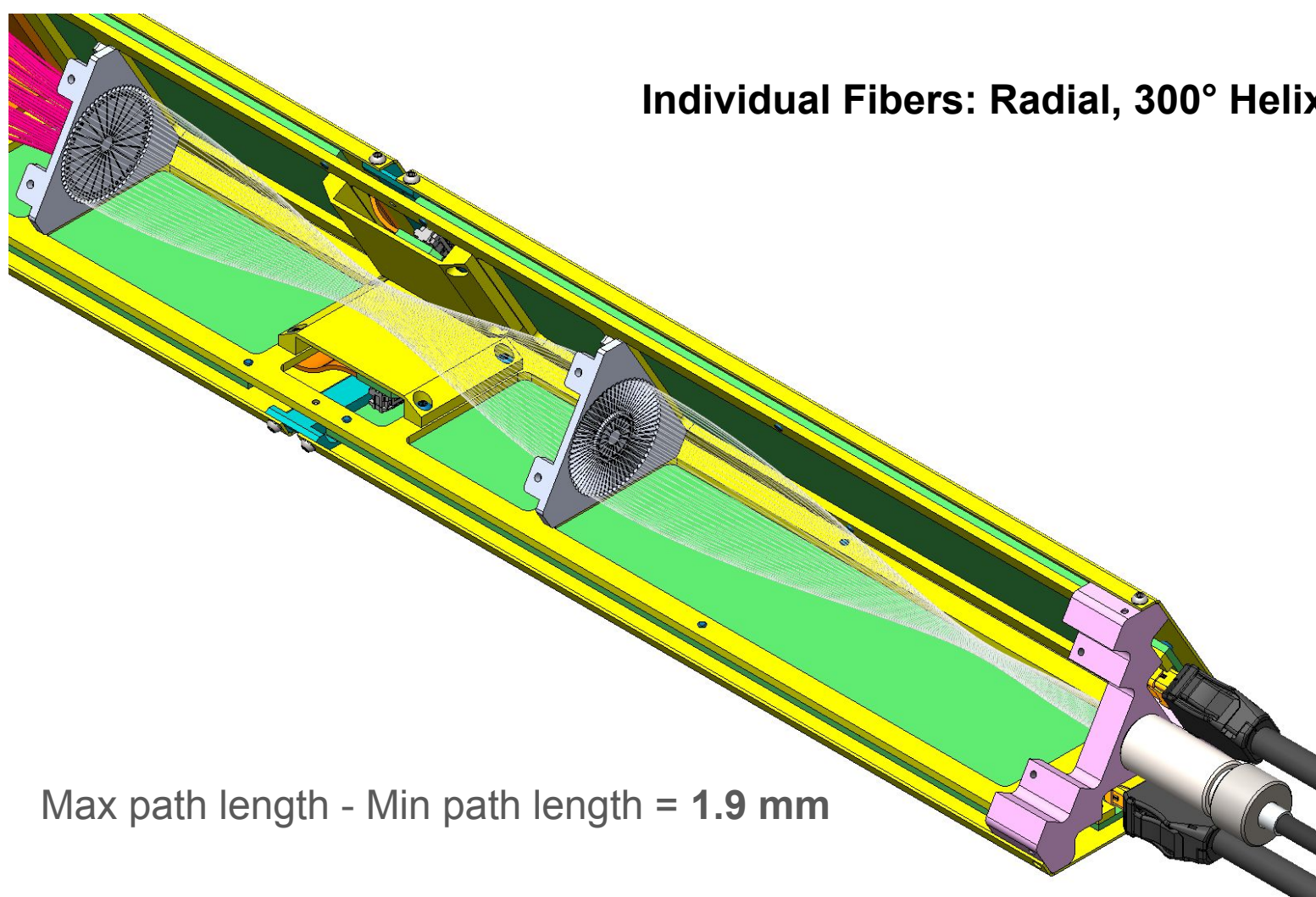
Max path length - Min path length = **0.2 mm**

Individual Fibers: Radial, 240° Helix



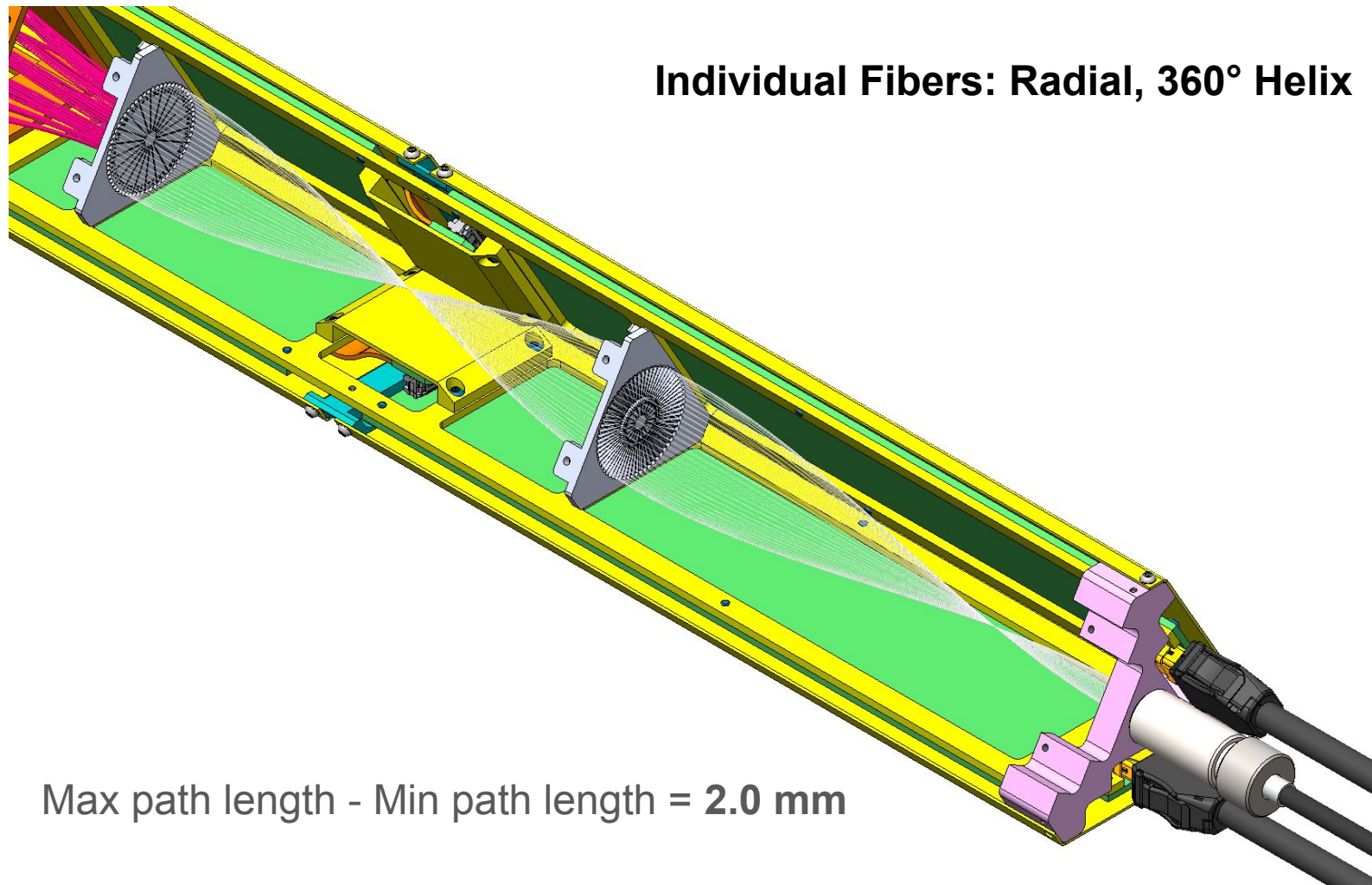
Max path length - Min path length = **1.5 mm**

Individual Fibers: Radial, 300° Helix

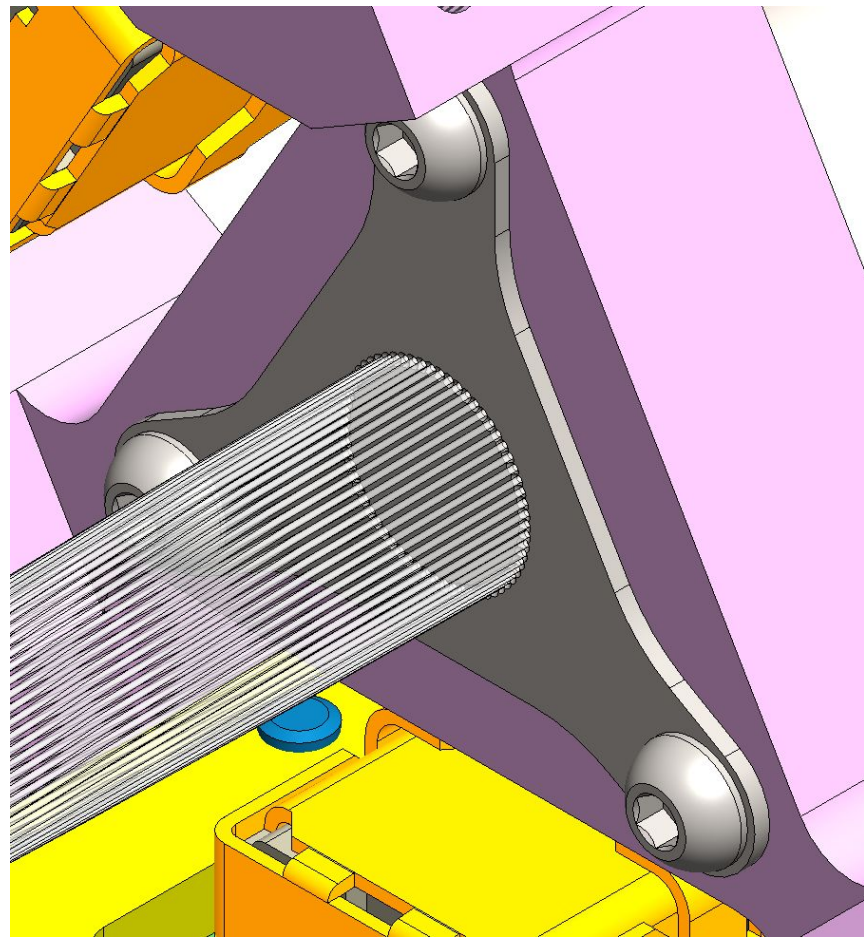
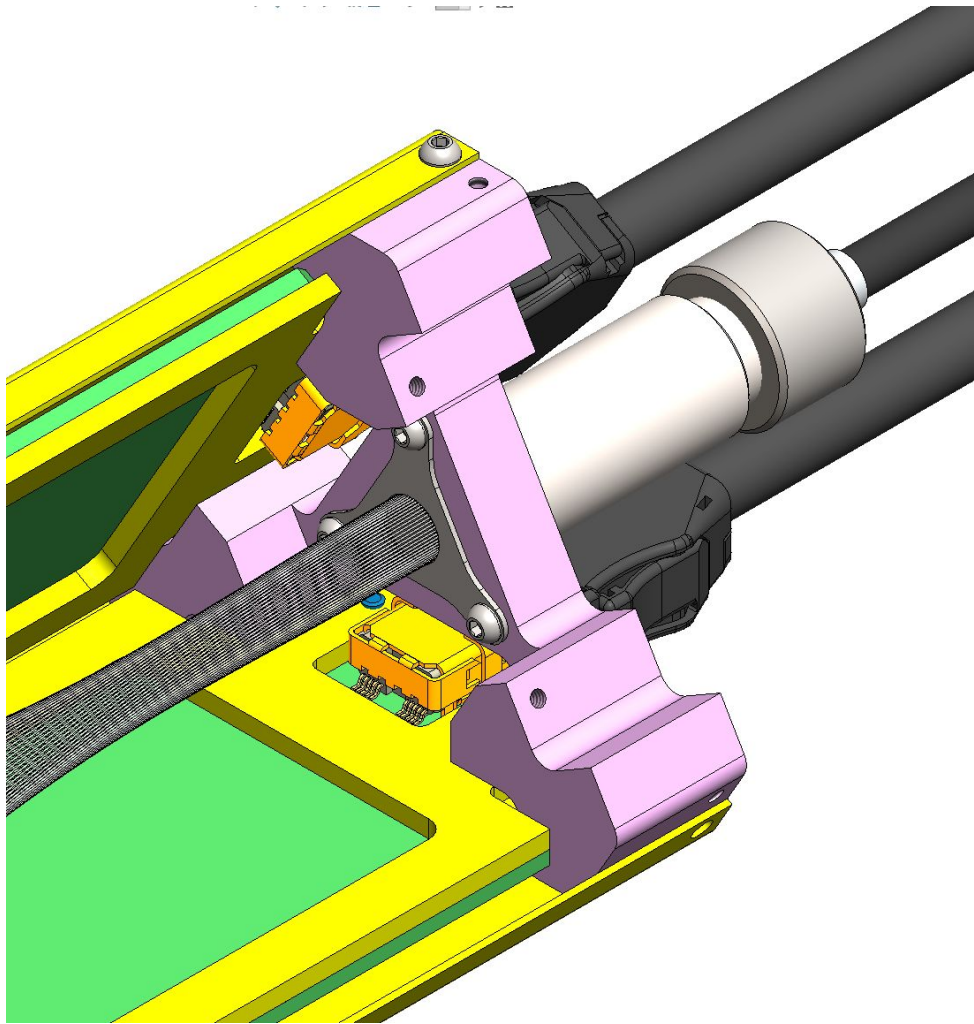


Max path length - Min path length = 1.9 mm

Individual Fibers: Radial, 360° Helix



Max path length - Min path length = 2.0 mm

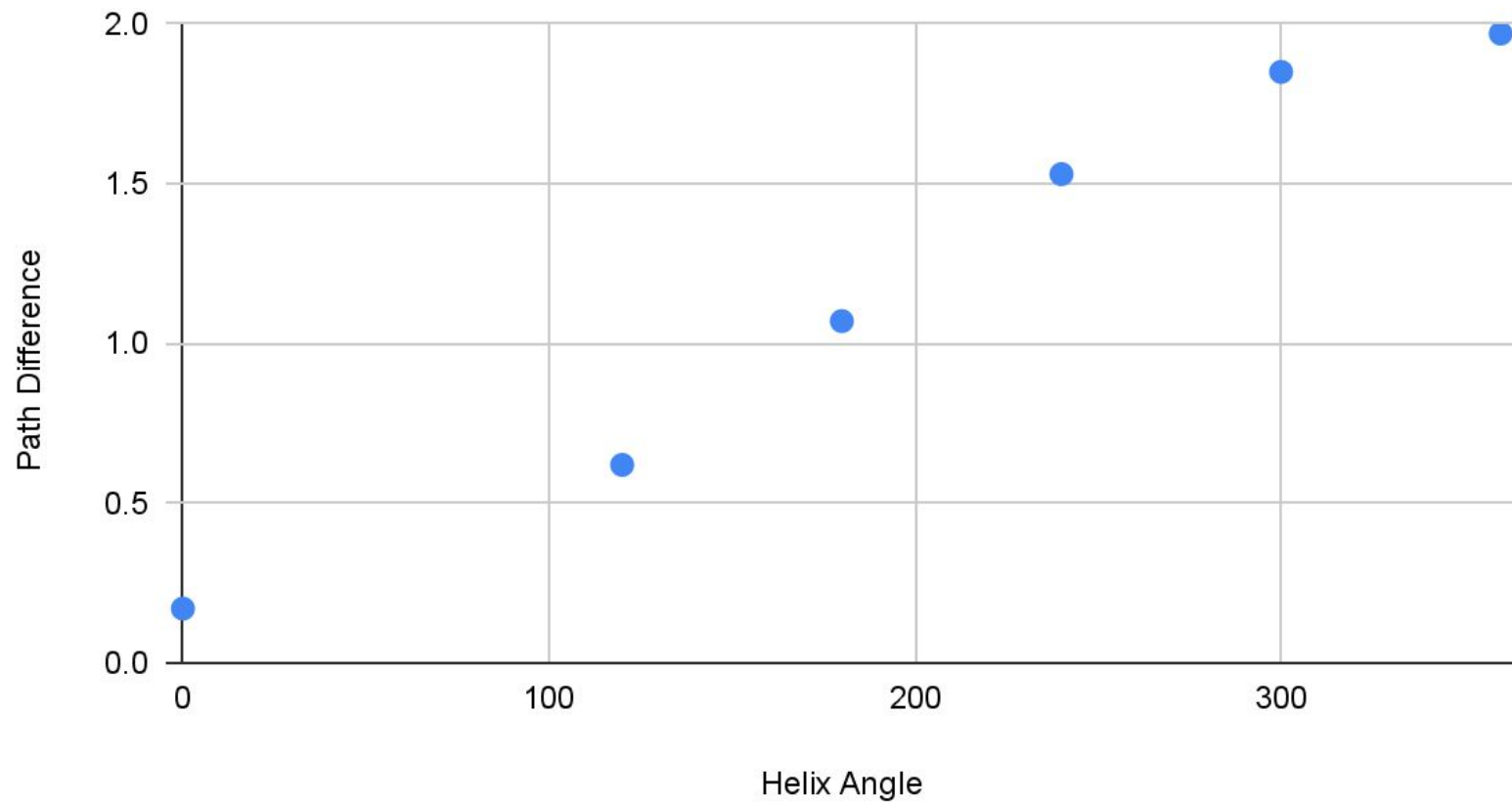




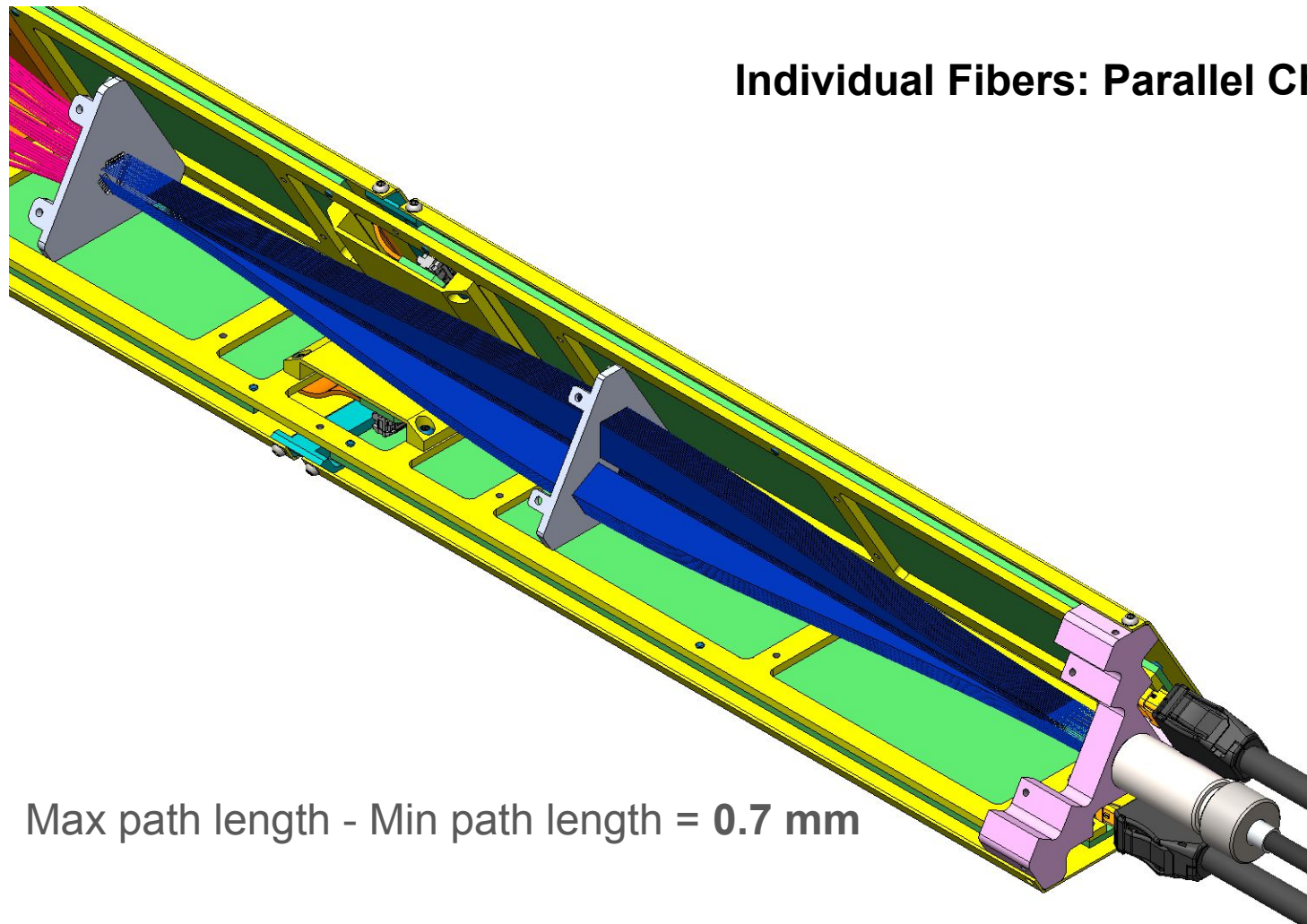
Kathryn Lybarger Mede News

30,000 member his-cuba union
and being elected president
of the California Labor Federation.

Individual Fiber: Radial Guide



Individual Fibers: Parallel Channels



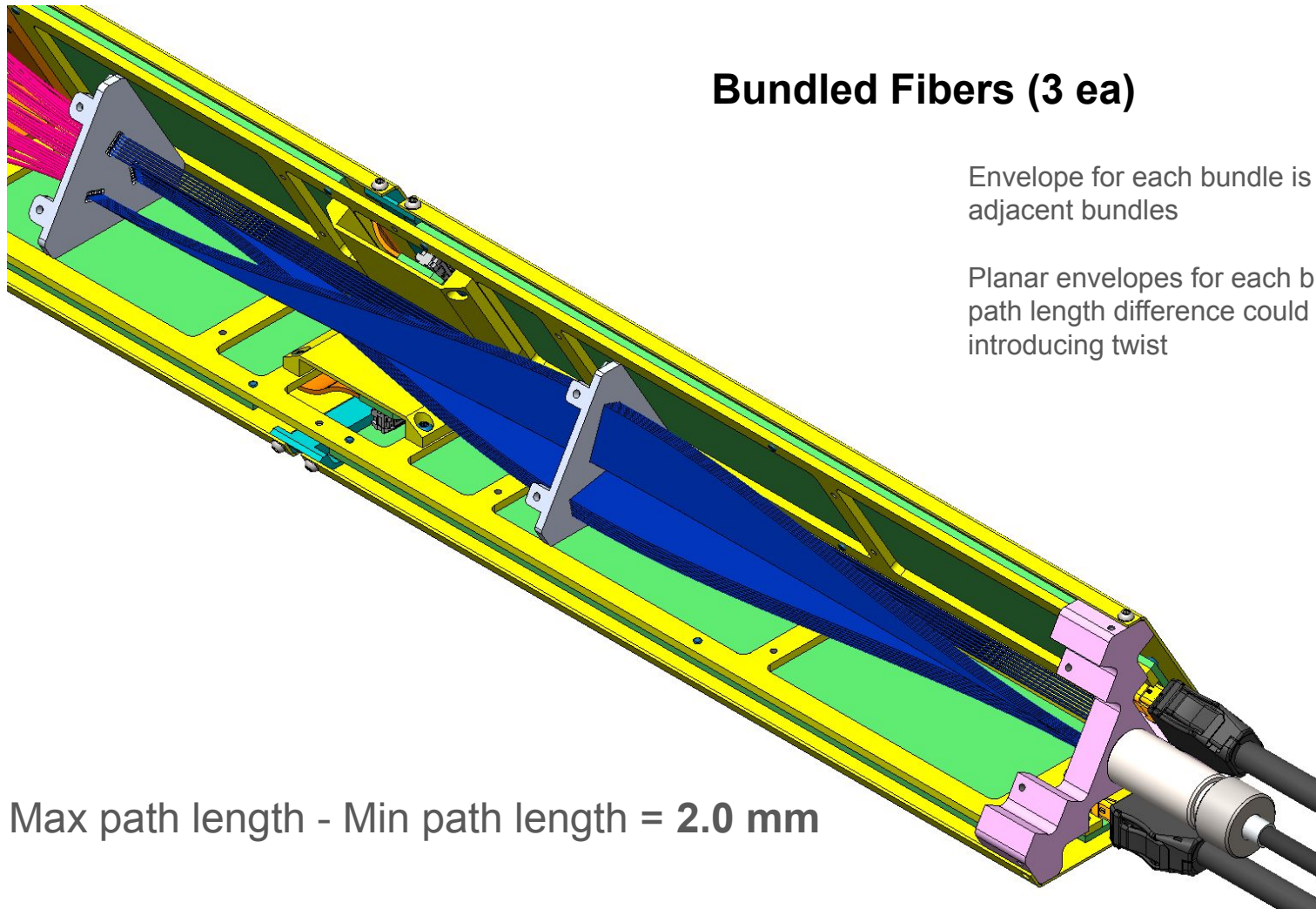
Max path length - Min path length = **0.7 mm**

Bundled Fibers (3 ea)

Envelope for each bundle is non-intersecting with adjacent bundles

Planar envelopes for each bundle are shown. Larger path length difference could be achieved by introducing twist

Max path length - Min path length = 2.0 mm

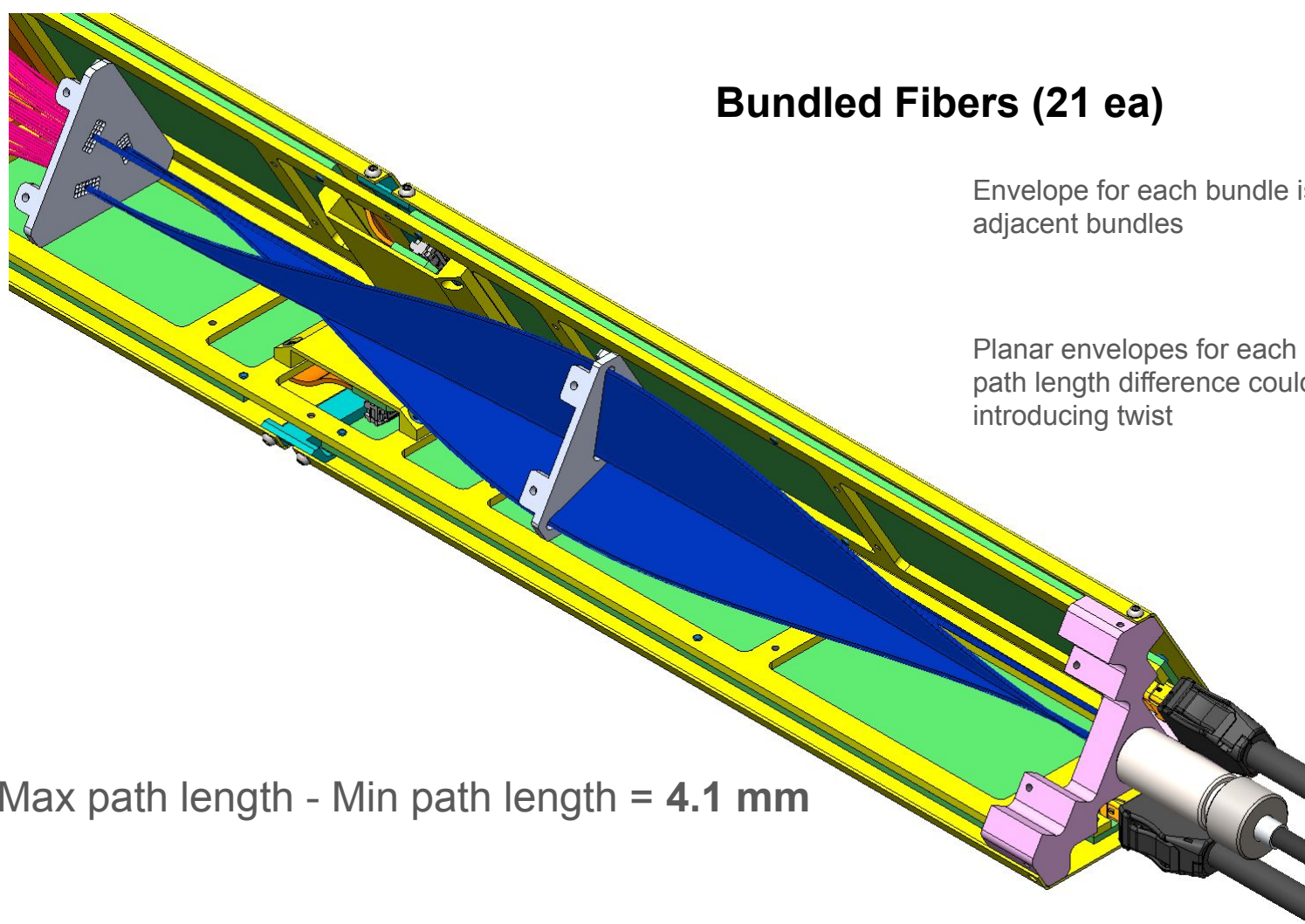


Bundled Fibers (21 ea)

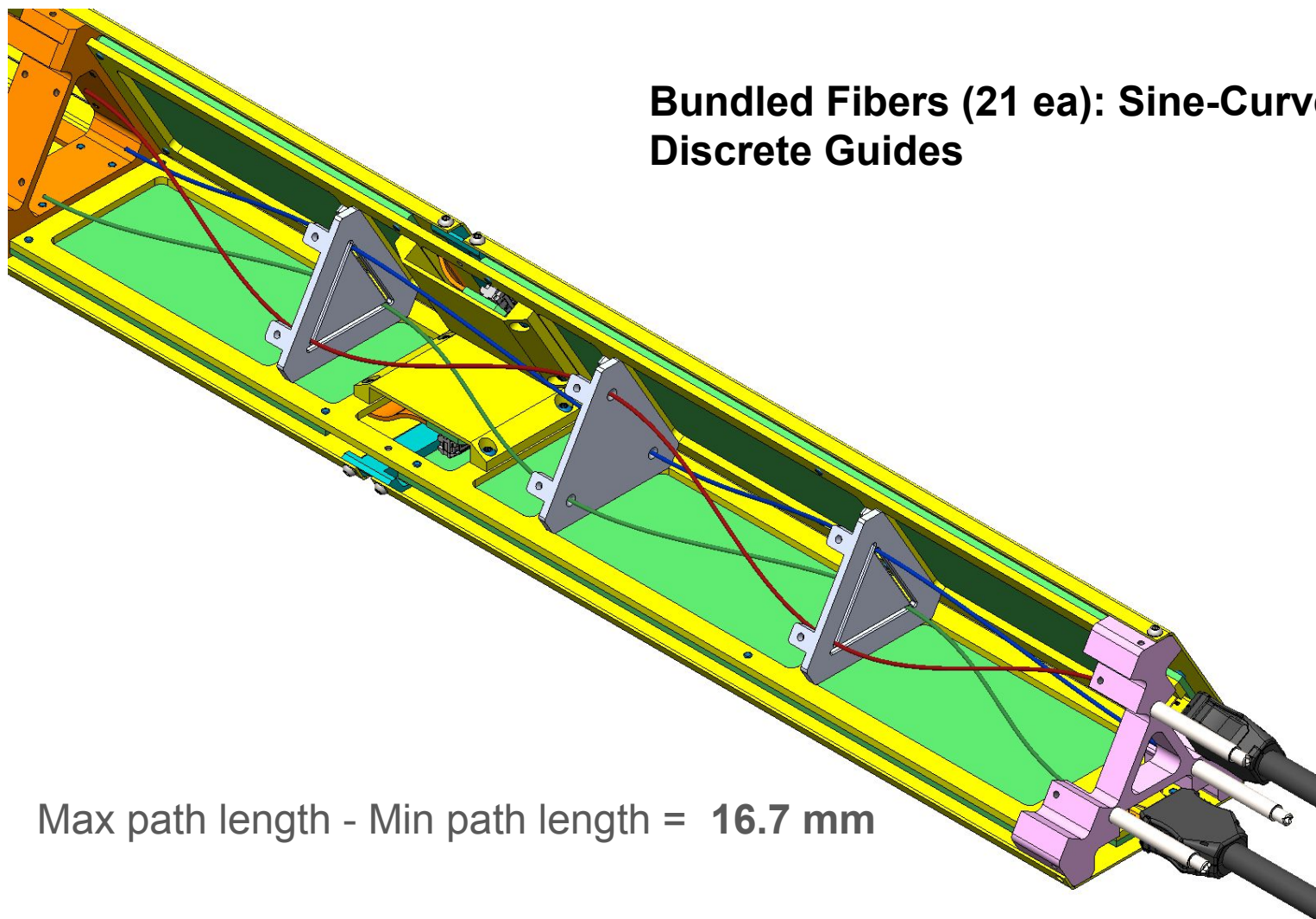
Envelope for each bundle is non-intersecting with adjacent bundles

Planar envelopes for each bundle are shown. Larger path length difference could be achieved by introducing twist

Max path length - Min path length = 4.1 mm

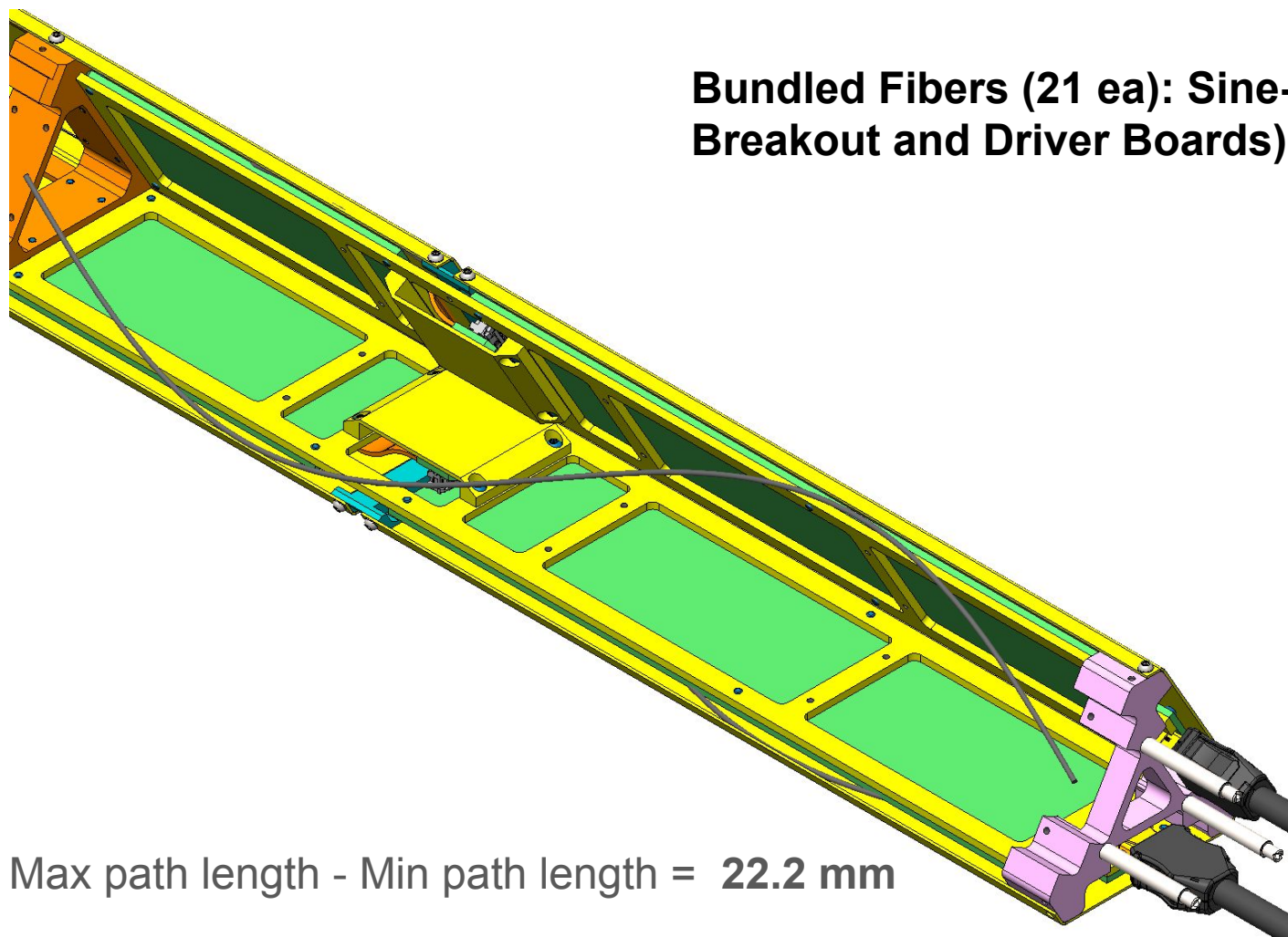


Bundled Fibers (21 ea): Sine-Curve with Discrete Guides



Max path length - Min path length = **16.7 mm**

Bundled Fibers (21 ea): Sine-Curve (Outside Breakout and Driver Boards)



Max path length - Min path length = 22.2 mm

Alternative Fiber Arm Designs

EDM Fiber Arm (DE-1000-0460A)

Ferrule Arms: 0.060 mm - 0.100 mm taper

Shaft Arms: 0.060 mm - 0.080 mm taper

B: EDM Fiber Arm: Shaft Arms

Static Structural

Time: 1. s

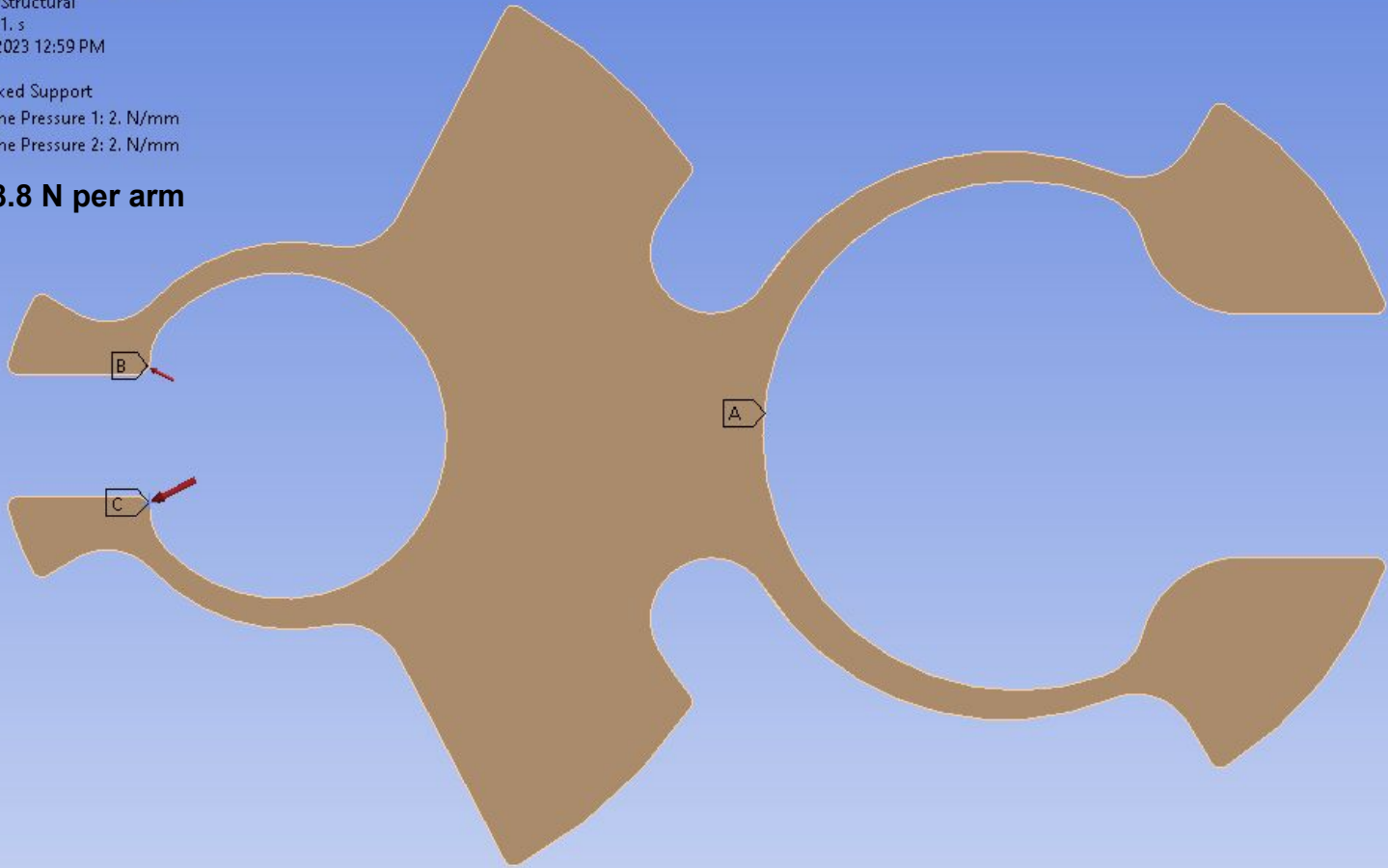
11/2/2023 12:59 PM

A Fixed Support

B Line Pressure 1: 2. N/mm

C Line Pressure 2: 2. N/mm

~28.8 N per arm



B: EDM Fiber Arm: Shaft Arms

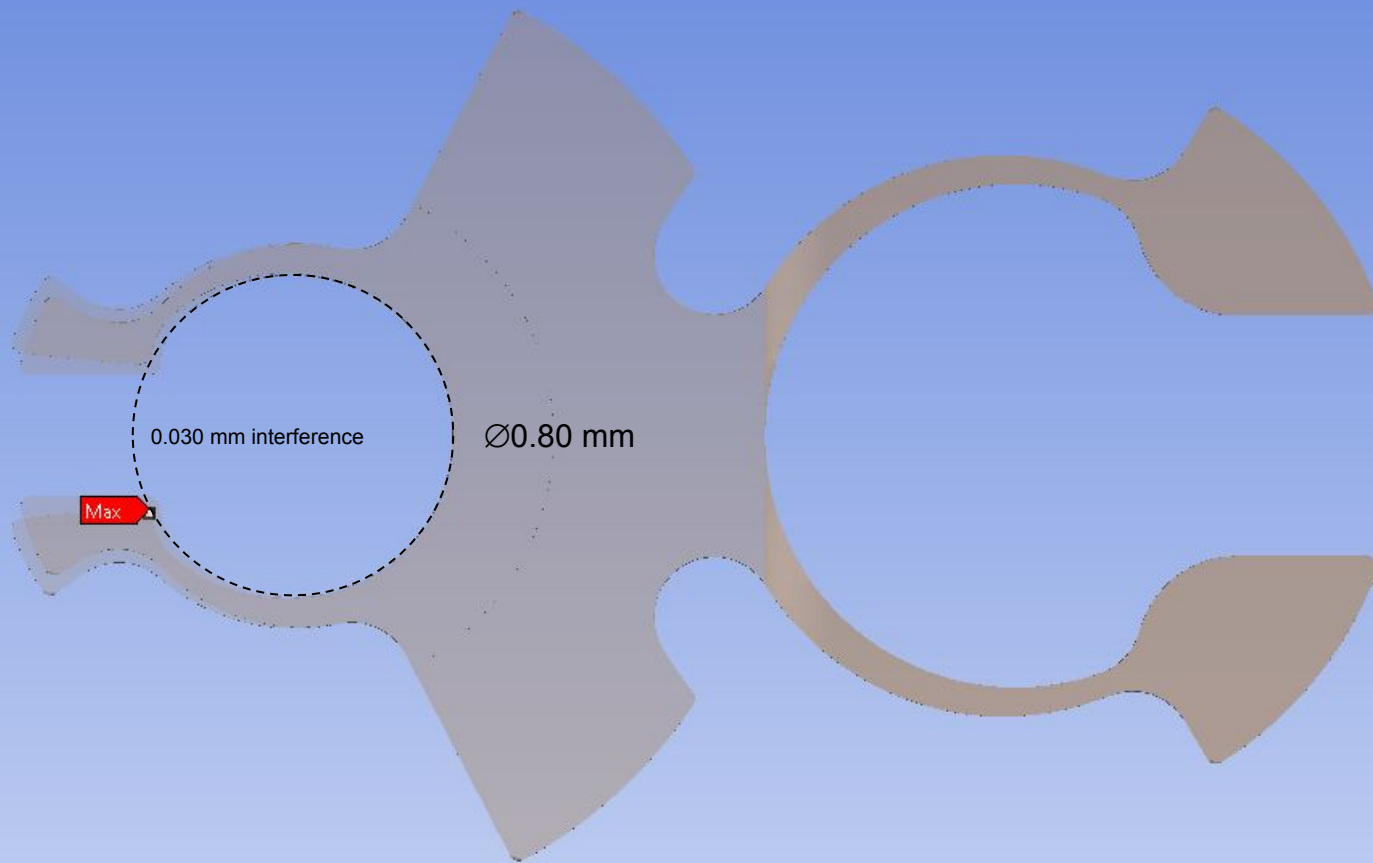
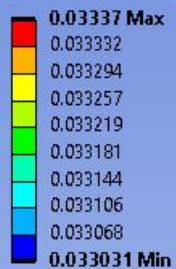
Total Deformation

Type: Total Deformation

Unit: mm

Time: 1 s

11/2/2023 1:03 PM



B: EDM Fiber Arm: Shaft Arms

Equivalent Stress

Type: Equivalent (von-Mises) Stress

Unit: MPa

Time: 1 s

11/2/2023 1:00 PM

851.76 Max

757.12

662.48

567.84

473.2

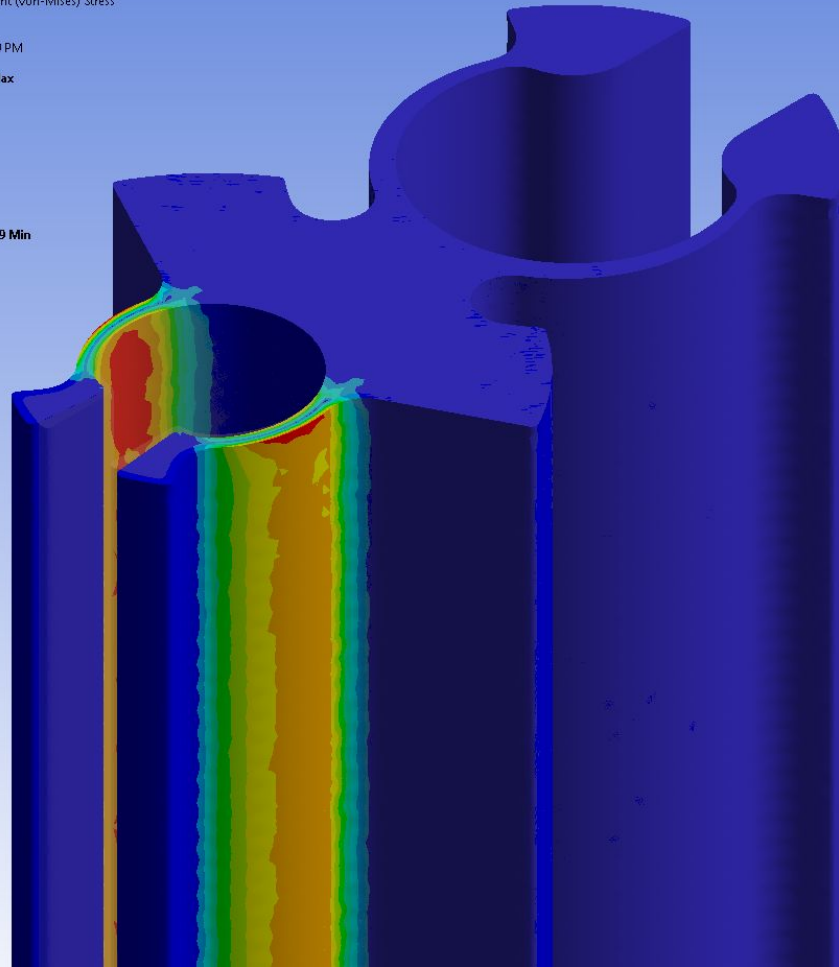
378.56

283.92

189.28

94.64

8.1866e-9 Min





Stiction: Glass Ferrule

~3.1 N

Insufficient forces?

Risks of low repeatability: Surface finishes, presence of lubricants, spring material properties, spring and ferrule geometries...

→ Increase forces, add features to constrain ferrule axially, and/or use adhesive



Stiction: Steel pin in shaft hole

~0.7 N

Not well constrained axially on shaft

→ Increase forces and/or use adhesive

Adhesive may be necessary for reliable axial constraint

A D-shaped shaft could be used for rotational constraint, but springs could be overcome, and adhesive may still be necessary

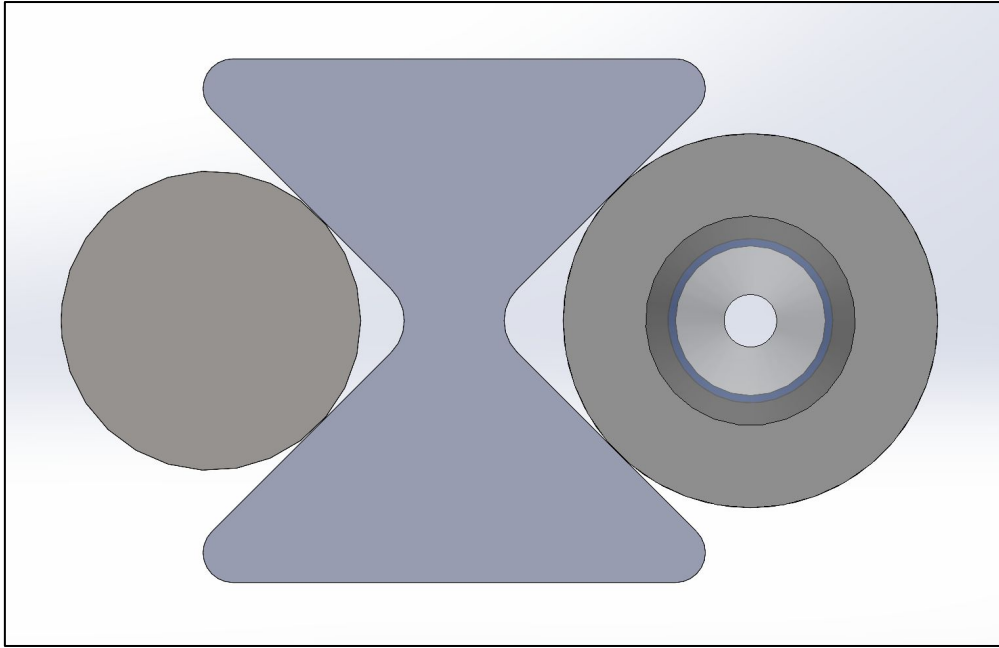
Stiction: Steel pin in ferrule hole

~2.7 N

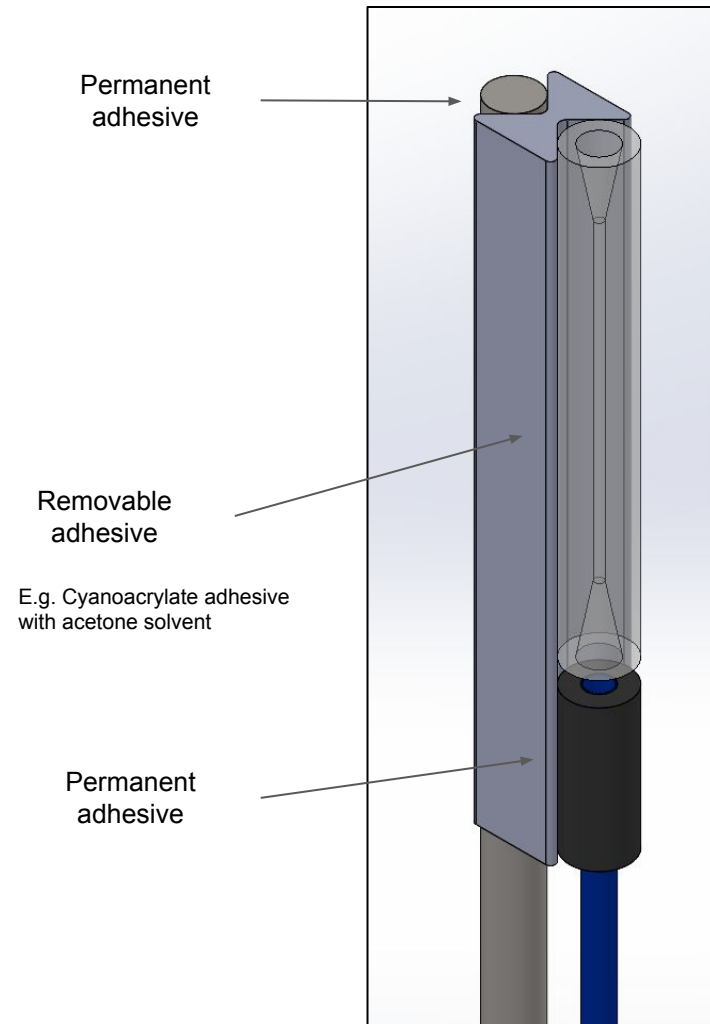


Other Conceptual Designs

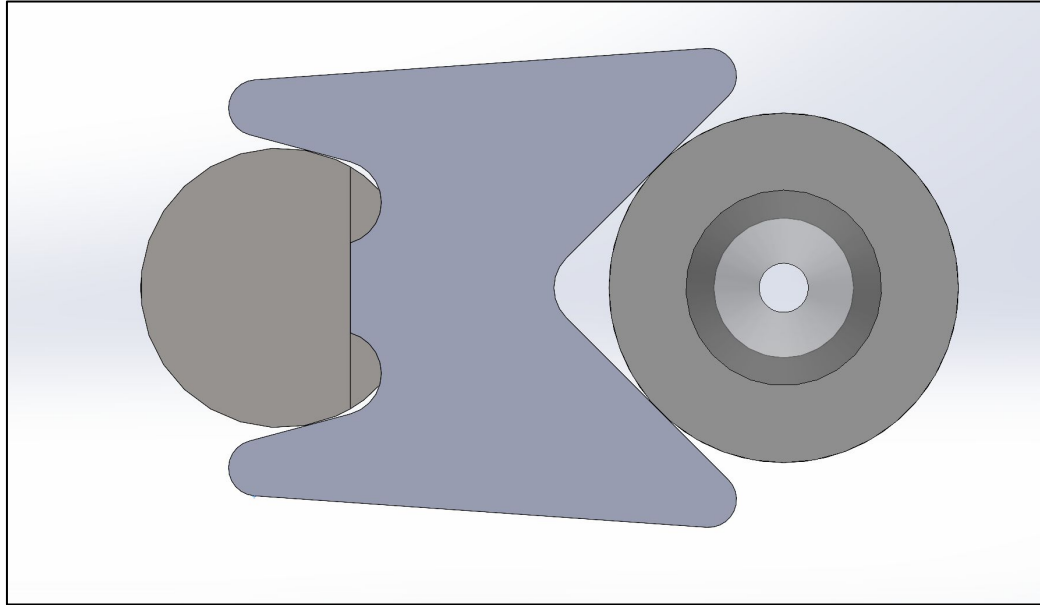
V-Block with Adhesives



Alignment with stop at the aft end of the beta shaft is set with tooling, then glued

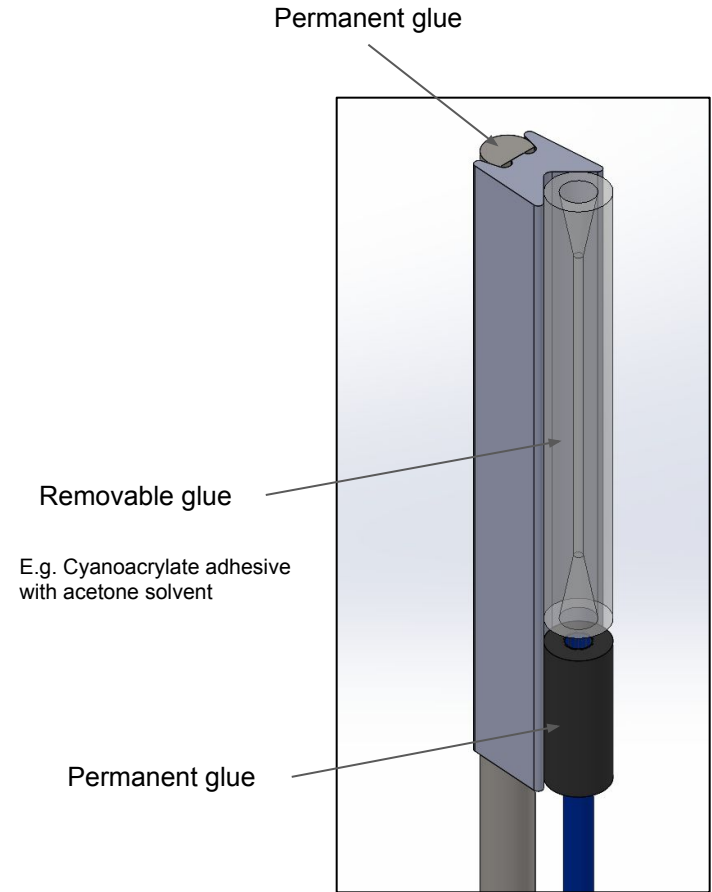


W-Block with Adhesives

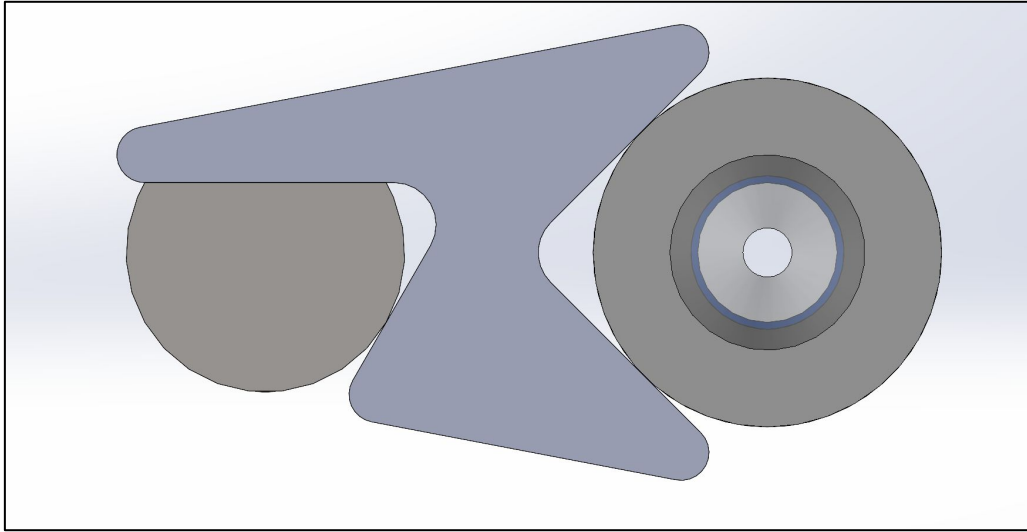


Metal (e.g. wire EDM)

Overconstrained...
Probably better to get rotational alignment with tooling

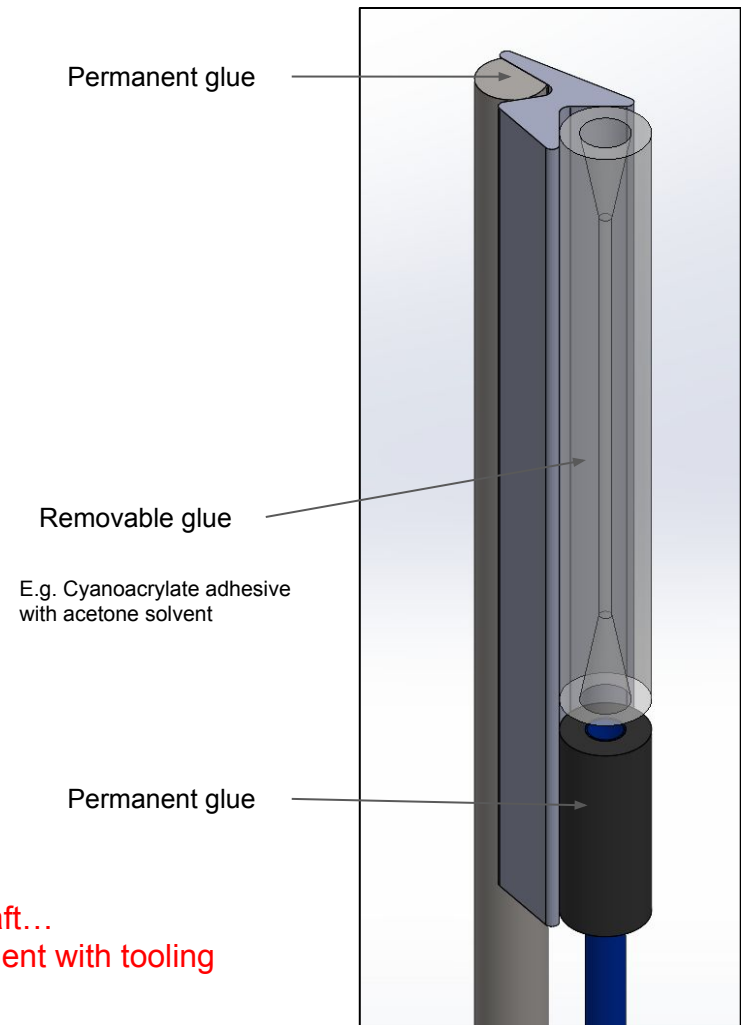


Z-Block with Adhesives

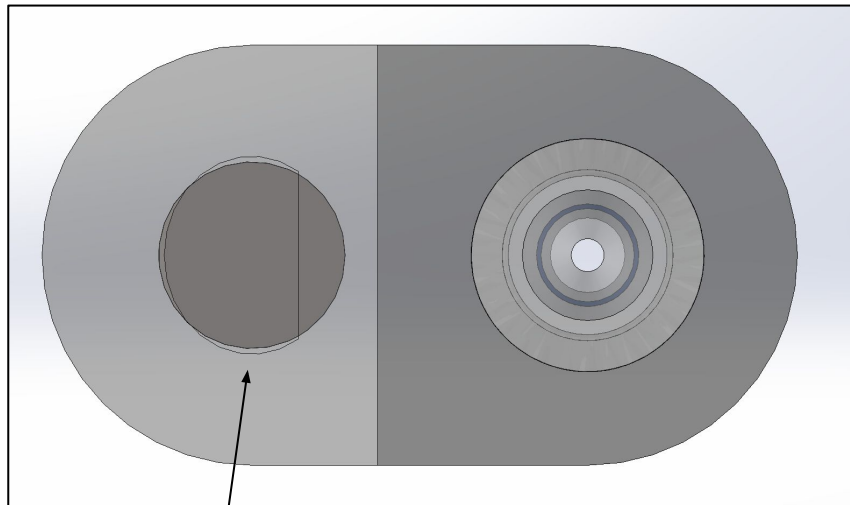


Metal (e.g. wire EDM)

Relies on accuracy of D-groove on shaft...
Probably better to get rotational alignment with tooling

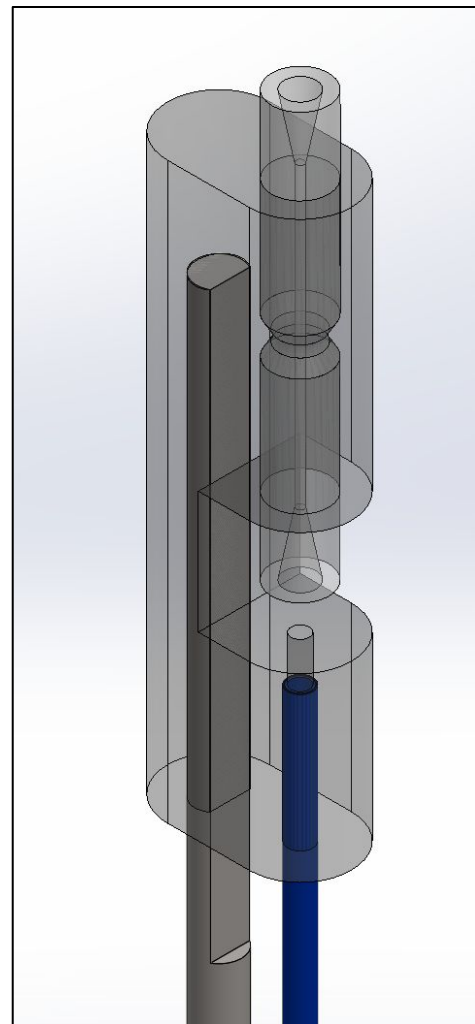
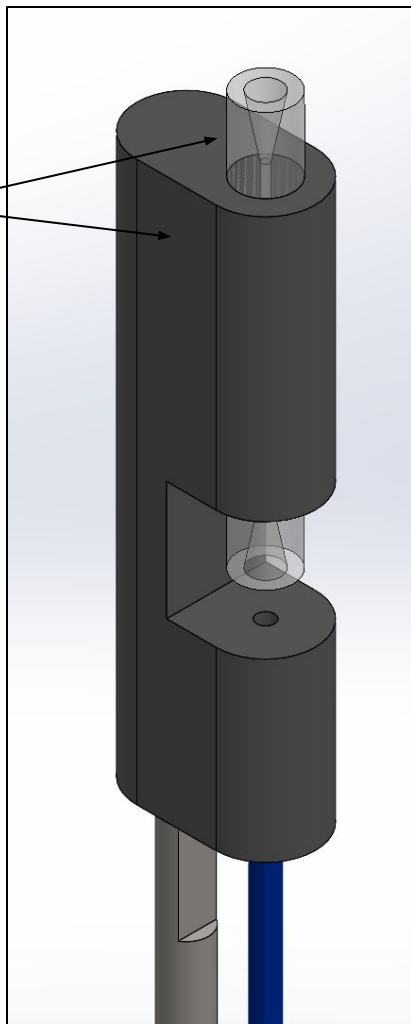


Press-Fit Plastic Arm Co-Molded with Ferrule



Press-fit with D-shaped ovalization

Co-molded



Collet-Lock with Adhesive

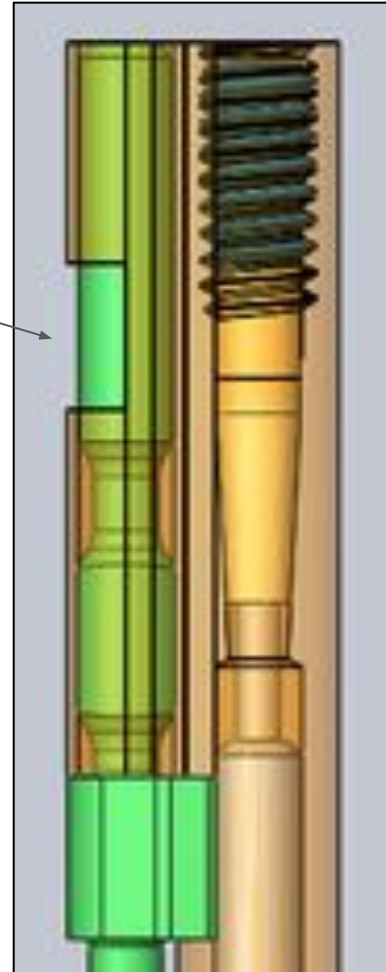
Metal (e.g. CNC machined or cast/CNC-machined)

Dab of optionally removable
adhesive

E.g.

Steel / Cyanoacrylate / Acetone

Complicated...
Lots of parts...
Have to apply torque to shaft...
Collet gets stuck...



Collet-Lock with Adhesive Plastic (e.g. injection molded)

Liquid Crystal Polymer

- Moderate bonding with cyanoacrylate
- Resistant to acetone

ABS/Cyanoacrylate/Acetone wouldn't work because ABS is affected by acetone...

PP/Cyanoacrylate/Acetone wouldn't work because PP doesn't bond well with cyanoacrylate...

Seems like it might be too much trouble to clean and reuse the plastic part. Maybe just throw away the plastic part of the fiber arm and reuse the collet and set screw? Or throw away entire assembly?

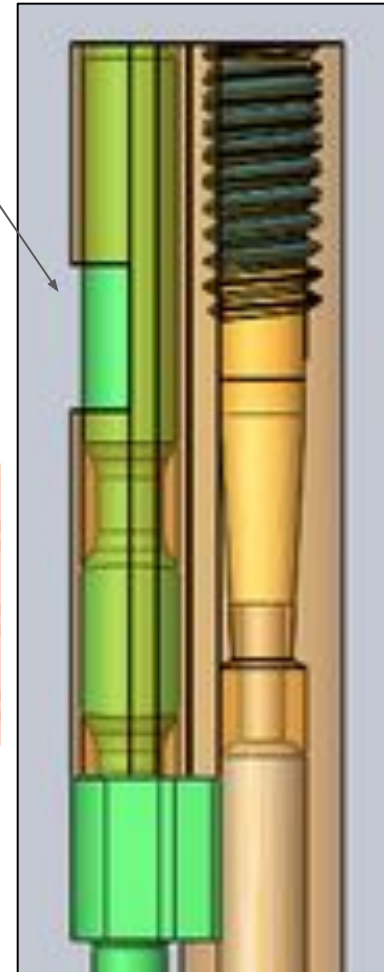
Table 4.2.1 - Chemical Resistance			
Rating: + Resistant - less than 2% change in weight and dimension, less than 1% change in mechanical properties. o Limited resistance - Not resistant			
Medium	Conditions (time/temperature)	Victrex® LCP grade	Rating
Acetic acid	(100%) 30 days/118°C 20 days/23°C	A950	+
		AE25	+
Acetone	180 days/50°C	A950	+
		A130	+
		AE25	+
Acetonitrile	120 days/23°C	AE25	+
Brake Fluid:			
Castrol® TLX 98BC	30 days/121°C	A130	o
		A950	+
		B950	+
MOR® brand DOT 3	90 days/121°C 90 days/121°C	A130	-
		A130	o
Chlorine gas	180 days/23°C	A950	+
		A130	+
		AE25	+
Chlorine water (saturated solution)	180 days/23°C	A950	+
		A130	+
		AE25	+
Chromic acid (50%)	90 days/50°C 180 days/50°C	AE25	o
		A950	o
		A130	o
		AE25	o
Chromic acid (70%)	30 days/70°C 60 days/70°C 180 days/70°C 30 days/88°C	A950	+
		A130	+
		A950	+
		A130	o
		AE25	o
Dimethyl formamide (DMF)	180 days/66°C	A950	+
		A130	+
		AE25	+
Diphenylamine	180 days/66°C	A950	+
		A130	+
		AE25	+
Diphenyl carbonate	10 days/250°C	A950	-
Ethanol	30 days/52°C	A950	+
Ethyl acetate 1	80 days/77°C	A950	+
		A130	+
		AE25	+
Ethylene diamine	30 days/100°C 180 days/23°C	A950	-
		A950	+
		A130	o
		AE25	+

Dab of optionally removable adhesive

- Dissolvable
- Thermoplastic?
 - Wax?
 - Stiff hot glue?
- Mechanical separation?

Table 9.2.2 - Lap Shear Strength				
a) Testing Performed at 22°C				
Adhesive Type	Range of Values, N/mm ²		Average Values, N/mm ²	
	As Molded	Surface Treated ^a	As Molded	Surface Treated ^a
2 Part Epoxy	3.1 - 6.0	5.5 - 14.5	4.8	9.0
1 Part Epoxy	4.1 - 9.0	5.5 - 9.7	6.2	10.7
Cyanoacrylate	2.1 - 4.8	3.4 - 6.8	3.4	5.5
2 Part Acrylic	1.7 - 5.5	3.4 - 5.5	3.1	4.8
b) Testing Performed at 100°C				
Adhesive Type	Range of Values, N/mm ²		Average Values, N/mm ²	
	As Molded	Surface Treated ^a	As Molded	Surface Treated ^a
2 Part Epoxy	1.0 - 2.1	1.0 - 2.8	1.4	2.1
1 Part Epoxy	1.4 - 4.8	1.7 - 5.5	3.4	4.3
Cyanoacrylate	0.5	2.1 - 3.4	3.1	2.8
2 Part Acrylic	0.7 - 1.4	1.4 - 2.1	1.0	1.7
c) Testing Performed at 150°C				
Adhesive Type	Range of Values, N/mm ²		Average Values, N/mm ²	
	As Molded	Surface Treated ^a	As Molded	Surface Treated ^a
2 Part Epoxy	0.7 - 1.4	0.7 - 3.4	0.7	1.0
1 Part Epoxy	0.7 - 2.1	0.7 - 2.1	1.4	1.4
Cyanoacrylate	0.2 - 0.3	0.3 - 0.7	0.2	0.7
2 Part Acrylic	0.3	0.7	0.3	0.7

^aLight sanding or grit blasting and solvent wash



Complicated features for injection molding...

Set-Screws

Metal (e.g. CNC machined or cast/CNC-machined)



Hard to access set screws...
Slop in both joints...

Orbray Prototype (Jan. 2024)

Set-Screw to Shaft with Adhesive to Ferrule

Metal (e.g. CNC machined or cast/CNC-machined)

E.g.

Steel / Cyanoacrylate / Acetone

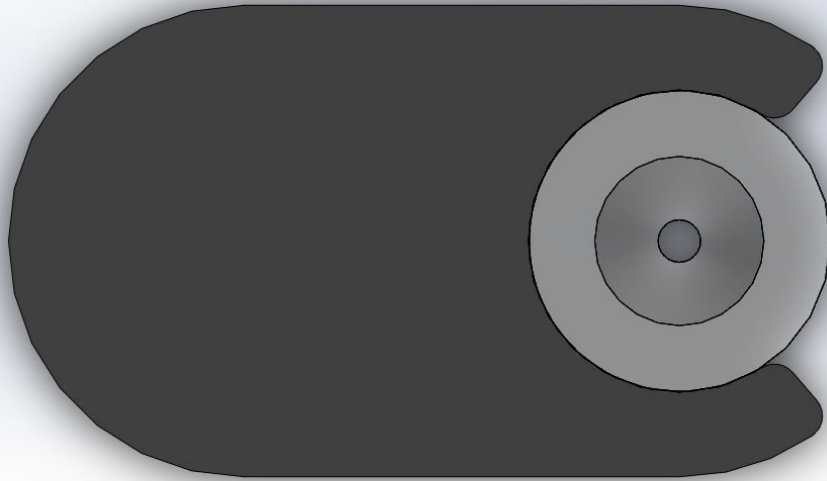
Slop in screw screw joint...
Difficult/expensive to clean/reuse?

Dab of adhesive



Press-Fit to Shaft with Clip to Ferrule

E.g. Injection-molded plastic



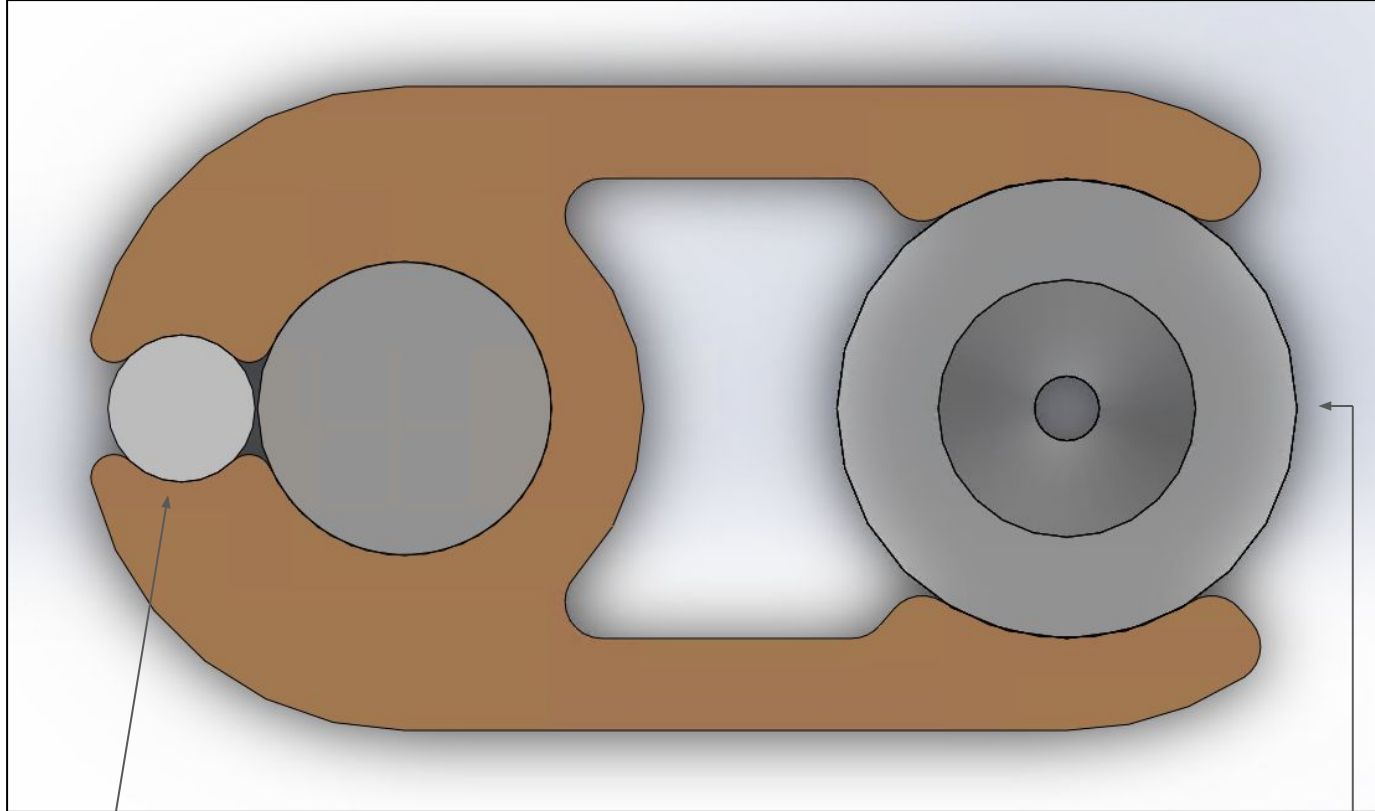
Hard to get secure/repeatable clipping forces?



Press-fit shaft (e.g.
with D-shaped
ovalization)

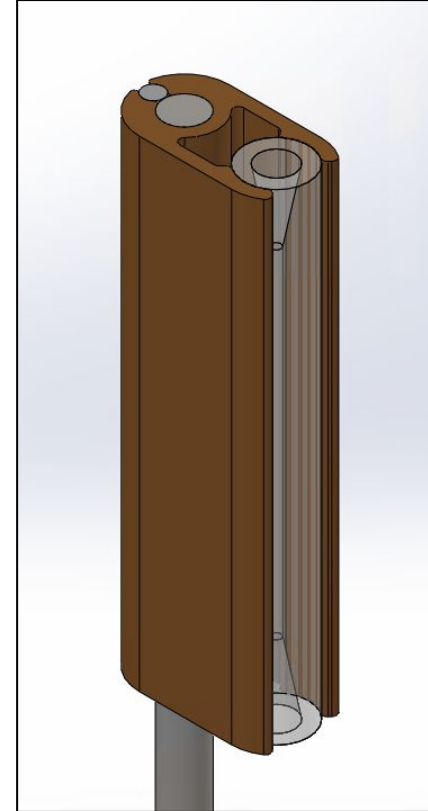
Braze to Shaft and Clip to Ferrule

Metal (e.g. wire EDM)



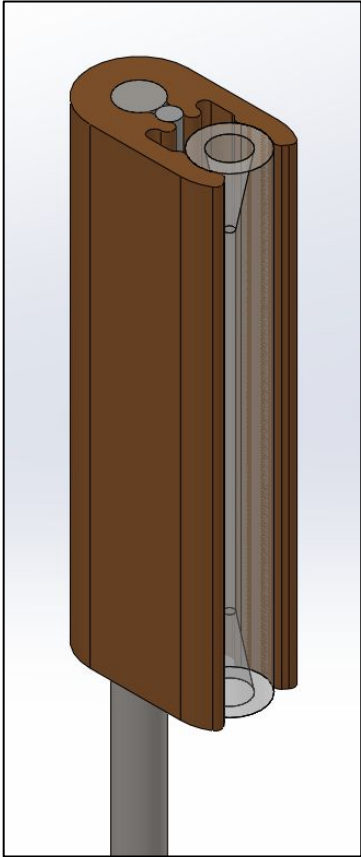
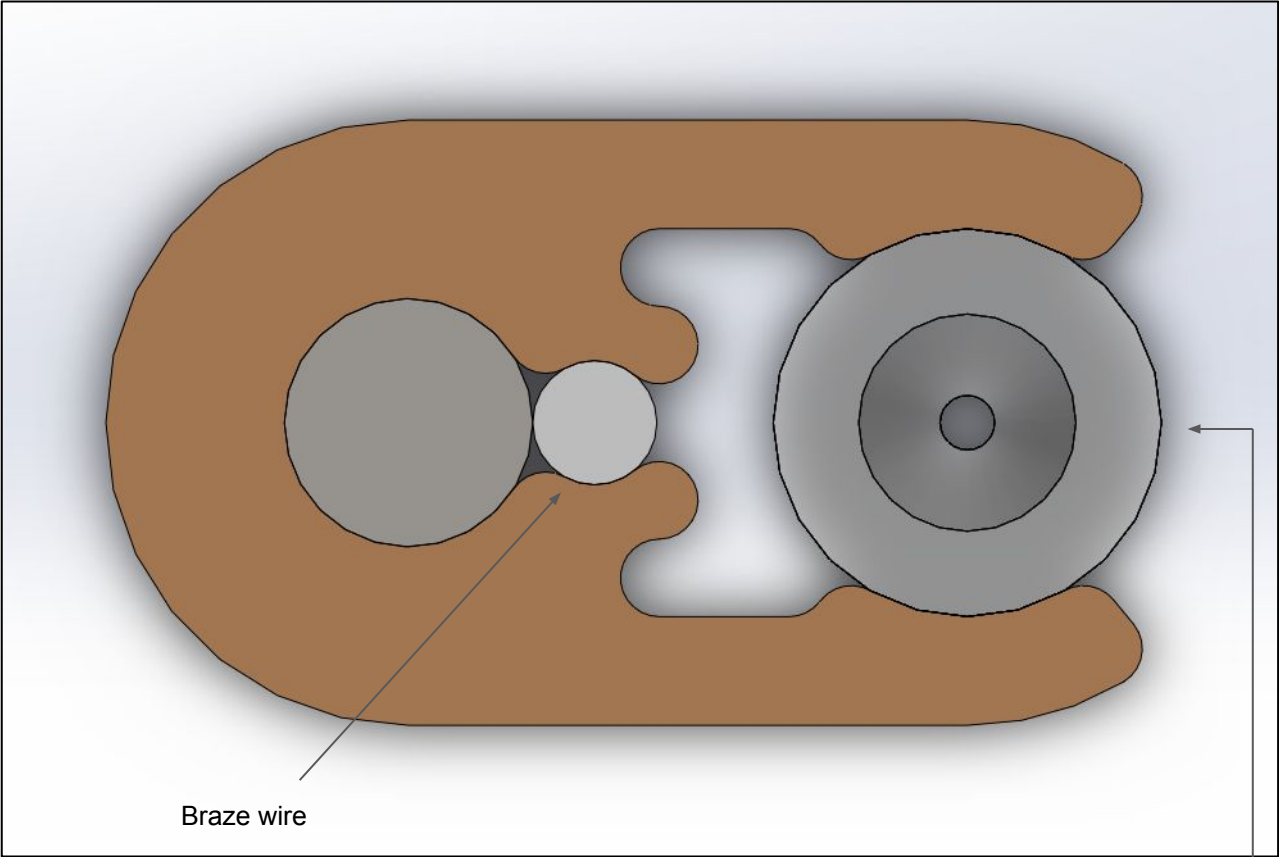
Braze wire

Optional: Dab of removable adhesive (e.g. cyanoacrylate)



Braze to Shaft and Clip to Ferrule

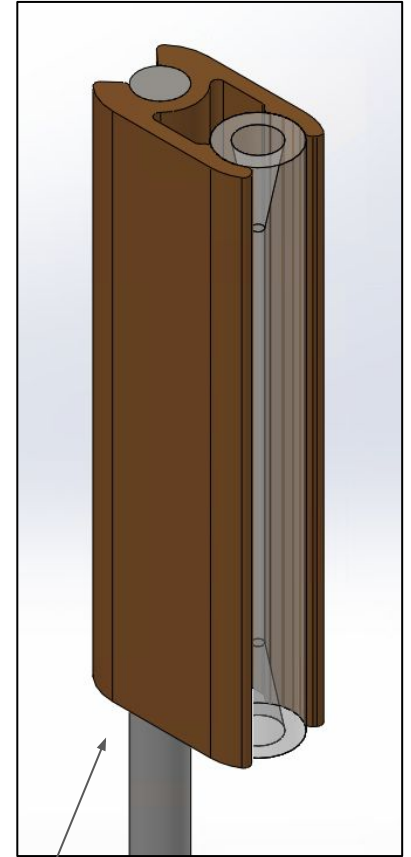
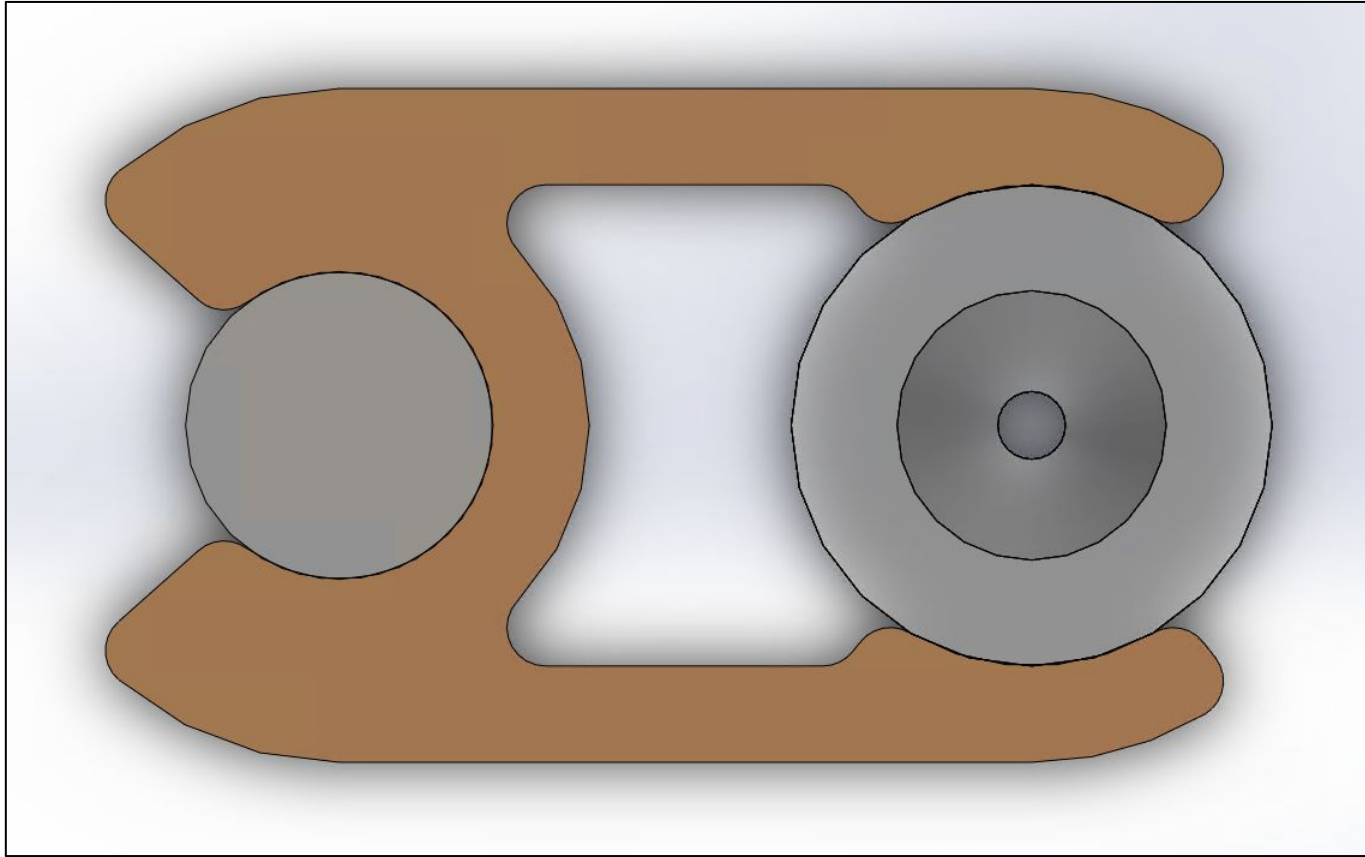
Metal (e.g. wire EDM)



Optional: Dab of removable adhesive (e.g. cyanoacrylate)

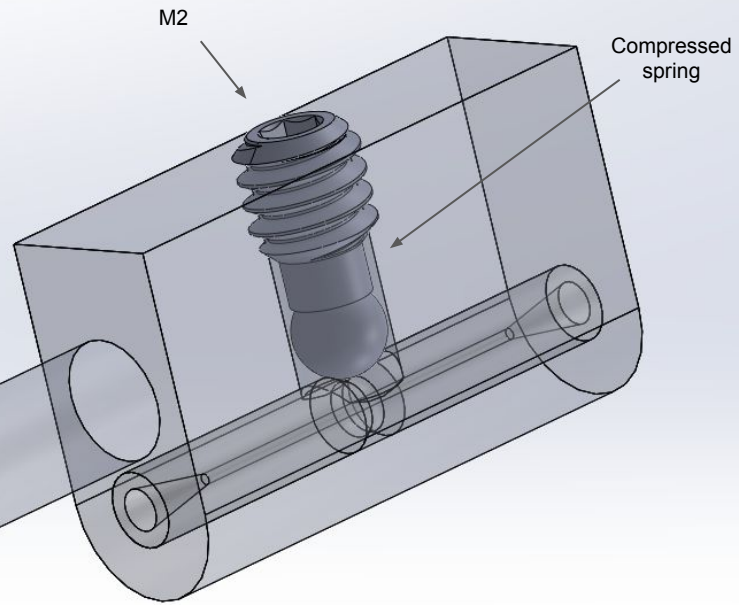
Snap to Shaft and Clip to Ferrule

Metal (e.g. wire EDM)

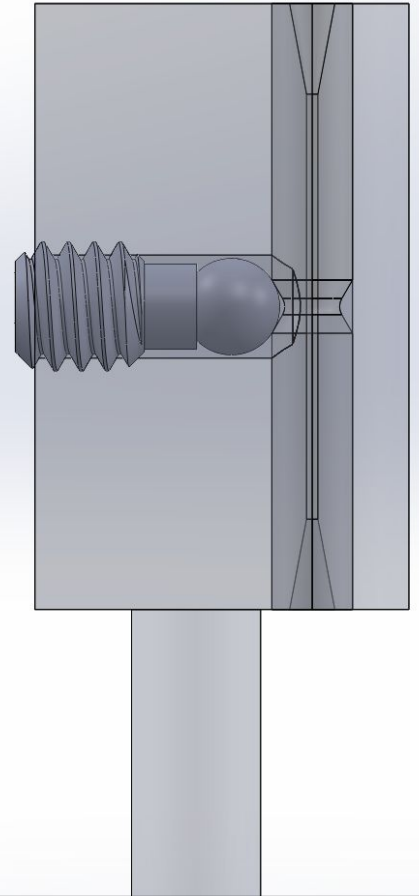


Snap-on shaft

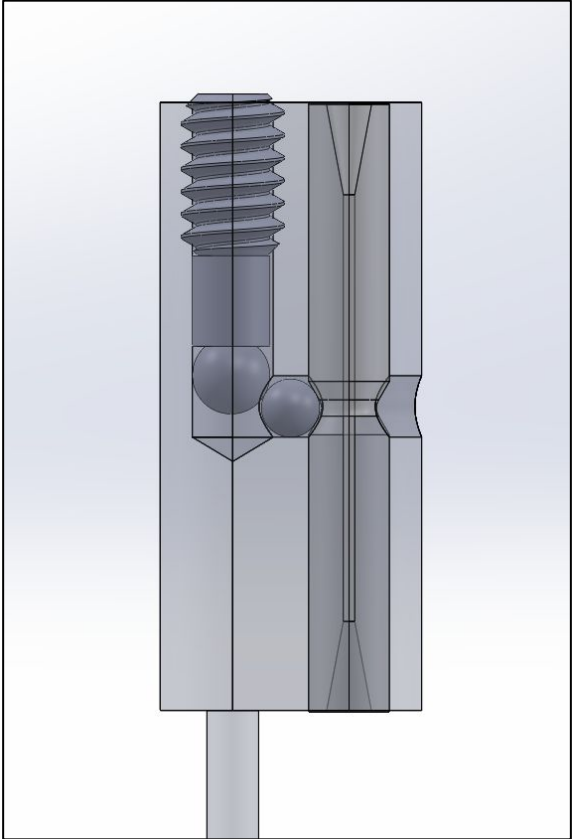
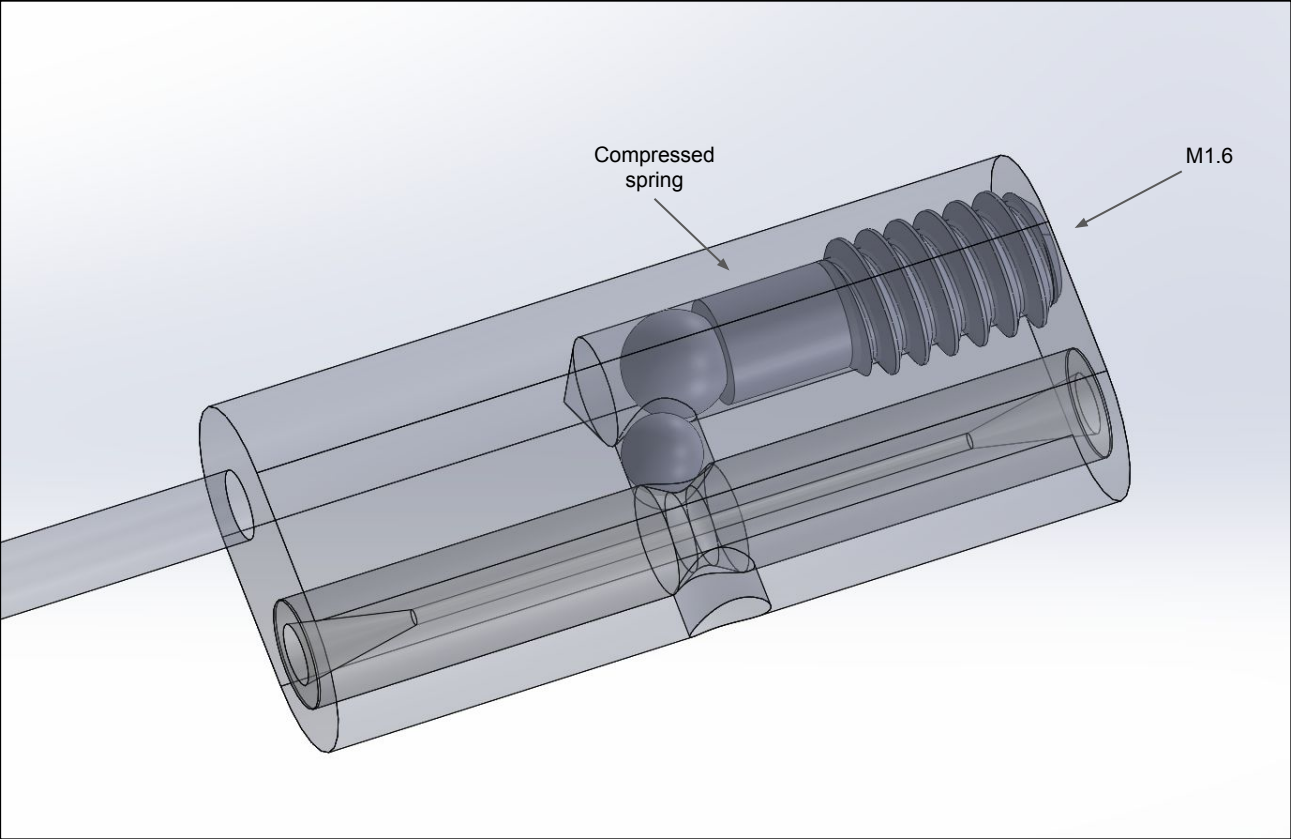
Ball Detent



Bulky...
Smaller hardware might be difficult to handle...

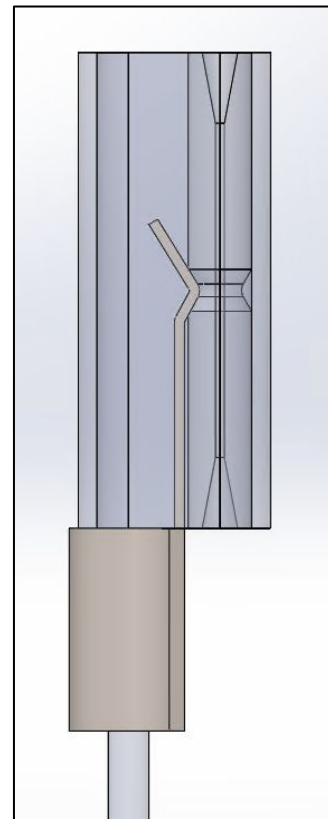
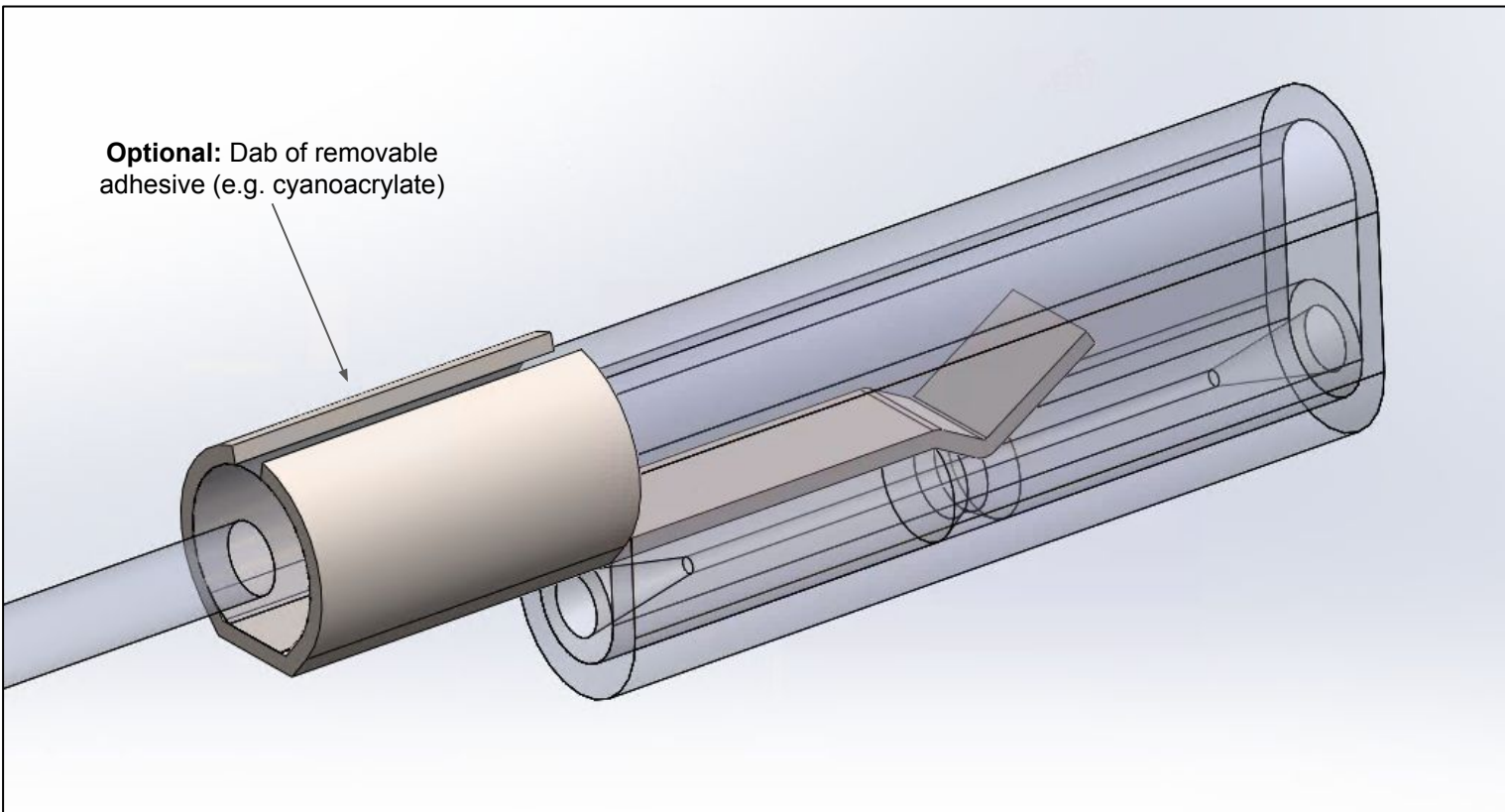


Ball Detent with End Entry

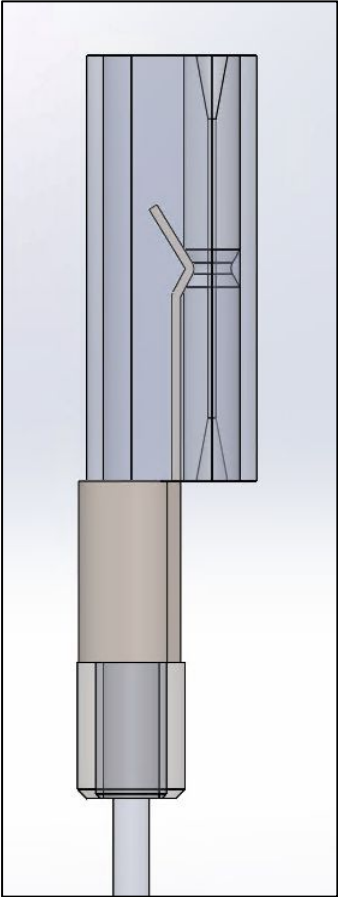
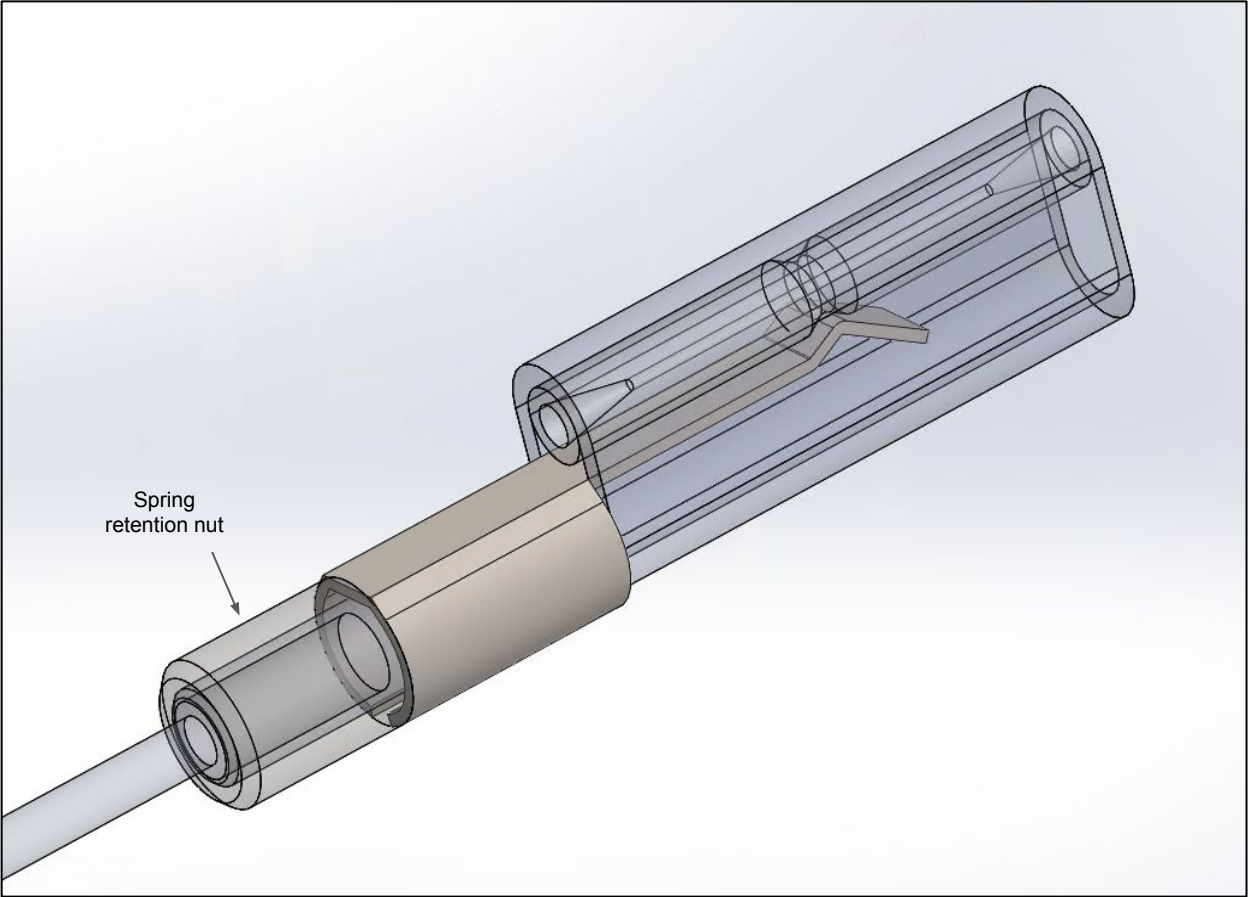


Strap Spring - Friction Fit

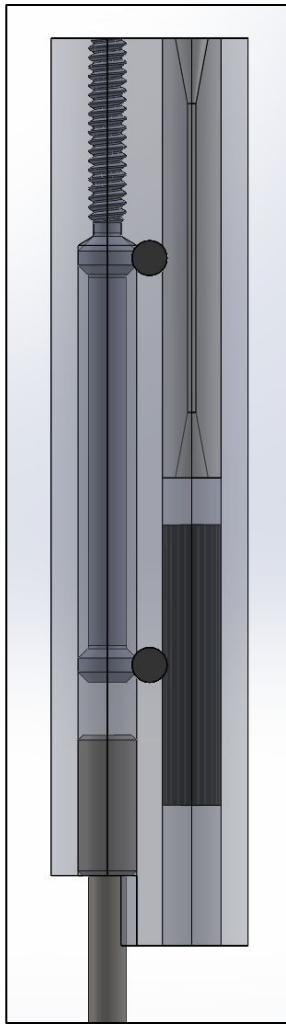
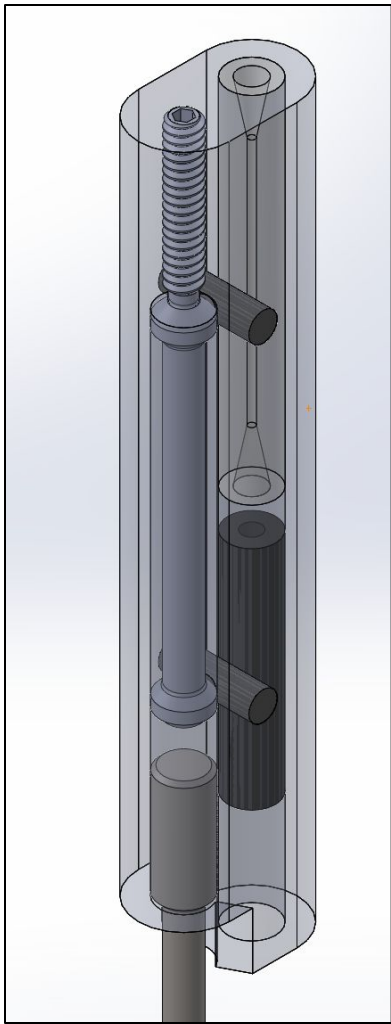
Optional: Dab of removable adhesive (e.g. cyanoacrylate)



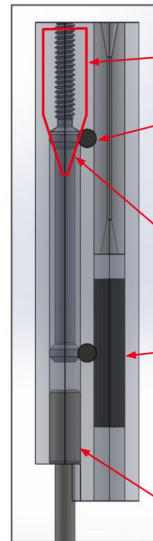
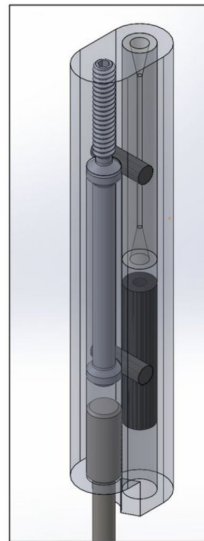
Strap Spring - Retention Nut





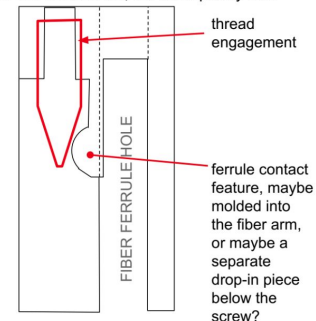


Pinch Fiber Arm 1.0



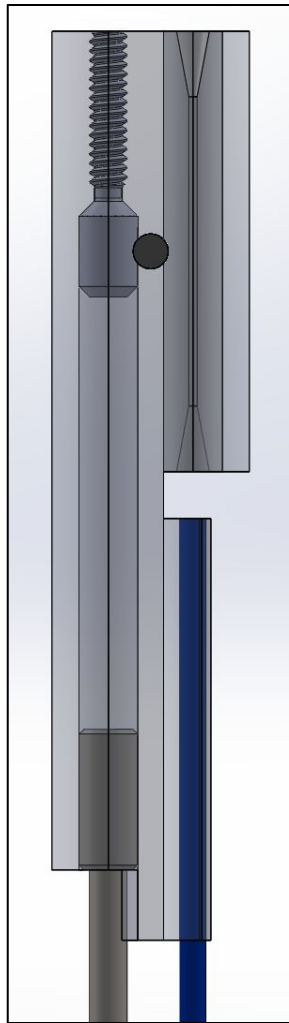
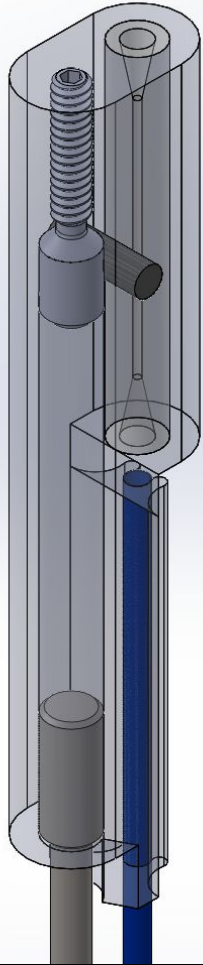
- 0-80 or M1.6 if possible
- try to make a bigger diam rod, or perhaps a larger non-cylindrical thingy that is injection-molded and can be dropped in the screw hole before the screw?
- cone point compresses rod (no wheel action)
- Machine a snap-in feature for the fiber sleeve? I don't think you need a ferrule in this design. Just clip the plastic tube into an appropriate c-shaped metal feature. See example in prior fiber arm model in the pdm repo.
- I think having an enlarged shaft end like this will either cost extra parts or make design of assembly below be more difficult

Another question - if this part were a single piece of injection-molded plastic, could it have a sort of built-in flap that does the job of the "rod"? I'm not sure if this works in 3D, but conceptually like:



- The cylindrical faces that touch the rubber rods seem too short to me, I'd want them maybe 4x longer to ensure the rod actually gets compressed without needing fancy tolerances.
- Having the fiber ferrule and sleeve ferrule locked in by the same screw position seems to constrain assembly/repair scenarios. Could the cylinder that presses the fiber ferrule rod be offset for the sleeve ferrule, such that it disengages at a different screw axial position?
- I worry that having orthogonal faces (the cylinders) sliding along the rod in principle isn't good, because then they can rotate the rod, and the rod drives the ferrule up and down like a drive wheel.
- Rod material needs to be selected to minimize risk of creep. (Notching the ferrule for positive engagement would give some fallback safety.)

Pinch Fiber Arm 2.0



Better to make the threads on the set screw bigger than the pinching feature

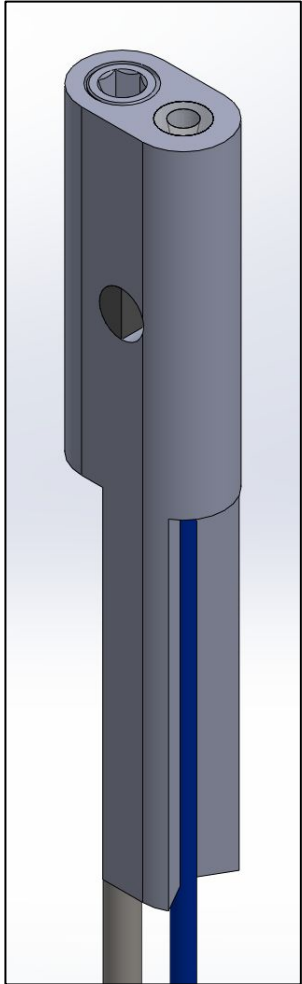
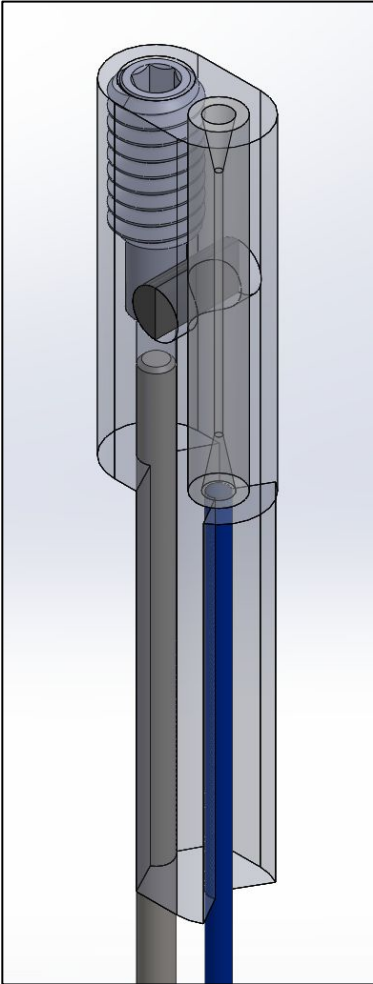
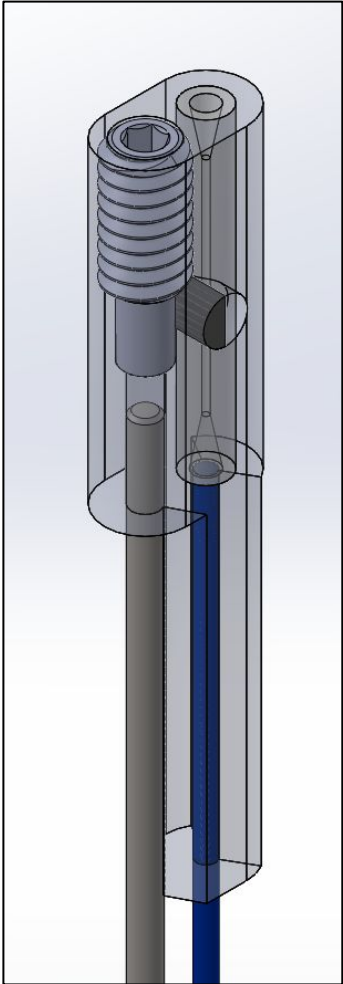
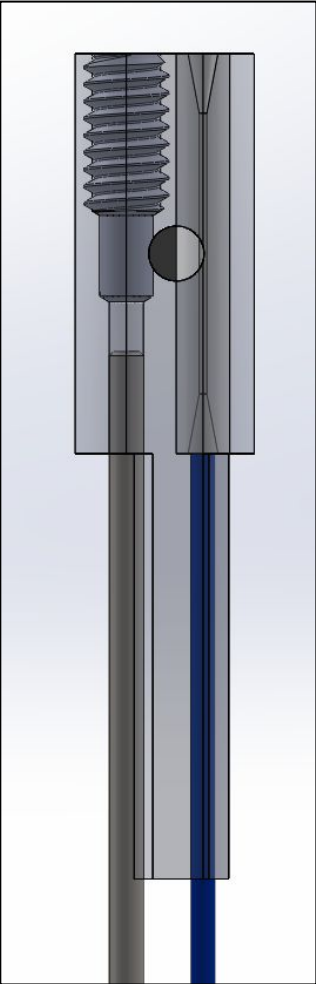
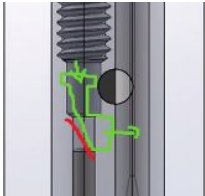
That way you can maintain a small hole for the beta shaft and don't need the expanded diameter on the beta shaft

Pinch Fiber Arm 3.0

Probably want to avoid CNC machined metal-metal press-fits at this scale

Ways to capture plug?
How to fabricate plug?
What material?

What about 3D printing or injection molding something and dropping it into the hole, squishing it with set screw?



One or both adhesives could be removable

Might be easiest to remove fiber arm from shaft during repairs: Easier to clean and reuse shaft than to clean and reuse fiber arm

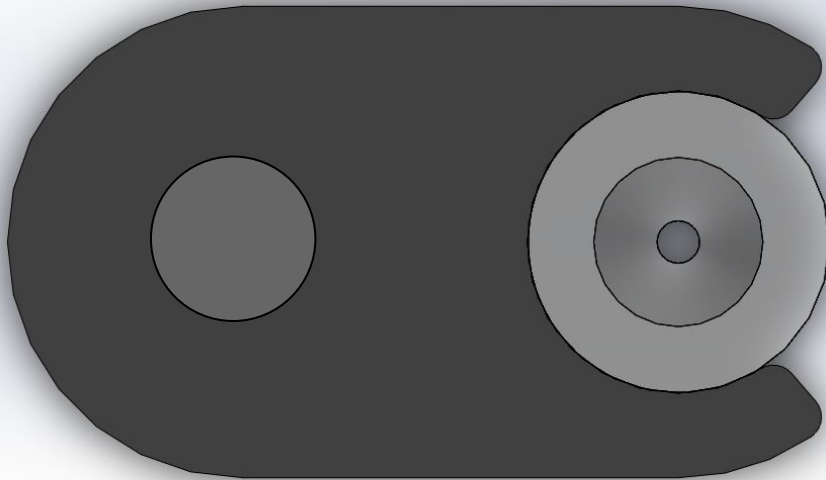
Dab of adhesive

Dab of adhesive

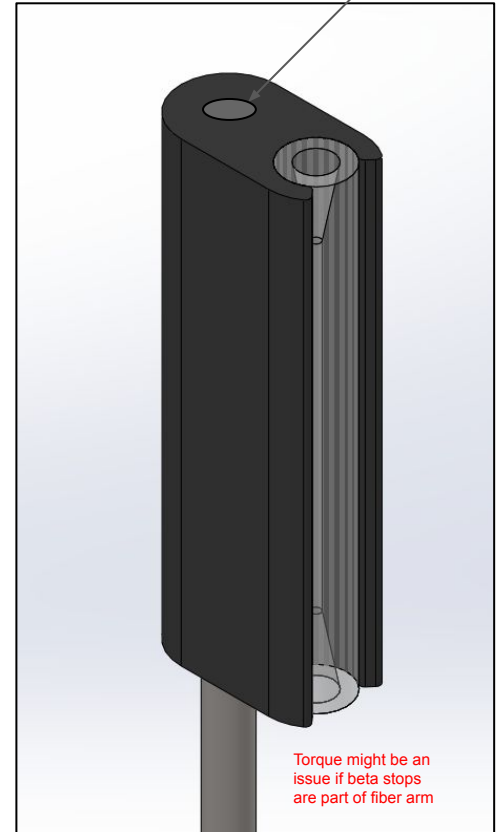
Adhesive Joints

Plastic (e.g. injection-molded)

Need biasing for alignment in holes...

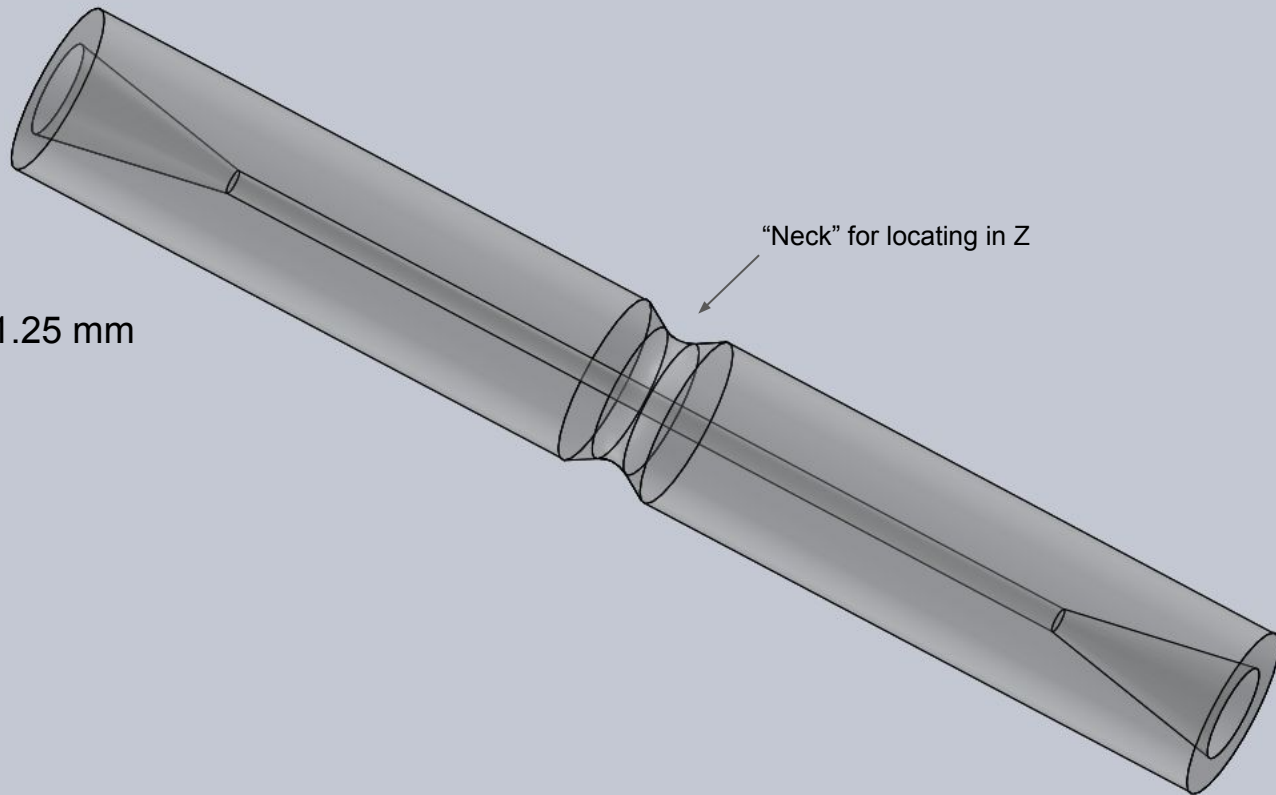


E.g. LCP plastic with cyanoacrylate adhesive and acetone solvent



“Necked” Glass Ferrule:

OD: 1.25 mm

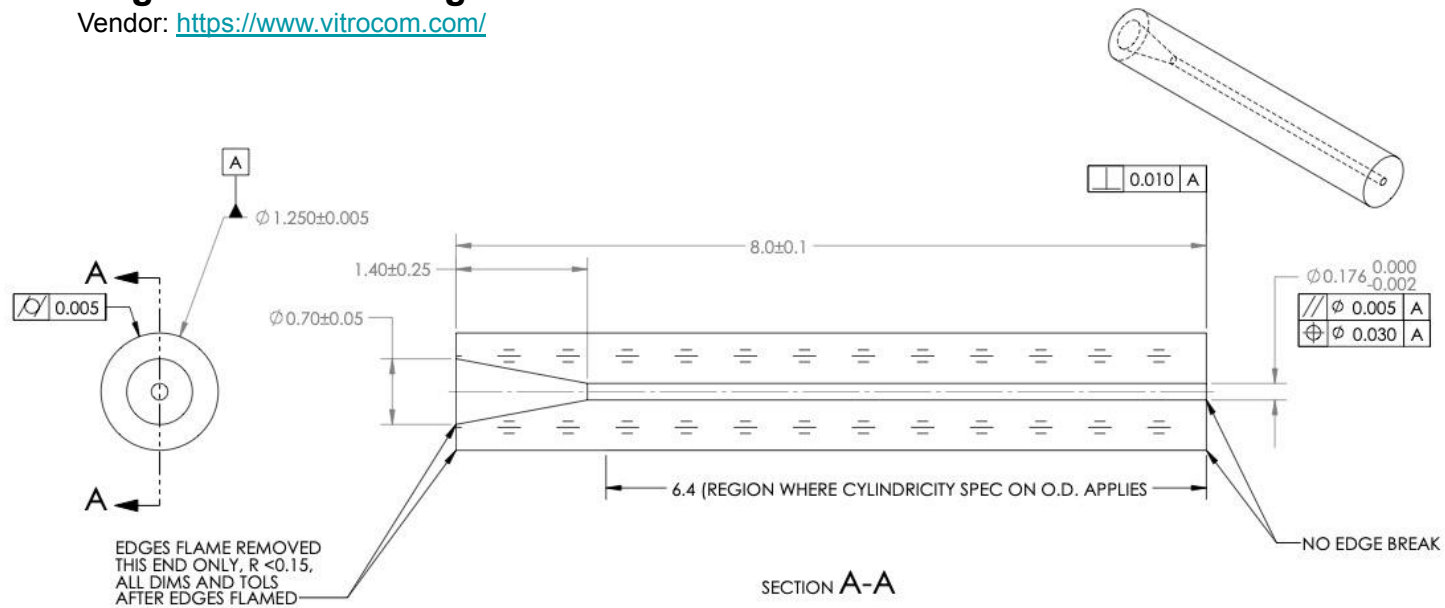


“Neck” for locating in Z

REV.	DESCRIPTION	REVISED BY	CHECKED	SYS. E.	CM	DATE
v1	INITIAL RELEASE	RWB	JE	RWB	RWB	5/21/15

Original Ferrule Design:

Vendor: <https://www.vitrocom.com/>



[Edges in production units were lapped, not flame polished]

NOTES: UNLESS OTHERWISE SPECIFIED

- MATERIAL: DRAWN BOROSILICATE GLASS, POLISHED FINISH.
- BAG AND TAG IN GROUPS OF 100.

UNLESS OTHERWISE SPECIFIED:

DIMENSIONS IN MM
 TOLERANCES:
 ANGULAR $\pm 0.5^\circ$
 X ± 0.5
 XX ± 0.1
 XXX ± 0.05
 XXXX ± 0.010

GD&T PER: ASME Y14.5M-2009

MATERIAL: SEE NOTES

MACHINING FINISH: SEE NOTES

EDGE BREAK: SEE GRAPHICS

PROJECTION: THIRD ANGLE

DO NOT SCALE DRAWING

Ernest Orlando Lawrence
 Berkeley National Laboratory

PROJECT
 DESI

TITLE
 TAPERED GLASS FERRULE

SOLIDWORKS MODEL REVISION AT TIME OF RELEASE: 0004

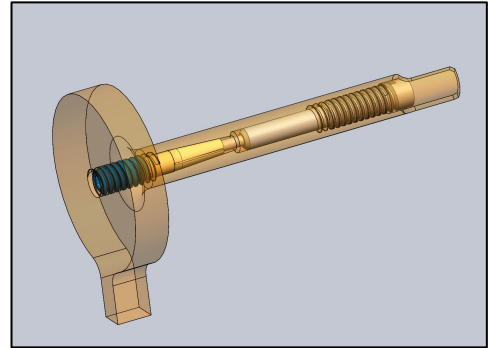
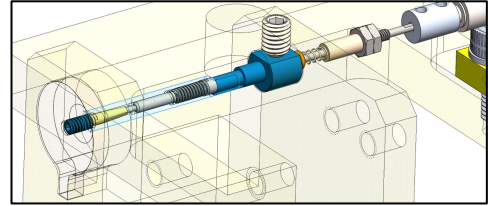
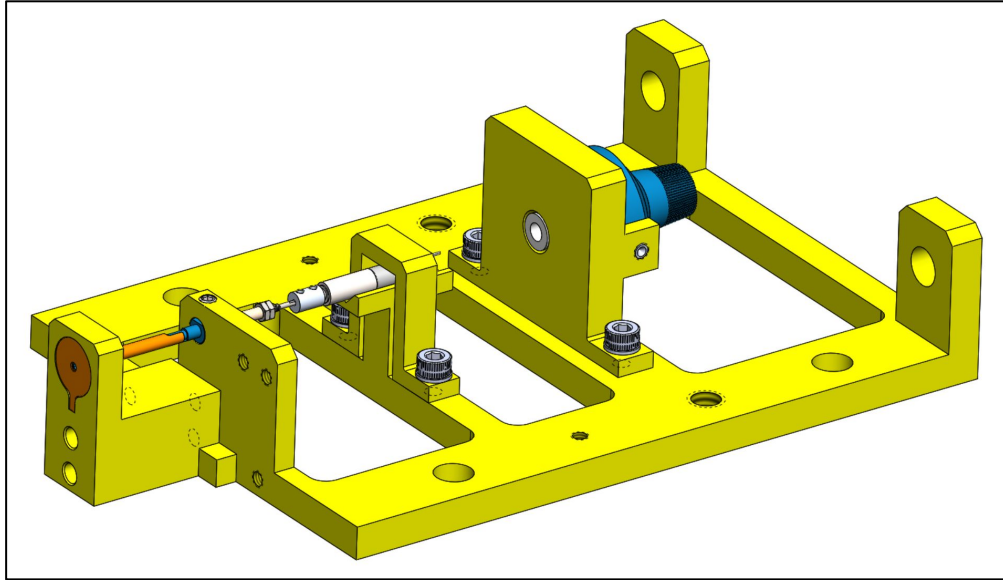
SIZE DWG. NO.
B DESI-1136

REV
v1

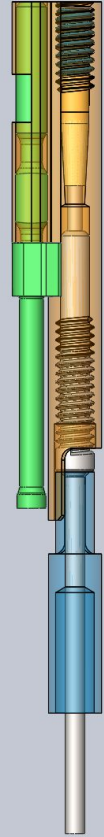
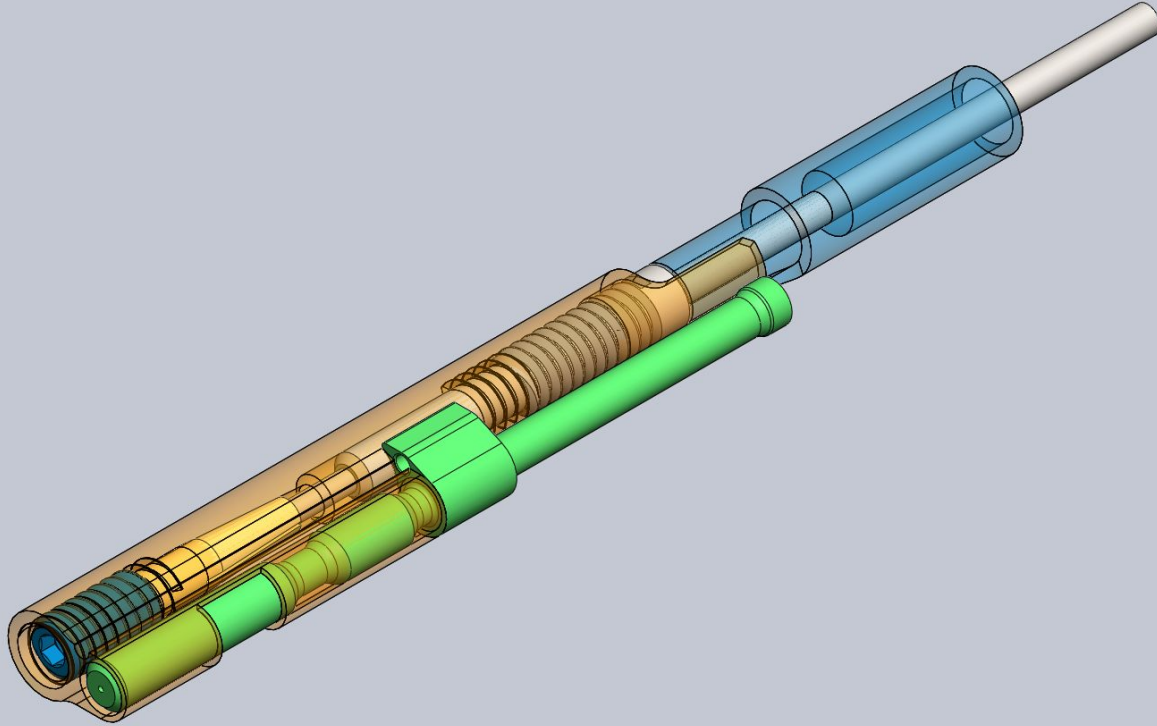
SCALE: 25:1 WT: Kg

SHEET 1 OF 1

“Auto-Focus” Fiber Arm Prototypes

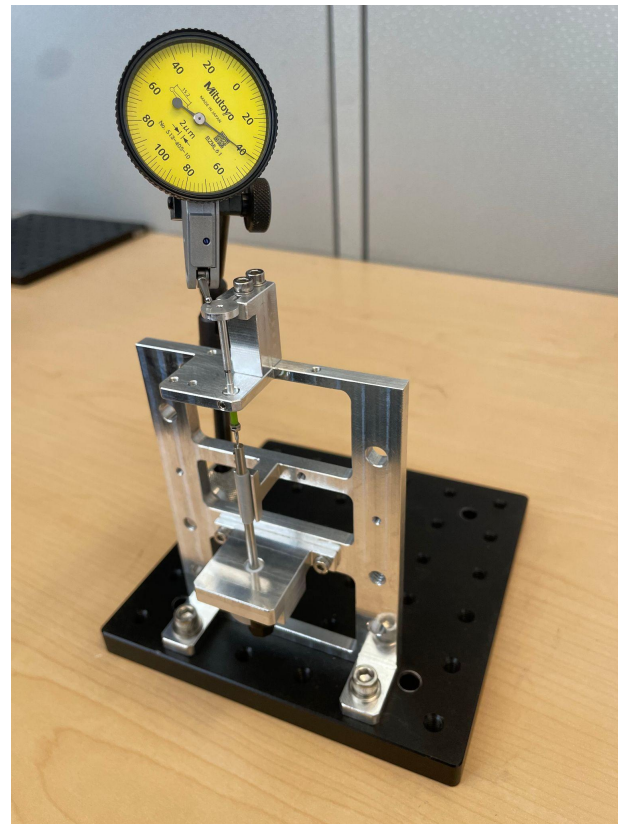
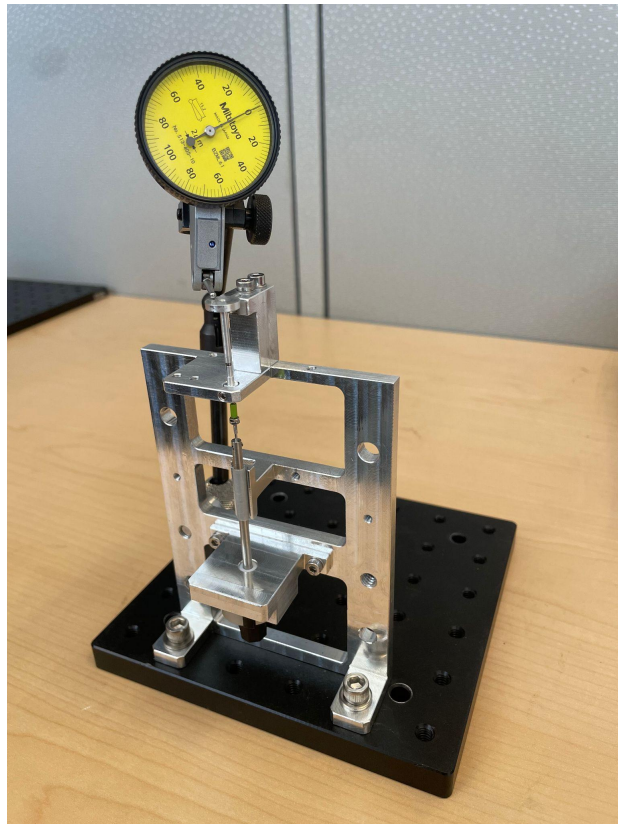
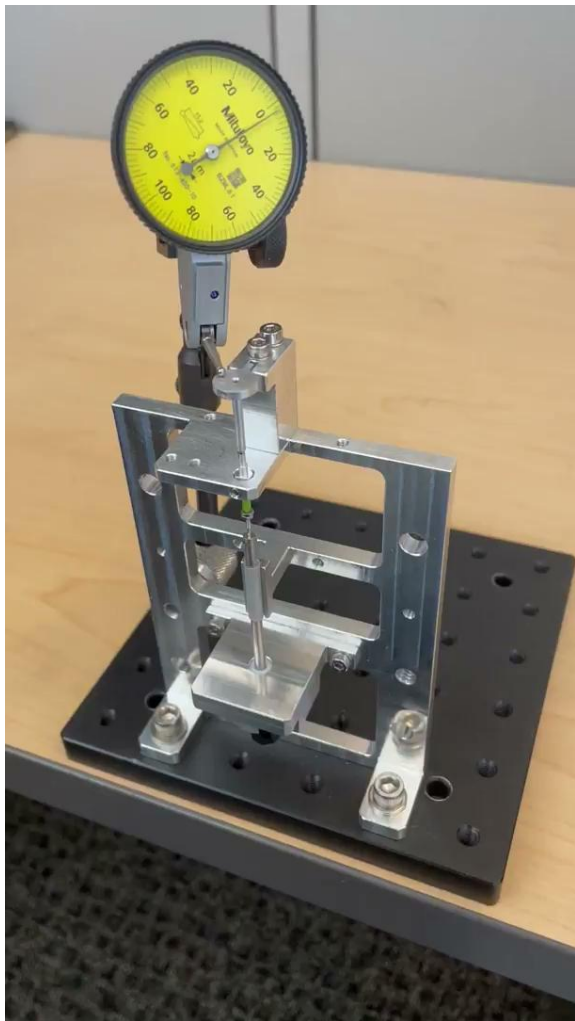


Test assembly is based on MM Fiber Arm Assy concept



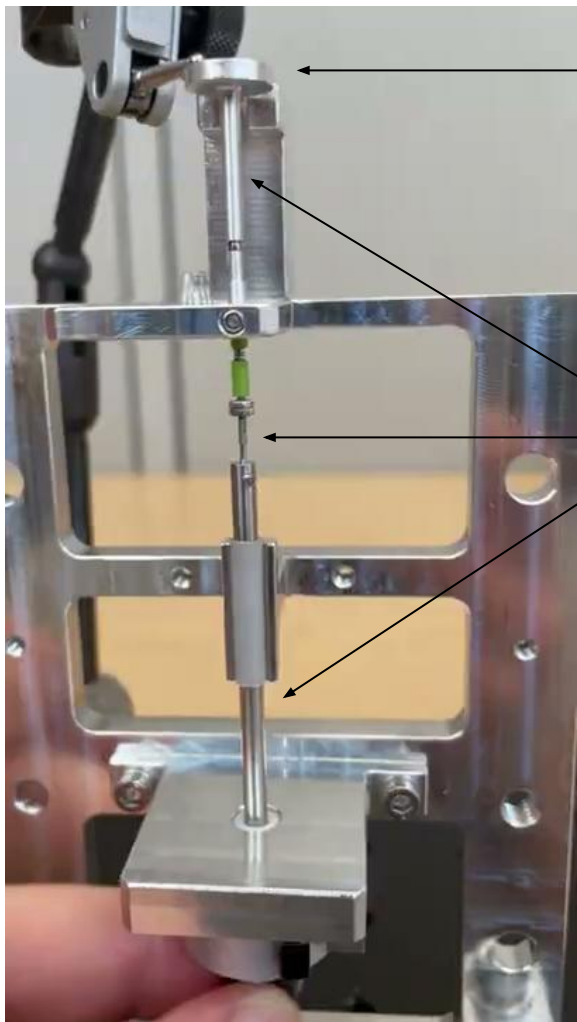
Trial 1:

Manual Knob Actuation with Unglued Fiber Arm



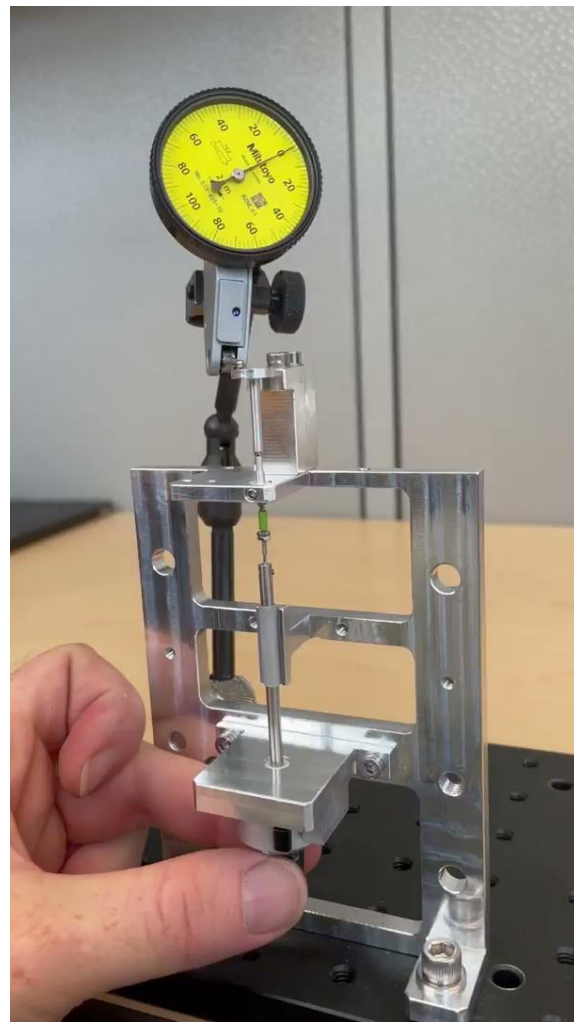
Top Surface of Fiber Arm:

~40 μm max displacement



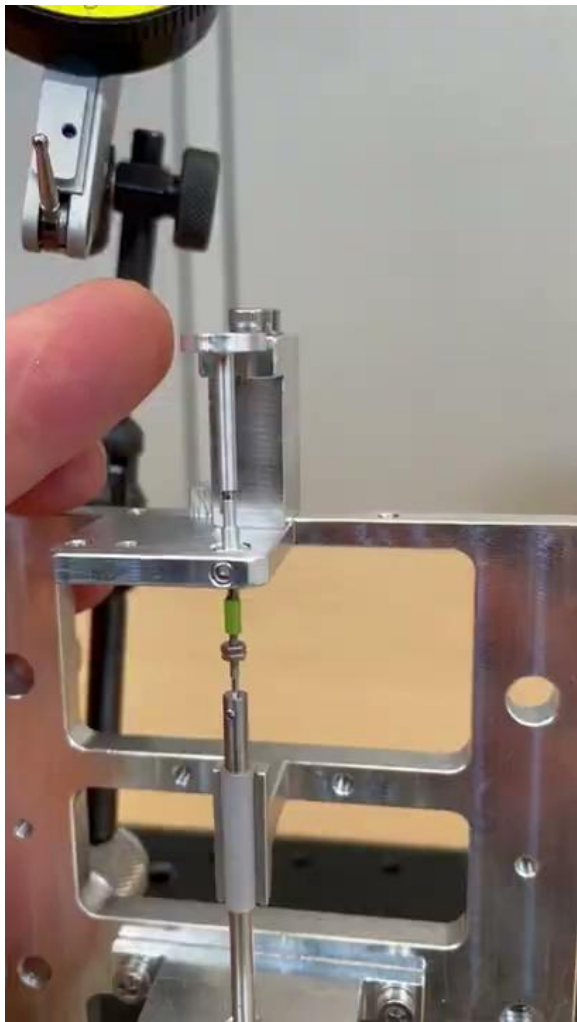
Precession

Axial misalignment



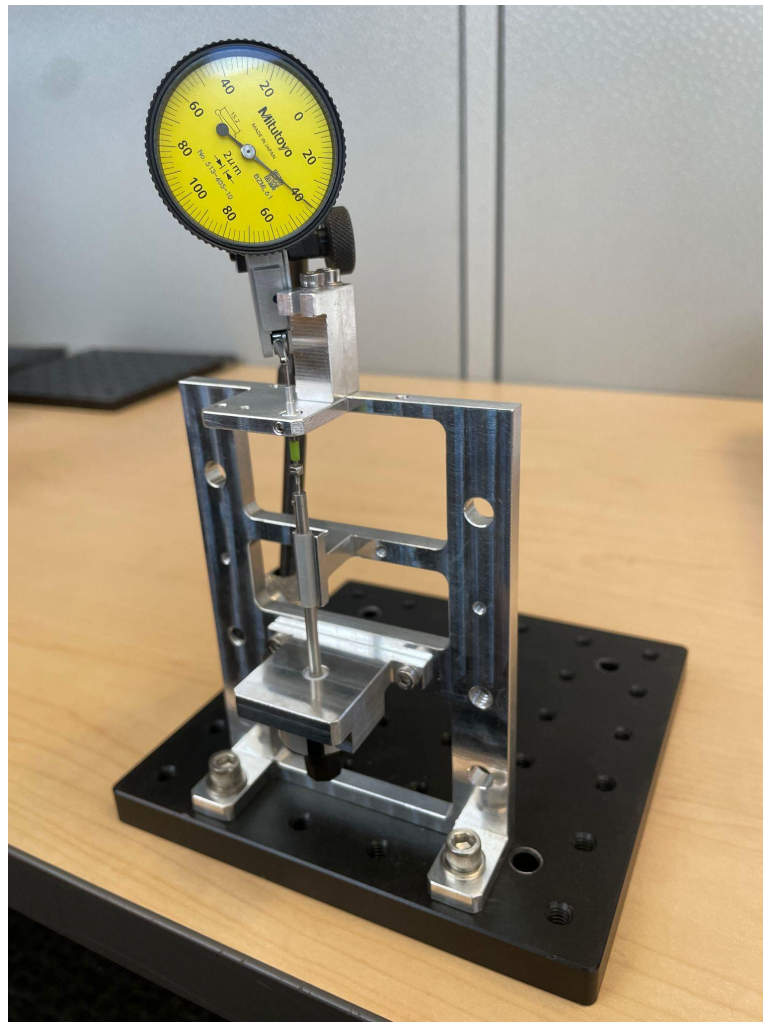
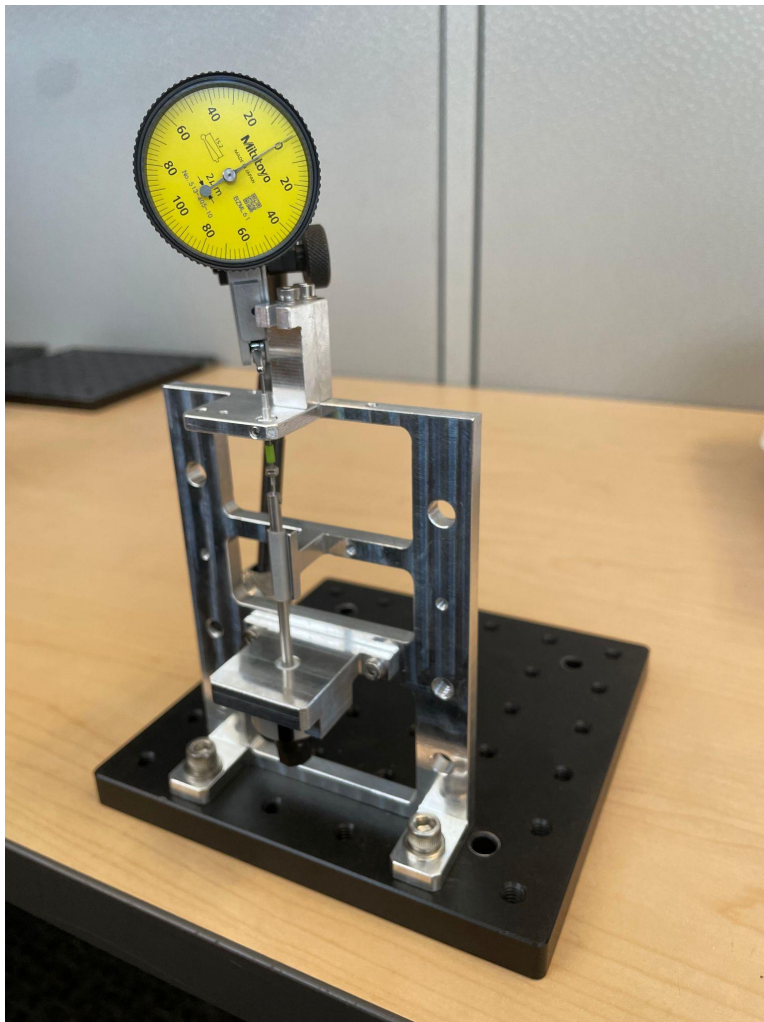
Knob does not fully constrain shaft

- ~10 μm axial displacement
- Unknown angular displacement



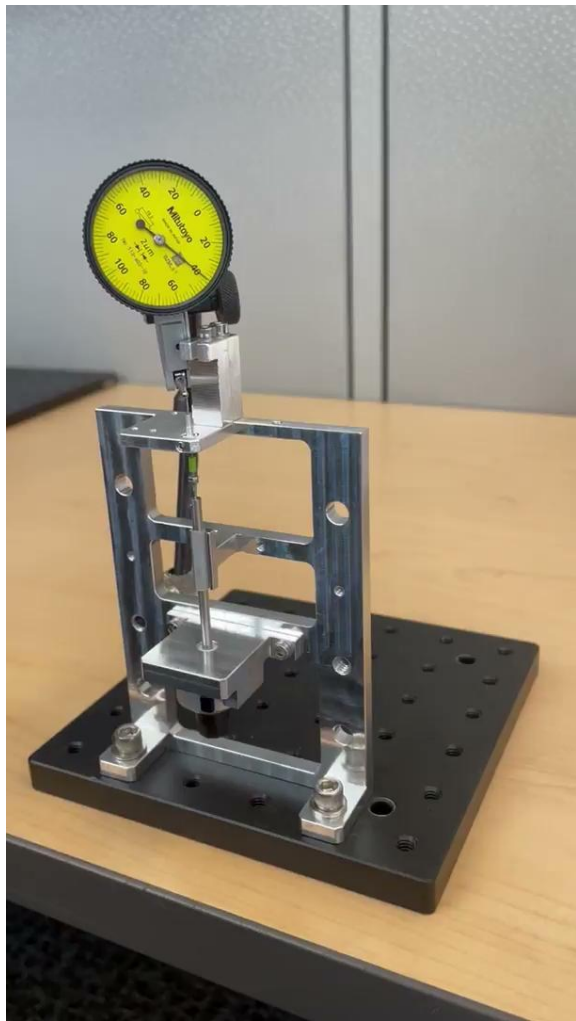
Fiber arm has significant play
before glueing

**⇒ Axial alignment may be
off after glueing**



Top Surface of
Beta Motor
Shaft:

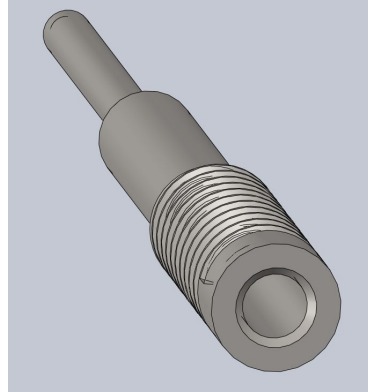
~40 μm max
displacement



Top Surface of
Beta Motor Shaft:

~40 μm max
displacement

Axial displacement
may originate in
turn-indicating knob



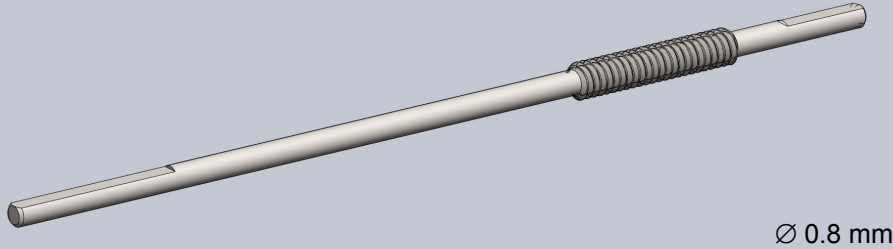
Difficult to thread MM Threaded Beta Shaft End into MMAF Fiber Arm

Required re-tapping and cleaning fiber arm a few times and gripping circular faces with sharp-nosed tweezers

Issue may be resolved when components are glued and threading can be driven by manual knob or motor

One fiber arm was ruined by inadvertent cross threading with the tap

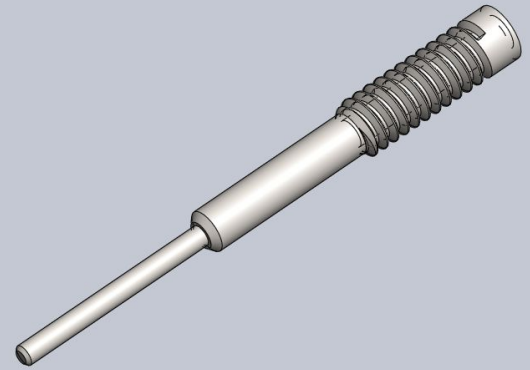
MMAF Beta Motor Shaft



Ø 0.8 mm

Small diameter parts risk getting bent

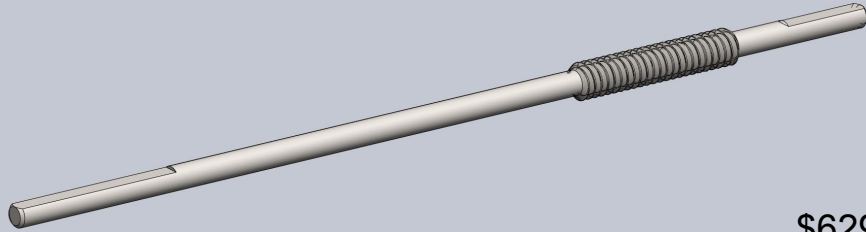
MMAF Threaded Beta Shaft End



Ø 0.6 mm

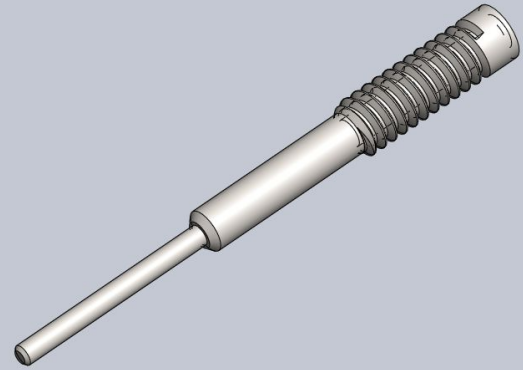
Small, high precision parts with multiple features can be costly

MMAF Beta Motor Shaft

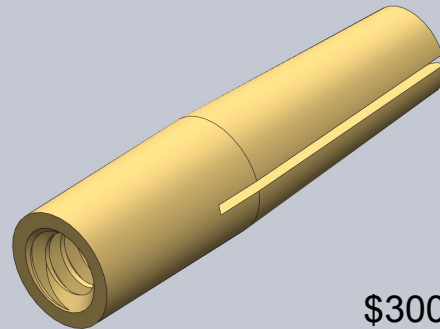


\$629

MMAF Threaded Beta Shaft End



\$629



\$300

Conclusions

- Challenges

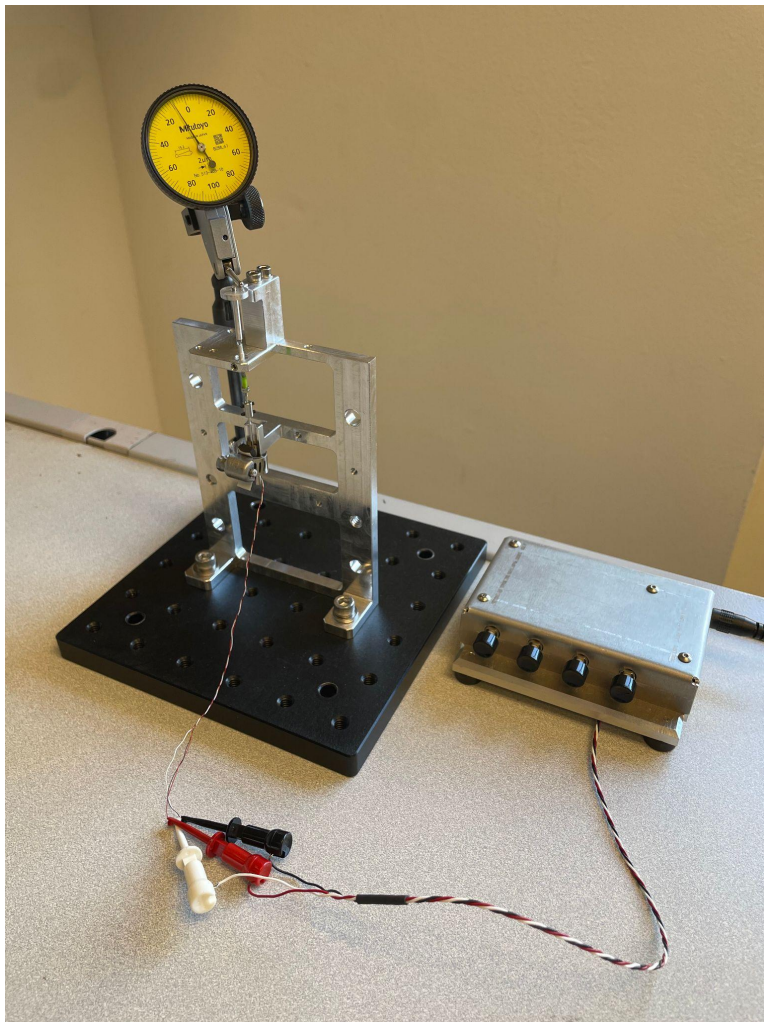
- Non-alignment between part axes \Rightarrow precession
- Main shaft is underconstrained
- Small diameter parts risk getting bent
- Large number of small, high tolerance parts with multiple features
- Potentially high cost

- Potential Solutions

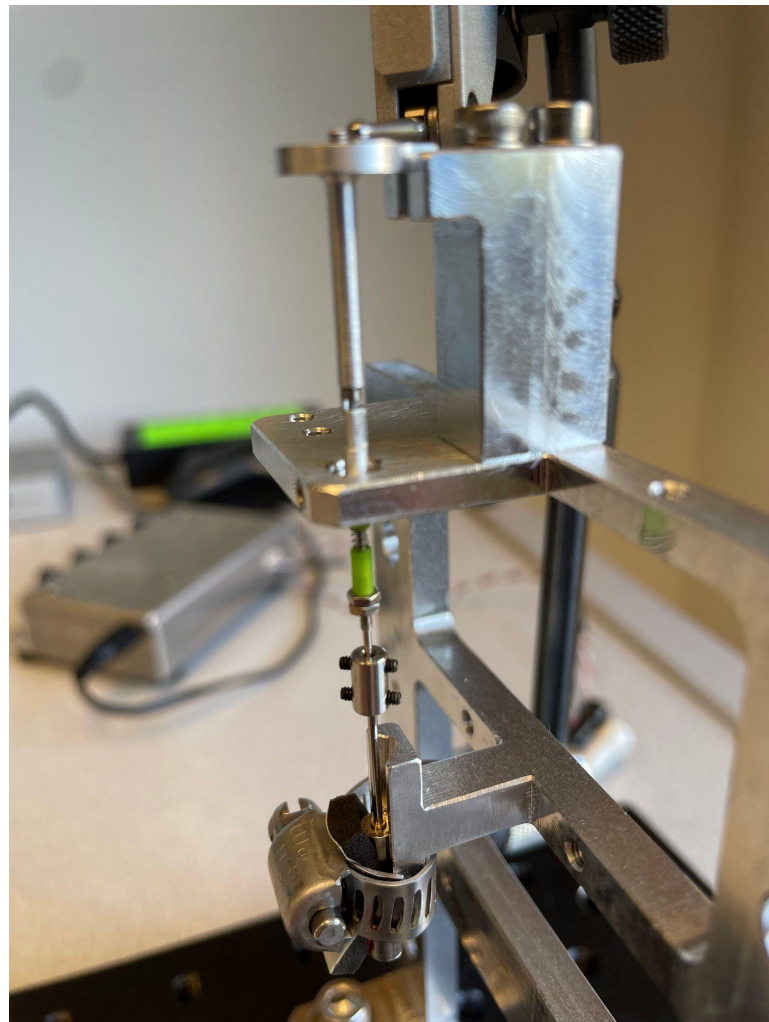
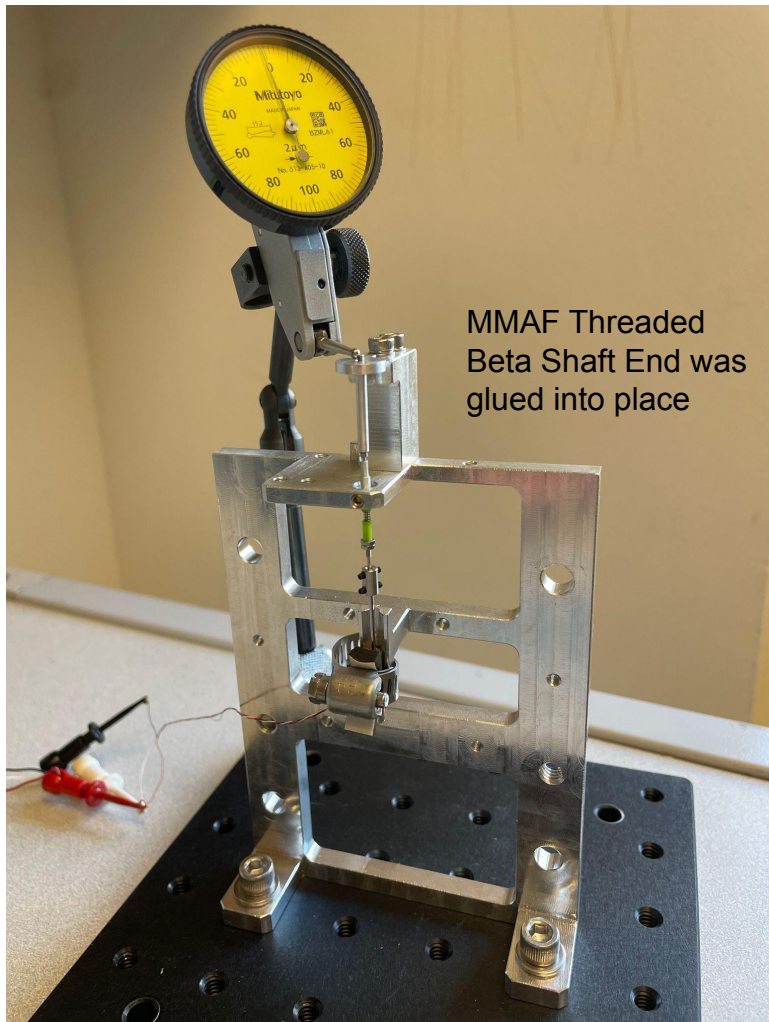
- Constrain main shaft
- Improve axial alignments
- Ensure temper/material of small diameter parts prevents plastic deformation

Trial 2:

Motor Actuation

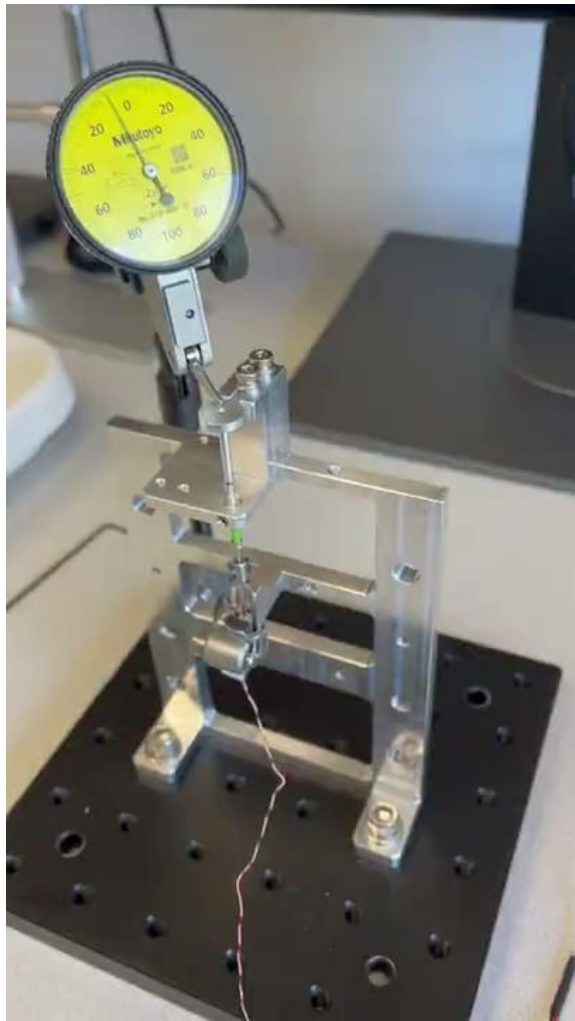


Motor and driver were installed





With the set screw loose (and having never been tightened against the collet) the motor successfully raised and lowered the fiber arm



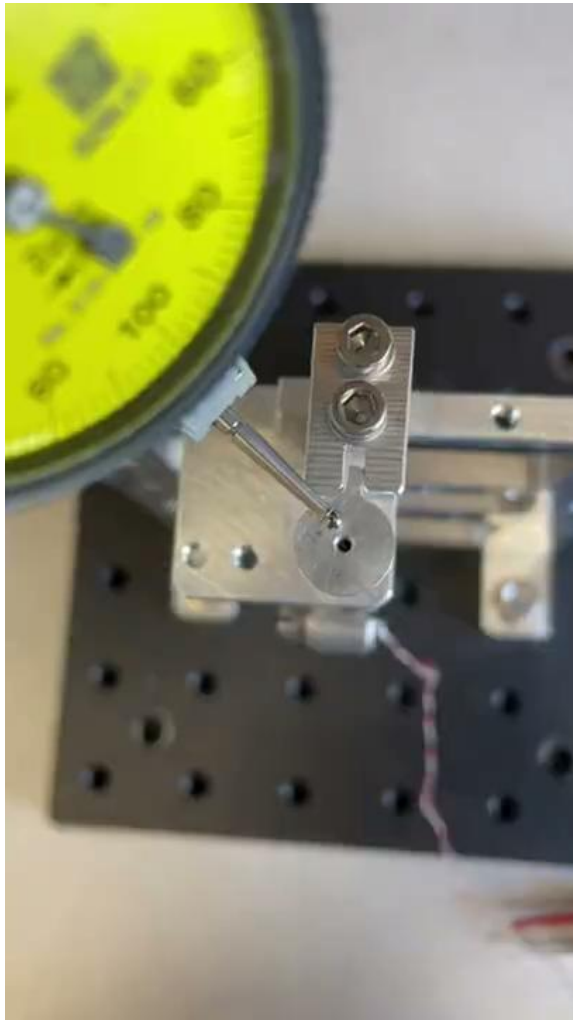
The set screw in the fiber arm was tightened

Tightening torque was not measured, but the set screw was tightened “lightly”, anticipating the risk of jamming the collet

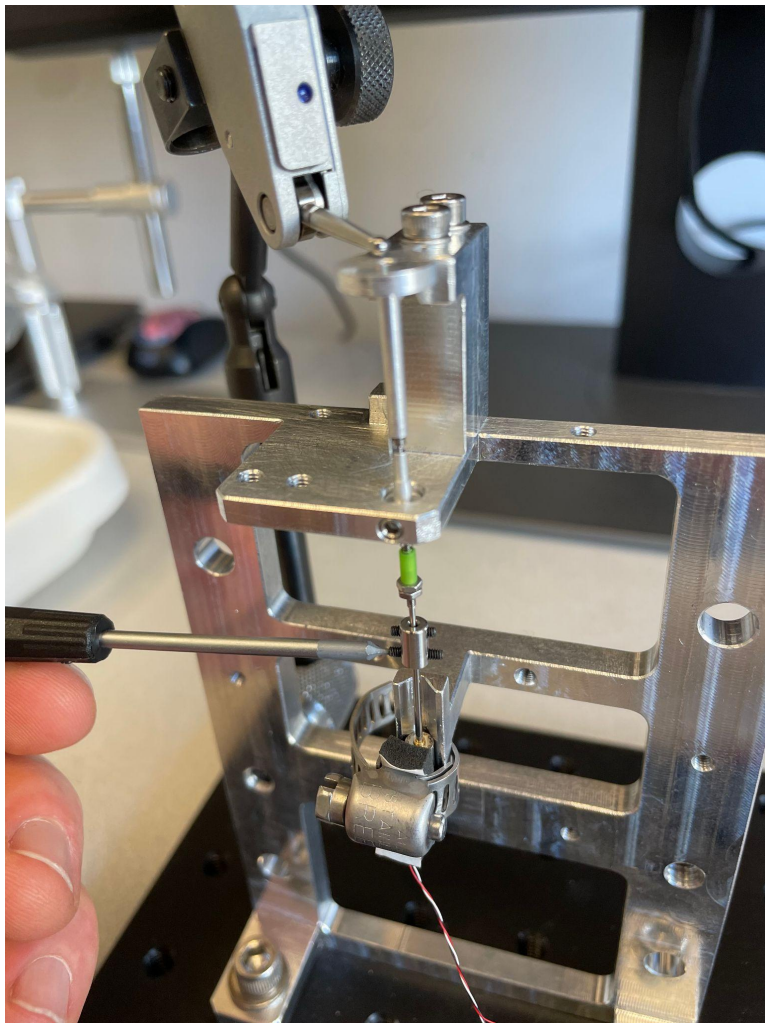
As expected, the fiber arm was locked in place and the motor could not advance the screw



When the set screw was loosened, the fiber arm was still locked in place



Even with the set screw fully removed, the motor was unable to overcome friction



The shaft was decoupled from the motor and advanced manually, raising and lowering the fiber arm several times

Significant friction was present



The shaft was recoupled to the motor
and the set screw was left removed

The motor was still unable to overcome
friction and advance the screw

⇒ After being tightened, the collet appears to get jammed into place such that the motor cannot overcome the resulting friction

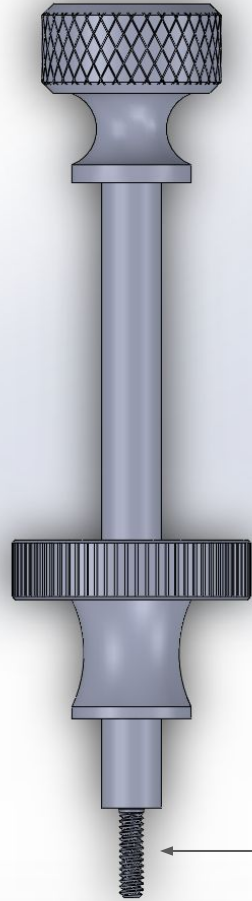
Conclusions

- Challenges
 - The collet appears to get jammed such that the motor cannot overcome the resulting friction
- Potential Solutions
 - Manually unjam the collet each time the fiber arm is adjusted by pulling it out with a tool threaded into the collet's internal threads



Challenge:

If the beta shaft is inserted too deeply into the collet, the internal threads of the collet are obstructed



Machine M0.8 threads
in bottom section of
thumb screw (or
drill/tap and insert a
threaded post)

Collet Removal Tool

Stainless Steel Flared-Collar Knurled-Head Thumb Screw
M2 x 0.40 mm Thread Size, 24 mm Long



Flanged Knurled-Head Thumb Nut
18-8 Stainless Steel, M2 x 0.40 mm Thread Size, 8 mm Head Diameter